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SUBMITTED BY

SOHAIL AHMED .

ID 7907

SECTION A .

SUBJECT MOS II

SUBMITTED TO

ENGR. SAIB .

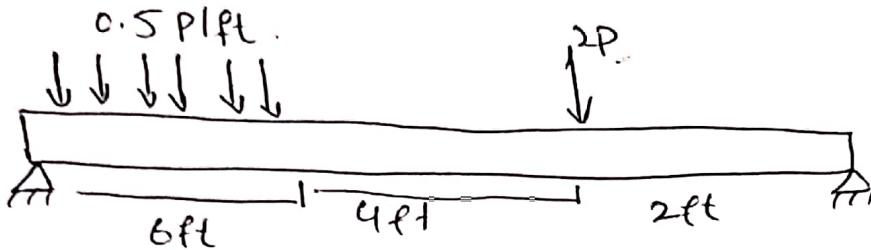
DATE 16 - 04 - 2020

(P1)

ANSWER TO GIVEN QUESTION.

WHERE P IS CONSIDER OUR ID LAST TWO DIGIT.

→ My ID is 7907. So My I put the value of P is 07.

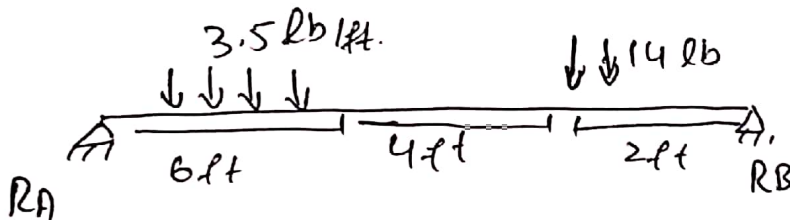


So we get .

$$\Rightarrow 2P = 2 \times 07 = 14 \quad \text{while}$$

$$\Rightarrow 0.5 \times 07 \Rightarrow 3.5$$

★ FREE BODY DIAGRAM :-



★ SUPPORT REACTION :- AS we know that

$$\sum F_y = 0 \quad \uparrow^+ \quad \downarrow^-$$

$$R_A + R_B = 17.5 \text{ lb} \quad \dots \quad 3.5 + 14 = 17.5$$

Now

$$\sum M = 0 \quad \curvearrowleft^+ \quad \curvearrowright^-$$

$$R_B \times 12 - 14 \times 10 - 21 \times 3$$

$$\boxed{R_B = 16.91 \text{ lb}}$$

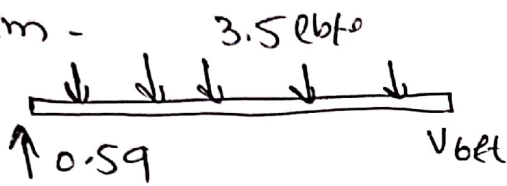
Where

$$R_A + R_B = 17.5$$

$$R_A = 17.5 - 16.91$$

$$R_A = 0.59 \text{ lb}$$

★
⇒ Now shear force at change point of beam -



$$\therefore R_A + R_B = 17.5$$

$$0.59 + 16.91 = 17.5$$

⇒ $17.5 = 17.5 \text{ lb}$
Hence in equilibrium.

