

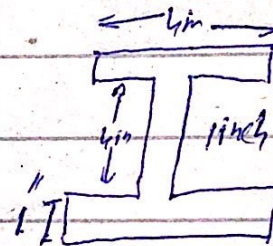
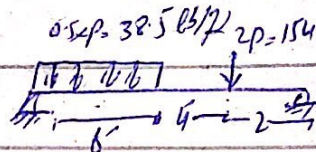
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Subject = MAS11

Pg # 1

Q. Construct the Mohr's circle diagram and find Principle Stress and max in plane Shear Stress for the stress at point C (located at the center of uniformly distributed load and 1 inch below the top fiber of beam cross-section) Show in fig. However, construct the Mohr circle it is necessary to draw Shear Stress and Flexural Stress variation diagrams for max Shear force & bending moment respectively. Compare result from Mohr circle with stress transformation equation.



ID = 7277

$$0.5 \times P = 0.5 \times 77 = 38.5 \text{ lb/ft}$$

$$2 \times 77 = 154 \text{ lb}$$

$$P = 77$$

Solution:

1st find reaction forces

R_A & R_B

$$\sum \epsilon_{MA} = 0$$

$$-(-38.5 \times 6 \times 3) - (154 \times 10) + 12 R_B = 0$$

$$R_B \times 12 - 847 = 0$$

$$R_B = \frac{847}{12}$$

$$R_B = 70.5 \text{ lb}$$

Now we know that

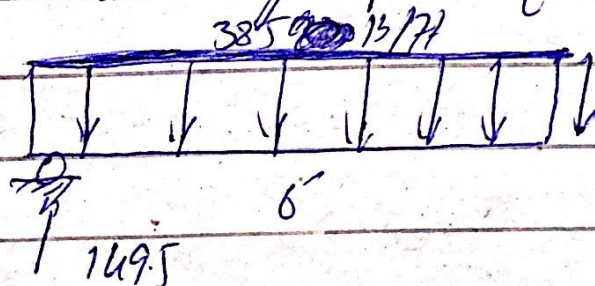
$$R_A + R_B = 220 \text{ lb}$$

$$R_A = 220 - R_B$$

$$R_A = 220 - 70.5$$

$$R_A = 149.5 \text{ lb}$$

Now Shear force at Change point of beam



Shear force at 6th from left

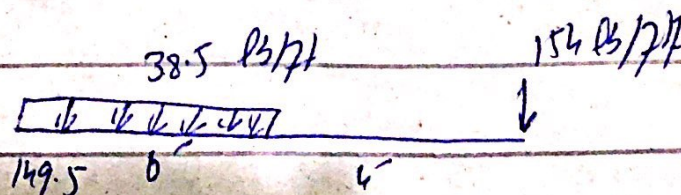
$$\Sigma F_y = 0 \uparrow +$$

$$149.5 - (38.5 \times 6) - V_f = 0$$

$$149.5 - 231 = V_f$$

$$V_f = -81.5 \text{ lb}$$

Now Shear at 10th

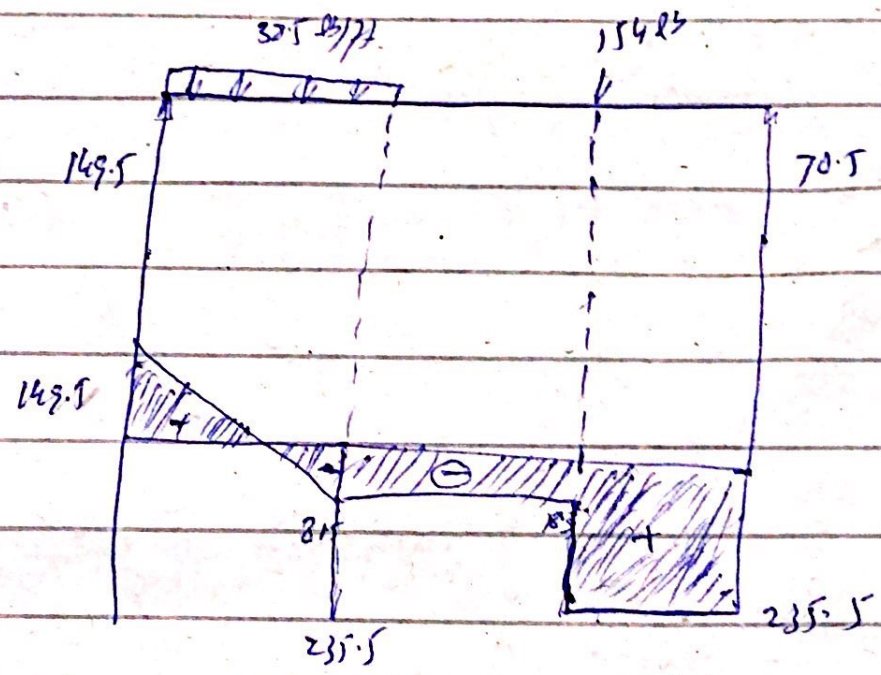


$$\sum F_y = 0 + \uparrow$$

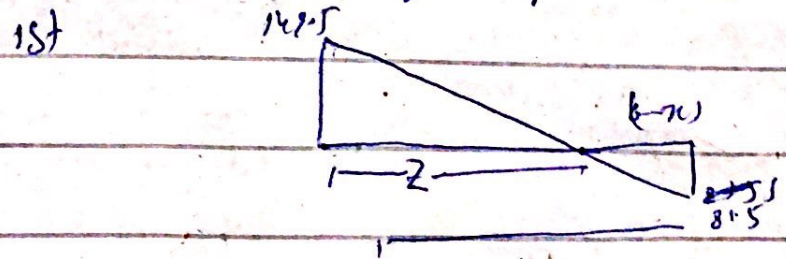
$$149.5 (38.5 + 6) - 154 - 127.10 = 0$$

$$V_{1071} = 235.5 \text{ (-ve)}$$

Shear force and moment bending



Moment at change point and zero shear point



