

NAME :- ALI HASNAIN TARIQ

ID :- 7966

SEC :- B

SUBJECT :- NMOS - II

TEACHER :- SIR SAQIB

EXAM :- FINAL

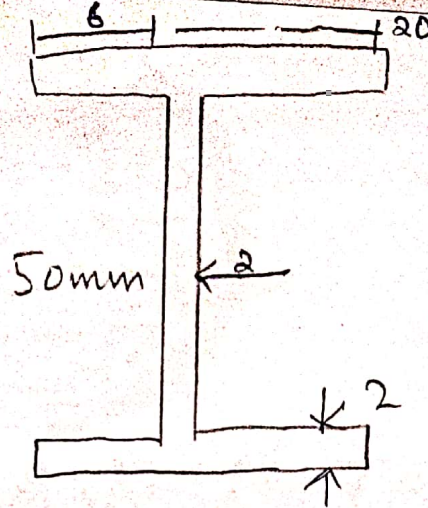
SEM :- 4

DEPT :- CIVIL

Question

1

(a)



①

Required:- location of shear centre

Sol:- As we know that

$$e = \frac{6ph^2b^2}{4I}$$

and

$$\begin{aligned} I &= 2 \left( \frac{bh^3}{12} + Ay^2 \right) + \left( \frac{bh^3}{12} + Ay^2 \right) \\ &= 2 \left( \frac{26(2)^3 + (20 \times 2)(25)^2}{12} \right) + \left( \frac{2(50)^3}{12} + 0 \right) \end{aligned}$$

$$I = 50034.67 + 20833$$

$$I = 70867.98 \text{ mm}^4$$

$$e = \frac{2(50)^2(25)^2}{4(70867.98)} = 11.03 \text{ mm}$$

So,

$$\text{Shear centre } e = \underline{11.03 \text{ mm}}$$

Question :- 2 part (b)

Given data:-

$$h = 26 \text{ ft} \times 12$$

tangential stress = 6000 Psi

$$\text{specific weight} = 62.4 \text{ lb/ft}^3 = 62.4/12^3$$

$$\text{dia } d = 22 \text{ ft} \times 12$$

Required:-

thickness  $t = ?$

Solution:-

As,

$$\sigma_t = \frac{pD}{2t} \quad \text{--- (1) where } p \text{ exerted by water}$$

$$p = \gamma h$$

$$\sigma_t = \frac{\gamma h D}{2t}$$

OR

$$t = \frac{\gamma h D}{\sigma_t \times 2}$$

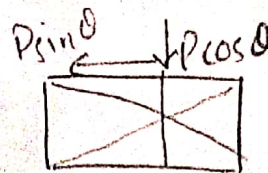
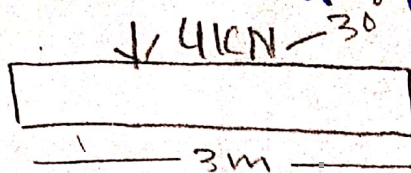
$$t = \frac{(62.4) \times 12 (26) \times 12 (22) \times 12}{(6000) \times 2}$$

$$t = 0.2474$$

$$t = 10.0208 \text{ ft}$$

# Question: 2 Part (A)

3

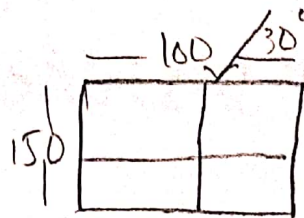


Given:

$$\text{Load} = 4 \text{ kN}$$

$$\theta = 30^\circ$$

$$\text{Length} = 3 \text{ m}$$



Required:

$$\sigma_x = \frac{M_y}{I_x} \quad (\text{Bending stress})$$

$$\sigma_y = \frac{M_x}{I_y}$$

Location of neutral axis = ?

$$\sigma = \sigma_x + \sigma_y$$

Solution:

$$M_x = P \cos \theta$$

$$= (\cos 30^\circ \times 2) \times 150/2$$

$$= (4 \times 0.866) \times (0.075)$$

$$= 259.8 \text{ Nm}$$

$$M_y = P \sin \theta \times 2$$

$$\Rightarrow M_y = 4 \sin 30^\circ \times 0.05$$

$$M_y = 100 \text{ Nm}$$

Now

$$I_x = \frac{bh^3}{12}$$

$$\Rightarrow I_x = \frac{0.4(0.15)^3}{12}$$

$$I_x = 2.8125 \times 10^{-5} \text{ m}^4$$

$$I_y = \frac{hb^3}{12} = \frac{0.15(0.1)^3}{12}$$

$$I_y = 2.25 \times 10^{-5} \text{ m}^4$$

$$\sigma_x = \frac{Mx}{I_x}$$

$$\Rightarrow \sigma_x = \frac{259.8}{2.8125 \times 10^{-5}}$$

$$\sigma_x = 9.237 \text{ MN/m}^2$$

$$\sigma_y = \frac{My}{I_y}$$

$$\sigma_y = \frac{100}{1.25 \times 10^{-5}}$$

$$\sigma_y = 8 \text{ MN/m}^2 \text{ (compression)}$$

Now

Total bending stress,  $\sigma = -\sigma_x + \sigma_y$

$$\Rightarrow \sigma = -9.237 + 8$$

$$\Rightarrow \underline{\underline{\sigma = -1.237 \text{ MNm}^2}}$$

Now for neutral axis  $\curvearrowright$   
unsymmetrical bending

$$\tan d = \frac{I_{xx}}{I_{yy}} \cdot \frac{M_y}{M_x}$$

$$= \frac{2.0105 \times 10^5}{12 \times 10^5} \times \frac{100}{259.8}$$

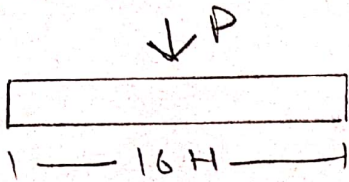
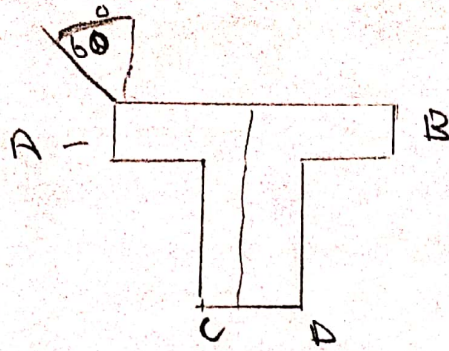
$$\tan d = 0.8666$$

$$d = \tan^{-1} 0.8666$$

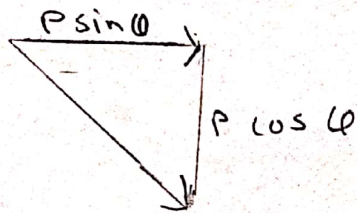
$$d = 40.89^\circ$$

(5)

(6)



|   |   |
|---|---|
| - | - |
| + | + |



the max moment of mid section is

$$M = \frac{PL}{4}$$

$$M_x = \frac{(P \cos 60)(16 \times 12)}{4}$$

$$M_y = \frac{(P \sin 60)(16 \times 12)}{4}$$

stress of A, B, C, D

A →

$$\sigma = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$= -\frac{(48P \cos 60)(3.07)}{112.6} - \frac{(48P \sin 60)(3)}{18.7}$$

$$= -0.654P - 6.6P$$

$$= -7.2 \text{ Compression}$$

$$\epsilon_p \text{ Compression} \leq 12000 \text{ psi}$$

$$12000 = -7.2P$$

$$P = 1655.116$$

B →

$$\sigma = \frac{-40P \cos 60(3.07) + (48P \sin 60)(3)}{112.6 + 18.7}$$

$$= -0.654P + 6.6P$$

$$= 6.01P \text{ Tension}$$

$$\epsilon_t \text{ Tension} \leq 500 \text{ psi}$$

$$5000 = 6.01P$$

$$P = \frac{5000}{6.01}$$

$$P = 831.94 \text{ lb}$$



$\rightarrow C$

$$b = \frac{(48 \cos 60)(3.07)}{112.6} + \frac{(48p \sin 60)(3)}{18.7}$$

$$= 0.654p + 6.6p$$

$$= 7.25p$$

$$\epsilon \text{ Tension} \leq 5000 \text{ psi}$$

$$5000 = 7.25p$$

$$p = 689.651b$$

$$\underline{D} = \frac{+(48p \cos 60)(3.07) - (48p \sin 60)(3)}{112.6 - 18.7}$$

$$= 0.654p - 6.6p$$

$$= -5.946p$$

$$\epsilon \text{ compression} = 1200 \text{ psi}$$

$$12000 = -5.946p$$

$$p = 2018.161b$$

Given data:-

$$\text{length } L = 10 \text{ ft}$$

As both sides are hinged

$$\text{So } l_e = L$$

$$E = 10.3 \times 10^6$$

$$\text{factor of safety} = 2$$

$$b = 0.75 \text{ inch}$$

$$h = 2 \text{ inch}$$

Required:-

safe load = ?

Solution:-

$$P_{cr} = \frac{\pi^2 EI}{l_e^2}$$

$$= I = A r^2$$

$$I = A r^2$$

$$r = \sqrt{I/A}$$

$$r = \frac{\sqrt{\frac{hb^3}{12}}}{bh} \Rightarrow \sqrt{\frac{b^2}{12}}$$

$$r = \frac{b}{2\sqrt{3}} \Rightarrow \frac{0.75}{2\sqrt{3}}$$

$$r = 0.216 \text{ inch}$$

$$P_{cr} = \frac{\pi^2 EA}{(l_e/r)^2}$$

$$\Rightarrow \frac{(3.14)^2 (10.3 \times 10^6) (1.5)}{(10/0.216)^2}$$

$$P_c = 853.8434$$

Safe load =  $\frac{\text{crippling load}}{\text{factor of safety}}$

$$\Rightarrow \frac{853.8434}{2}$$

$$\text{Safe load} \Rightarrow 426.917$$

\* for fixed ended column

$$l_e = l/2 = 10/2$$

$$l_e = 5 \text{ ft}$$

$$P_{c1} = \frac{\pi^2 EA}{(l_e/l)^2} = \frac{(3.14)^2 (10.3 \times 10^6) (1.5)}{(5/0.216)^2}$$

$$P_{c1} = 1974.207$$

Safe load =  $\frac{P_{c1}}{\text{factor of safety}}$

$$= \frac{1974.207}{2}$$

$$987.103$$