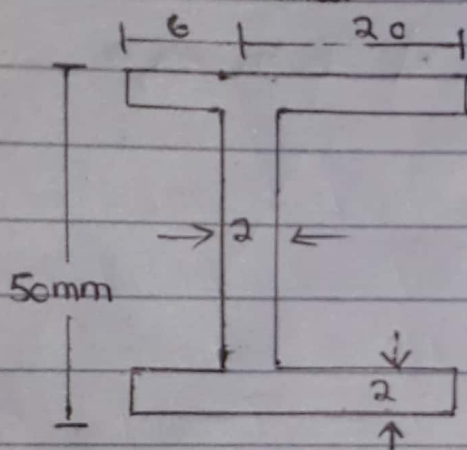


①

Question :- 01 (A)

Determine the location of the shear stress for the beams having the cross-sectional ..... ?



Required :-

Location of shear stress

Solution :-

$$e = \frac{t_f h^2 b^2}{4I} \quad \text{--- (1)}$$

And,

$$I = 2 \left( \frac{bh^3}{12} + Ay^2 \right) + \left( \frac{bh^3}{12} + Ay^2 \right)$$
$$= 2 \left( \frac{26(2)^3}{12} + (26 \times 2)(25)^2 \right) + \left( \frac{2(50)^3}{12} + 0 \right)$$

②

$$I = 2(32517.33) + 20833$$

$$I = 53350.66 \text{ mm}^4 + 32517.33$$

$$I = 85867.99 \text{ mm}^4$$

Now put the value of  $I$  in eq. ①,

$$e = \frac{2(50)^2(26)^2}{4(85867.99)}$$

$$e = 9.8406 \text{ mm}$$

Result :-

$$\text{Hence } e = 9.8406 \text{ mm}$$

Question :- 1 (PART B)

Determine the thickness of wall of water tank constructed ----- ?

Given that :-

$$h = 26\text{ft} \times 12$$

$$\text{Tangential stress} = 6000 \text{ psi}$$

$$\text{Specific weight} = 62.4 \text{ lb/ft}^3 = 62.4 / 12^3$$

$$\text{Diameter} = d = 22\text{ft} \times 12$$

Required :-

Thickness,  $t = ?$

Solution :-

As,

$$G_t = \frac{pD}{2t} \quad \text{--- (1)}$$

where pressure exerted by water,

$$p = rh$$

$$G_t = \frac{rhd}{2t}$$



(4)

$$t = \frac{2bD}{26t}$$

$$t = \frac{(0.036)(312)(264)}{6000 \times 2}$$

$$t = 0.247''$$

$$t = 0.0205 \text{ ft}$$

Result :-

Hence thickness,  $t = 0.0205 \text{ ft}$

5

Question :- Q2 (A)

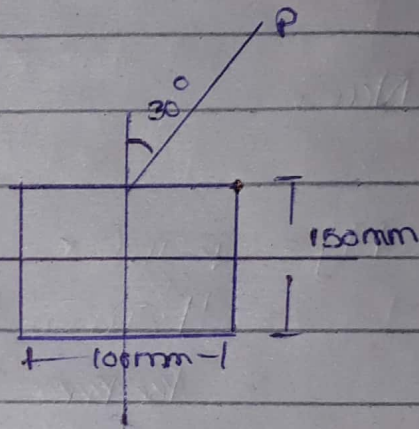
The 100 by 150 mm wooden beam shown in figure 2 ..... of the Beam.

Given :-

$$\text{Load} = 4 \text{ kN}$$

$$\theta = 30^\circ$$

$$\text{Length of span} = 3 \text{ m}$$



Required :-

$$\sigma_x = \frac{M_{xy}}{I_x} \quad (\text{Bending Stress})$$

$$\sigma_y = \frac{M_{yx}}{I_y} \quad (\text{Bending stress})$$

Location of Neutral Axis = ?