by similar triangle

$$\frac{149.5}{x} = \frac{81.5}{(6-x)}$$

$$149.5 \times 6 = 897$$

$$897 = 149.5x + 81.5x$$

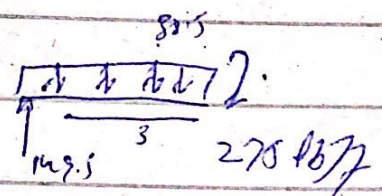
$$x = \frac{897}{231}$$

$$x = 3.8871$$

∴ at centre of UDL 37 from left

$$C^+ M = -149.5 \times 3 + (38.5 \times 3 + \frac{1.5}{2})$$

$$M = 275.1677$$

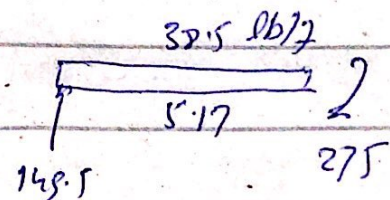


at a distance 3.88 from left support

$$(ii) M = -149.5 \times 3.88 + (38.5 \times 3.88 + \frac{3.88}{2})$$

$$M = 275.1677$$

$$M = 290.21$$



$$\sum F_y = 149.5 - 38.5 + 3 - V = 0$$

$$V = 34 \text{ lb}$$

Find moment of inertia

$I_{xx} = I_{xx1} + I_{xx2} + I_{xx3}$ from given dimensions

$$I_{xx1} = \frac{1}{12}(4)(1)^3 + 4(2.5)^2 = 25.33$$

$$I_{xx2} = \frac{1}{12}(4)(1)^3 + 4(1)^2 = 5.33$$

$$I_{xx3} = \frac{1}{12}(1)^3(4) + (3-5.5)^2 4 = 25.33$$

$$I_{xx} = I_{xx1} + I_{xx2} + I_{xx3}$$

$$I_{xx} = 25.33 + 5.33 + 25.33$$

$$I_{xx} = 56 \text{ in}^2$$

$$I_{yy} = I_{yy1} + I_{yy2} + I_{yy3}$$

$$= \frac{bh^3}{12} + \frac{bh^3}{12} + \frac{bh^3}{12}$$

$$= \frac{4(1)^3}{12} + \frac{1(4)^3}{12} + \frac{1(4)^3}{12}$$

$$I_{yy} = 11 \text{ in}^2$$

finding shear at point C 1 below from top zigzag

$$\tau_{xy} - \tau_{yx} = \frac{VQ}{Ib}$$

$$Q = Ay$$

$$A = 1 \times 4$$

$$A = 4 \text{ in}^2$$

$$Q = 1 \times 4 \times 2.5$$

$$Q = 10 \text{ in}^3$$

$$\tau_{xy} = \tau_{yx} = \frac{34 \times 10}{56 \times 4}$$

$$\tau_{xy} = 2.12 \text{ lb/in}^2$$

$$\sigma_x = \frac{My}{I}$$

$$= \frac{12 \times 242.61 \times 12}{56}$$

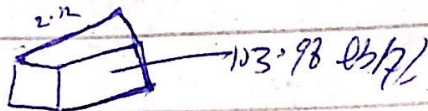
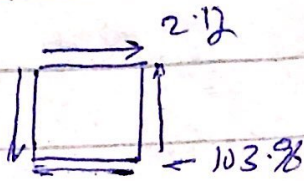
$$\sigma_x = 103.97 \text{ lb/in}^2$$