So

Shear force at 6 ft from left support -

$$\sum F_y = 0 \quad \uparrow \quad \downarrow^+$$

$$\rightarrow V_{6ft} - 0.59 + 3.5 \times 6 = 0$$

$$V_{6ft} + 20.41 = 0$$

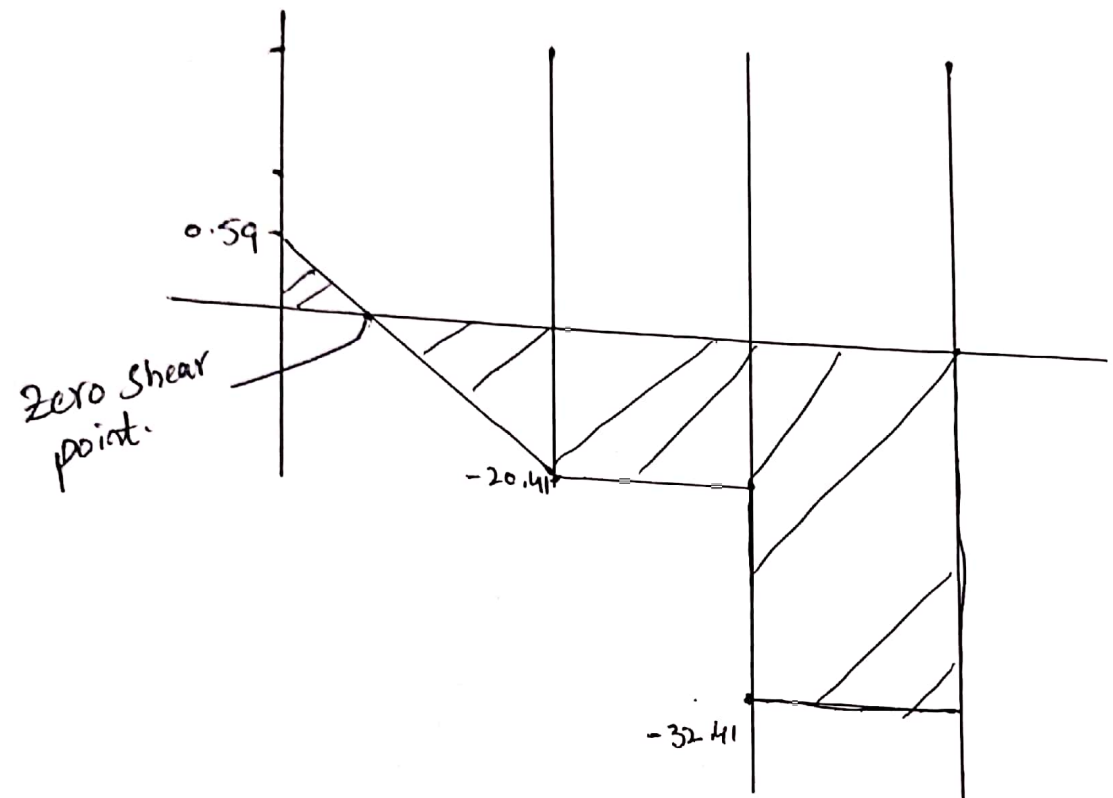
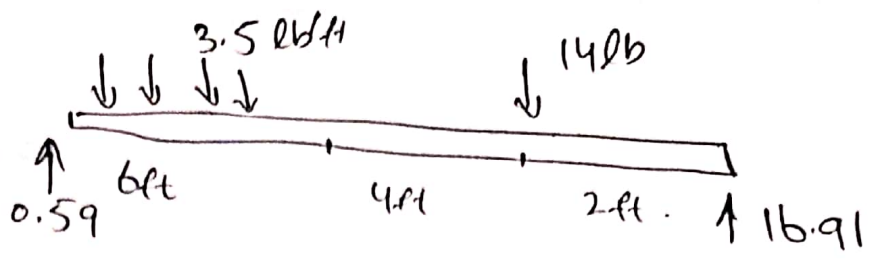
$$V_{6ft} = -20.41 \text{ lb}$$

⇒ Now find shear force at V_{10ft} .

$$\sum F_y = 0 \quad \uparrow \quad \downarrow^+$$

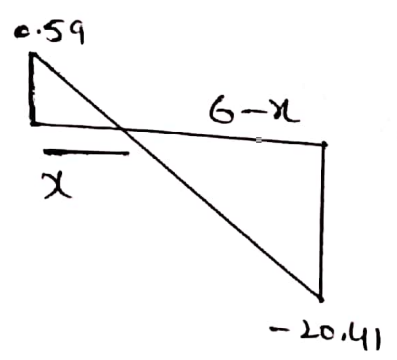
$$-0.59 + 21 + 12 + V_{10ft} = 0$$

$$V_{10ft} = -32.41 \text{ lb}$$



⇒ Now we find moment at change point.
 Find zero shear point.