Solution :-

$$\sigma = -\sigma_x + \sigma_y$$

Now for  $M_x$ ,



6

$$\begin{aligned}M_x &= \rho \cos \theta \\&= 4 \times 10^3 \cos(30^\circ) \times \cancel{\text{Area}} \quad (\cancel{1/2}) \\&= 4 \times 10^3 \times 0.866 \times \cancel{\text{Area}} \quad (\cancel{1/2}) \\&= \cancel{\text{Area}} \times 3.46 \times 10^3\end{aligned}$$

Now,

$$\begin{aligned}M_y &= \rho \sin \theta \\&= 4 \times 10^3 \sin 30^\circ \quad (\cancel{1/2}) \\&= \cancel{\text{Area}} \times 2000\end{aligned}$$

Now,

$$I_x = \frac{bh^3}{12}$$

$$I_x = \frac{0.1(0.15)^3}{12}$$

$$I_x = 2.8125 \times 10^{-5} \text{ m}^4$$

$$I_y = \frac{hb^3}{12}$$

$$= \frac{0.15(0.1)^3}{12}$$

$$I_y = 1.25 \times 10^{-5} \text{ m}^4$$

⑦

So,

$$\begin{aligned} \sigma_x &= \frac{M_x}{I_x} \\ &= \frac{259.8}{2.8125 \times 10^{-5}} \end{aligned}$$

$$\sigma_x = 9.237 \text{ MNm}^{-2}$$

$$\sigma_y = \frac{M_y}{I_y}$$

$$\sigma_y = \frac{100}{1.25 \times 10^{-5}}$$

$$\sigma_y = 8 \text{ MNm}^{-2}$$

Now, Total Bending stress

$$\sigma_s = -\sigma_x + \sigma_y$$

$$\sigma = -9.237 + 8$$

$$\sigma = -1.237 \text{ MNm}^{-2}$$

Hence the total Bending stress is Negative which is in Compression.



⑧

Now for Neutral Axis of Unsymmetrical Bending.

$$\tan \alpha = \frac{I_x \cdot M_y}{I_y \cdot M_x}$$

$$\tan \alpha = \frac{2.8125 \times 10^{-5}}{1.25 \times 10^{-5}} \times \frac{3.46 \times 10^3}{2 \times 10^3}$$

$$\tan \alpha = \cancel{0.225} \quad 3.8925$$

$$\alpha = \tan^{-1} (\cancel{0.225}) (3.8925)$$

$$\alpha = \cancel{12.89^\circ} \quad 75.59^\circ$$



(9)

Question - 02(B)

The T section shown in the figure - 03 is the cross-section of a ..... beam?

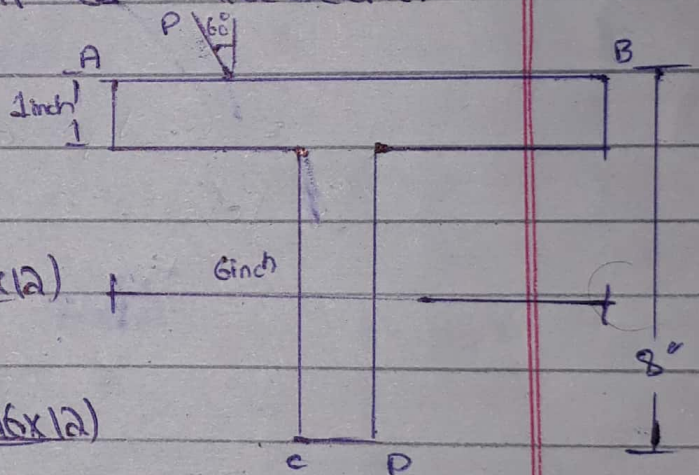
Solution:-

The maximum moment at mid section is,

$$M = \frac{PL}{4}$$

$$M_x = \frac{(P \cos 60^\circ)(16 \times 12)}{4}$$

$$M_y = \frac{(P \sin 60^\circ)(16 \times 12)}{4}$$



Now we have to find stresses at A, B, C and D.