Flure stress at point C

$$\sigma = 103.97 \text{ lb/in}^2$$

Stress at point C

$$\tau = 2.12 \text{ lb/in}^2$$



Assume that element rotates at 30 rotation
as we derive following equation for stress transformation

$$\sigma_x = \frac{\sigma_x + \sigma_y}{2} - \left(\frac{\sigma_x - \sigma_y}{2} \right) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_x = \frac{103.76 + 0}{2} + \left(\frac{-103.76 + 0}{2} \right) (\cos 2\theta + 30) + 2.12 \sin 60$$

$$\boxed{\sigma_x = -172.07 \text{ lb/in}^2}$$

for σ_y

$$\sigma_y = \frac{\sigma_x + \sigma_y}{2} - \left(\frac{\sigma_x - \sigma_y}{2} \right) (\cos 2\theta) - \tau_{xy} \sin 2\theta$$

$$\sigma_y = \frac{-103.6 + 0}{2} - \left(\frac{-103.6 + 0}{2} \right) \cos 60 - 2 \sin 60$$

$$\boxed{\sigma_y = -121.00 \text{ lb/in}^2}$$

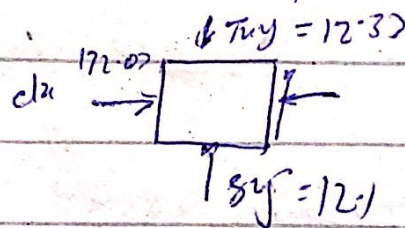
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$$\tau_{xy}' = \frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\tau_{xy}' = \frac{-103.6 - 0}{2} \sin 60 + 2.12 \cos 80$$

$$\tau_{xy}' = 12.33 \text{ lb/in}^2$$

Now stress state after 30° clockwise orientation to shown



2nd Principle Stress

we know that the principle stress equation is

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_{1,2} = \frac{-103.62 + 0}{2} \pm \sqrt{\left(\frac{103.62 - 0}{2}\right)^2 + (2.12)^2}$$

$$\sigma_{1,2} = -51.83 \pm 51.82$$

$$\sigma_y = \sigma_1 = 0.051 \text{ lb/in}^2$$

$$\sigma_{21} = \sigma_2 = -51.83 - 51.82 = -103.7 \text{ lb/in}^2$$

1st find θ ?

$$\tan 2\theta = \frac{\tau_{xy}}{(\sigma_x - \sigma_y)/2} = \frac{2.21}{(-103.67)} = -70 \text{ clockwise}$$

Put general equation

$$\sigma'_{x \max} = \frac{-103.67 + 0}{2} + \left(\frac{-103.67 - 0}{2} \right) \cos 2\theta + 2.21 \sin(-2.70)$$

$$\boxed{\sigma'_{x \max} = 103.67}$$

Max in Plan Shear Stress in this case

$$\tan 2\theta = \frac{(\sigma_x - \sigma_y)/2}{\tau_{xy}}$$

$$\tan 2\theta = 23.21$$

$$\theta = 175.11 \text{ Anticlockwise}$$

Put in the general equation for τ'_{xy}

$$\tau'_{xy} = -\left(\frac{\sigma_x - \sigma_y}{2} \right) \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\tau_{xy} = \left(\frac{103.67 - 0}{2} \right) \sin 2\theta (175.11) + 2 \cdot 21 \cos 2\theta (175.11)$$

$$\tau_{xy} = -6.57 \text{ lb/in} \quad \left[\text{max in plane shear stress} \right]$$

Mohr Circle

Center coordinate

$$(h, k) = \left[\frac{-103.67 + 0}{2}, 0 \right]$$

$$= [-51.33, 0]$$

Radius of Mohr Circle is

$$r = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2} \right)^2 + (\tau_{xy})^2}$$

$$r = \sqrt{\left(\frac{-103.67 - 0}{2} \right)^2 + (21)^2}$$

$$\boxed{r = 51.33 \text{ in}}$$