$$\frac{0.59}{x} = \frac{+20.41}{(6-x)}$$



→ $0.59(6-x) = 20.41x$

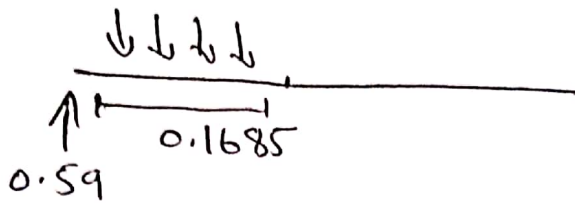
→ $3.54 - 0.59x = 20.41x$

$$3.54 = 20.41x + 0.59x$$

$$3.54 = \frac{21x}{21} \Rightarrow \boxed{x = 0.1685 \text{ ft}}$$

Now we will find moment at
Section of 0.1685ft from RA.

(P4)



$$\sum M_{0.1685} = 0$$

$$M_{0.1685} + 0.59 \times 0.1685 - 21 \left(\frac{0.1685}{2} \right) = 0.$$

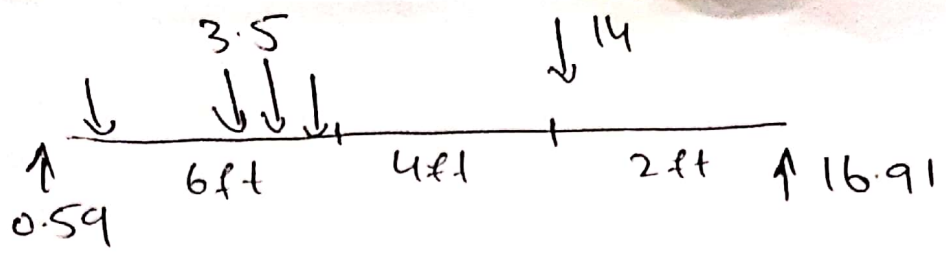
$$M_{0.1685} + 0.0994 - 1.7692 = 0$$

$$M_{0.1685} = 1.698 \text{ lb}\cdot\text{ft}$$

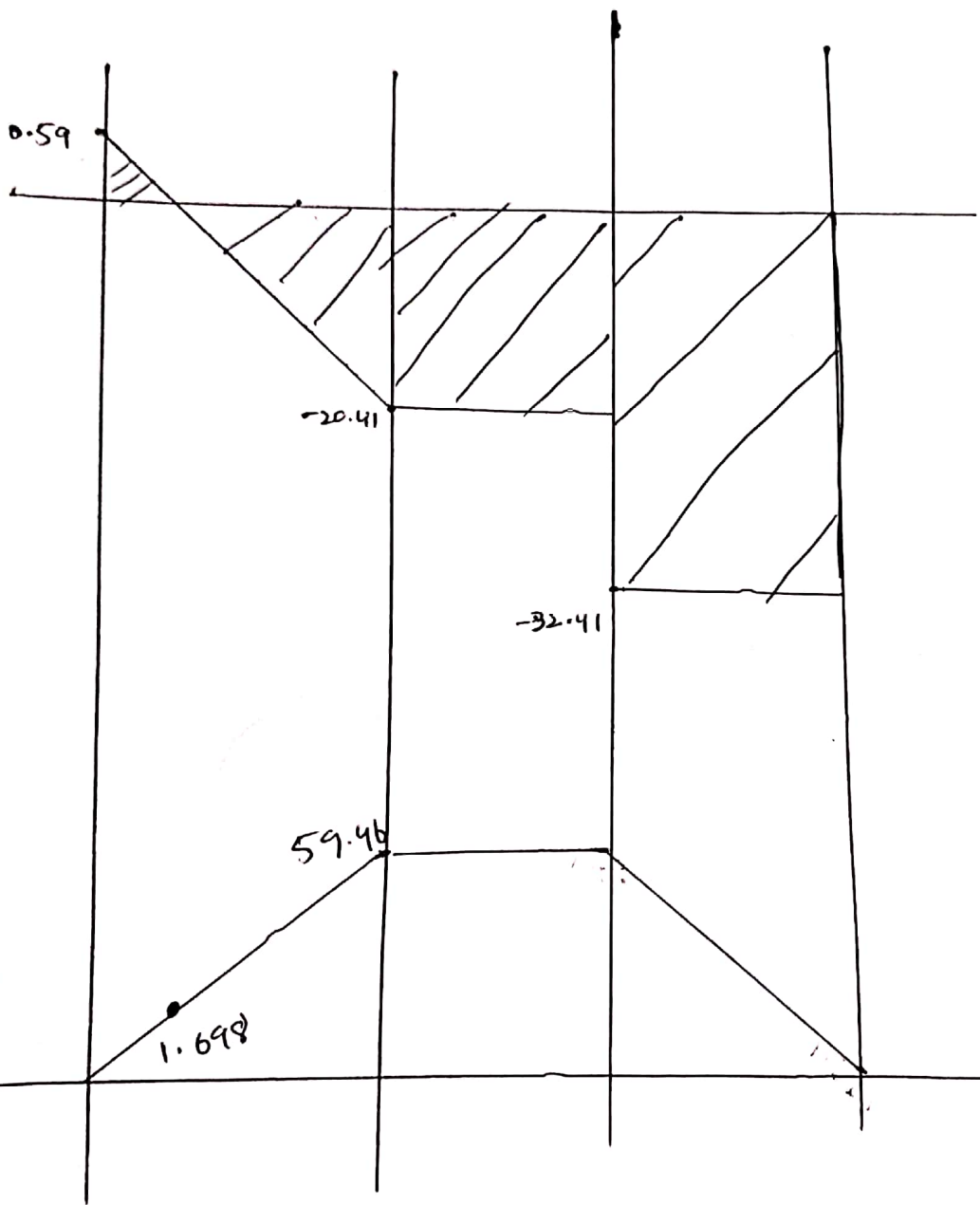
$$\Rightarrow \sum M = 0$$

$$M_{6\text{ft}} + 0.59 \times 6 - 3.5 \times 6 \times 3$$

$$M_{6\text{ft}} = 59.46 \text{ lb}\cdot\text{ft}$$



SFD



BMD

Shear stress :

As per Question the maximum Shear stress $\tau = \frac{VQ}{It}$ occurs where the maximum Shear force is 16.92 lb