At point(A) :-

$$\sigma_A = \frac{M_x y}{I_x} + \frac{M_y x}{I_y}$$

$$\sigma_A = \frac{-48 p \cos 60^\circ (3.07)}{112.6} - \frac{(48 p \sin 60^\circ)(3)}{18.7}$$

$$\begin{aligned} \sigma_A &= -0.654p - 6.6p \\ &= -7.2p \text{ (Compression)} \end{aligned}$$



(12)

and Compression  $\leq 12000$  psi

$$12000 = -7.25 P$$

$$P = 1655.1 \text{ lb}$$

point (B) :-

$$G_B = \frac{-48p \cos 60 (3.07)}{112.6} + \frac{(48p \sin 60)(3)}{18.7}$$

$$= -0.654p + 6.6p$$

$$= 6.01p \text{ Tension}$$

& Tension  $\leq 5000$  psi

$$5000 = 6.01p$$

$$p = \frac{5000}{6.01}$$

$$p = 831.94 \text{ lb}$$

Minimum value of  $p = 831.94 \text{ lb}$

point (C) :-

$$G_C = + \frac{(48p \cos 60)(3.07)}{112.6} + \frac{(48p \sin 60)(3)}{18.7}$$

$$= 0.654p + 6.6p = 7.25p$$

& Tension  $\leq 5000$  psi

$$5000 = 7.25p$$

$$p = 689.65 \text{ lb}$$

①

point (D) :-

$$\delta_D = \frac{+ (48p \cos 60^\circ)(3.07)}{1126} - \frac{(48p \sin 60^\circ)(3)}{18.7}$$

$$\delta_D = + 0.654p - 6.6p$$
$$= - 5.946p \text{ (compression)}$$

and Compression  $\leq$  12000 psi

$$12000 = - 5.946p$$

$$p = 2018.16 \text{ lb}$$

So, we have considered Minimum value of p,

$$p = 689.65 \text{ lb.}$$



(12)

Question :- Q3

A 10 ft long strut braced in the middle has ----- and  $E = 10.3 \times 10^6$ ?

Given Data :-

Length,  $L = 10$  ft

Breadth,  $b = 0.75$  "

Height,  $h = 2$  "

Factor of Safety = 2

$E = 10.3 \times 10^6$

Required Data :-

Safe Load,  $P_{safe} = ?$

Solution :-

Case :- 1

Strut column act as hinged column about an axis perpendicular to the 2 inch dimension then,

$$I = I_x = \frac{(0.75)(2)^3}{12} = 0.5 \text{ in}^4$$

(13)

$L_e = L$  (for Hinged Ended Column)

$$P_{cr} = \frac{n^2 EI \pi^2}{L_e^2}$$

$$P_{cr} = \frac{(1)^2 (10.3 \times 10^6) (0.5) (3.14)^2}{(10 \times 12)^2}$$

$$P_{cr} = 3526.17 \text{ lb}$$

$$\underline{\underline{P_{safe}}} = \frac{P_{cr}}{\text{factor of safety}}$$

$$P_{safe} = \frac{3526.17}{2}$$

$$P_{safe} = 1763.08 \text{ lb}$$

Case II :-

act  
Column as a fixed end about axis  
parallel to 2 inch. i.e. y-axis.

$$I = I_y = \frac{(2)(0.75)^3}{12}$$

$$I_y = 0.07 \text{ in}^4$$



(14)

Now for fixed ended,

$$L_e = L/2$$

$$P_{cr} = \frac{n^2 E I \pi^2}{L_e^2}$$

$$P_{cr} = \frac{(1)^2 (10.3 \times 10^6) (0.07) (3.14)^2}{(120/2)^2}$$

$$P_{cr} = 1974.65 \text{ lb}$$

For  $P_{safe}$  :-

$$P_{safe} = \frac{P_{cr}}{\text{factor of safety}}$$

$$P_{safe} = \frac{1974.65}{2}$$

$$P_{safe} = 987.32 \text{ lb}$$

In both case we take smaller value of  $P_{safe}$ .

$$P_{safe} = 987.32 < 1763.08$$



# FINAL TERM (SEMESTER 4)

NAME :- MUHAMMAD TALHA

AD :- 7965

SECTION :- "B"

SUBMITTED TO SIR SAQIB

SUBJECT :- MOS (II)

DATE :- 23 - June 2020

DEPARTMENT :- CIVIL ENGINEERING