So,

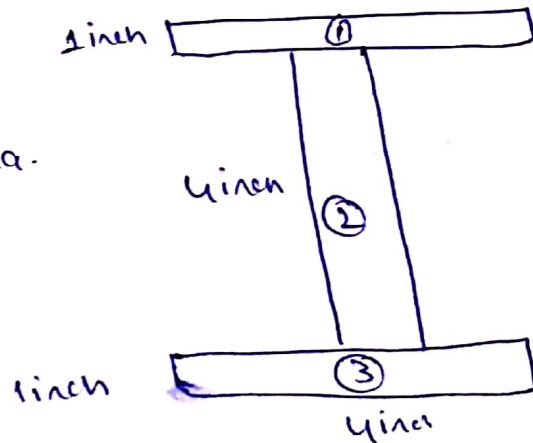
To find the Shear Stress we have the following formula.

$$\tau = \frac{VQ}{It}$$

So we will find the moment of inertia.

→ we will find the centroid by following formula.

$$\bar{y} = \frac{A_1 y_1 + A_2 y_2 + A_3 y_3}{A_1 + A_2 + A_3}$$



$$A_1 = 4 \times 1 = 4$$

$$A_2 = 4 \times 1 = 4$$

$$A_3 = 4 \times 1 = 4$$

$$\bar{y} = \frac{4 \times 0.5 + 4 \times 3 + 4 \times 5.5}{4 + 4 + 4}$$

$$\bar{y} = 3''$$

Moment of inertia.

(P-7)

| No | A(in ²) | \bar{I}_x (in ⁴) | $d = (\bar{y} - y_1)$ $(\bar{y} - y_2)(\bar{y} - y_3)$ |
|----|---------------------|--------------------------------|---|
| ① | 4 | $\frac{4 \times (1)^2}{12}$ | = 0.333 |
| ② | 4 | $\frac{1 \times (4)^3}{12}$ | = 5.333 |
| ③ | 4 | $\frac{4 \times (1)^3}{12}$ | = 0.333 |

(Now 'd').

- ① $d = (\bar{y} - y_1) = 3 - 0.5 = 2.5$
- ② $d = \bar{y} - y_2 = 3 - 3 = 0$
- ③ $d = 3 - 5.5 = -2.5$

Now d^2 .

- ① $4 \times (2.5)^2 = 25$
- ② $4 \times (0)^2 = 0$
- ③ $4 \times (-2.5)^2 = 25$

Now

$$\bar{I}_x = \bar{I}_x + Ad^2$$

- ① $0.333 + 25 = 25.333$
- ② $5.333 + 0 = 5.333$
- ③ $0.333 + 25 = 25.333$

Total

$$I = \bar{I}_1 + \bar{I}_2 + \bar{I}_3$$

$$I = 55.999 \text{ in}^4$$

Now shear stress

P8

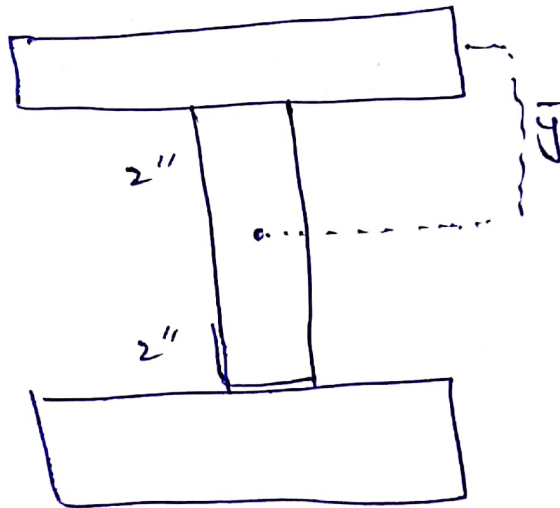
$$\tau = \frac{VQ}{Ib}$$

$$V_{max} = 16.91 \text{ lb}$$

$$Q = \bar{y}A$$

while $b =$ breadth of the fiber.

→ Shear stress at point c located at centre of uniformly distributed load Q 1 inch below the top fiber.



$$\bar{y} = 2 + 0.5 = 2.5$$

$$A = 4 \times 2 = 8$$

$$Q = 4 \times 2.5 = 10$$

As we know that

$$\tau = \frac{VQ}{Ib}$$

$$\tau = \frac{(16.91)(10)}{(55.996)(4)}$$

$$\tau = 0.7549 \text{ psi}$$

To find flexural stress,

(P9)

$$\sigma = \frac{MY}{I}$$

where M is maximum moment in
B.M.D.

$$M = 59.46 \text{ lb}$$

$$\sigma = \frac{(59.46)(2)}{55.996}$$

$$\sigma = 2.1237 \text{ psi}$$

So we know that shear stress at
point C is

$$\tau = 0.7549 \text{ psi}$$

Flexural stress at point C is

$$\sigma = 2.1237 \text{ psi}$$

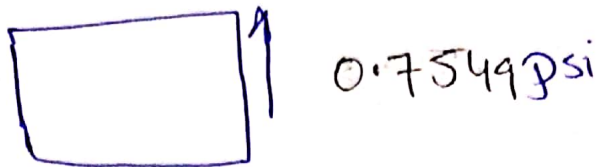
Now consider ' c ' is a planar element



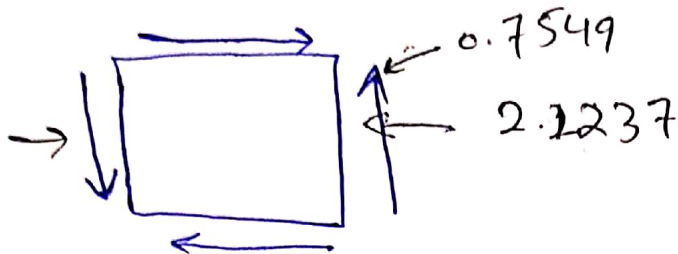
2.1237 psi is compressive because point C lies
in compression zone of beam cross section.

Now Shear Stress is

P10



Showing combine stress on 2D element.



Now we can find the stress state consider of point 'C' at a degree of 20° on clockwise.

$$\sigma_x = -2.1237 \text{ psi}$$

$$\sigma_y = 0$$

$$\tau_{xy} = 0.7549 \text{ psi}$$

$$\sigma_{x'} = ?$$

$$\sigma_{y'} = ?$$

$$\tau_{x'y'} = ?$$

As we know that

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

As we know that,

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta \quad \text{(P11)}$$

$$\sigma_{y'} = \frac{\sigma_x + \sigma_y}{2} - \frac{\sigma_x - \sigma_y}{2} \cos 2\theta - \tau_{xy} \sin 2\theta$$

⇒ For $\sigma_{x'}$

$$\sigma_{x'} = \frac{2.1237 + 0}{2} + \frac{-2.1237 - 0}{2} \cos(2(-20)) + (0.7549) \sin(2(-20))$$

$$\sigma_{x'} = 1.06185 + (-0.8134) + (-0.4852)$$

⇒ $\sigma_{x'} = -0.23675 \text{ psi}$ Compression.

For $\sigma_{y'}$

$$\sigma_{y'} = -\frac{2.1237 + 0}{2} - \frac{(-2.1237) - 0}{2} \cos 2(20) - (0.7549) \sin 2(-20)$$

$$= -1.06185 + 0.8134 + 0.4852$$

$\sigma_{y'} = 0.23655 \text{ psi}$ Tension

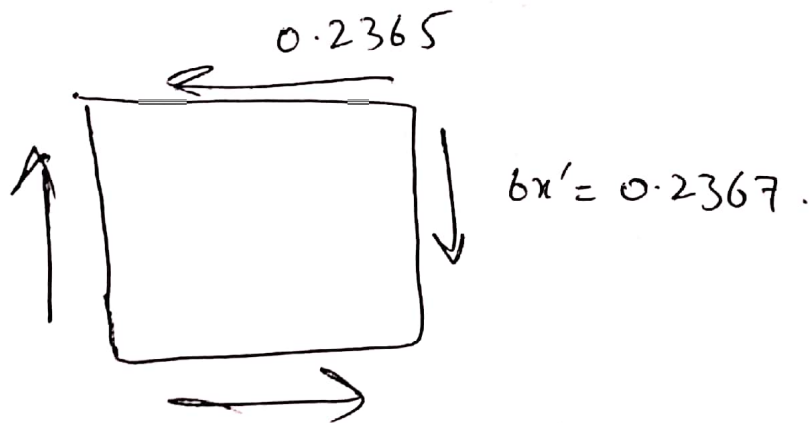
$$\tau_{x'y'} = \frac{-\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$= -\left(\frac{-2.1237 - 0}{2}\right) \sin 2(-20) + 0.7549 \cos 2(-20)$$

$$= -0.6825 + 0.5782$$

$$\tau_{x'y'} = +0.1043 \text{ psi}$$

→ Now the New stress state after 20° clockwise rotation is.



$$\tau_{x'y'} = 0.1043$$

Now we will find its principle stress.

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$= \frac{-2.1237}{2} \pm \sqrt{\left(\frac{-2.1237}{2}\right)^2 + (0.7549)^2}$$

$$= -1.06185 \pm 1.3028$$

$$\sigma_y = \sigma_1 = -1.06185 + 1.3028$$
$$= 1.24095$$

$$\rightarrow \sigma_x = \sigma_2 = -1.06185 - 1.3028$$
$$\Rightarrow -2.36465$$

Max in plane shear stress.

$$\tau_{xy} = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$
$$= \sqrt{\left(\frac{-2.1237}{2}\right)^2 + (0.7549)^2}$$
$$= \sqrt{1.1275 + 0.5698}$$

$\tau_{xy} = 1.3028 \text{ PSI}$

The co-ordinate of circle can be find by this

$$\left(\frac{bx+by}{2}, 0\right)$$

Centre coordinate.

$$(h, k) = \left(\frac{-2 \cdot 1237}{2}, 0\right)$$

$$= (-1.06185, 0)$$

Now to find radius of Mohr's circle,

$$r = \sqrt{\left(\frac{bx+by}{2}\right)^2 + [xy]^2}$$

$$r = \sqrt{\left(\frac{2 \cdot 1237 - 0}{2}\right)^2 + [(0.7549)]^2}$$

$$r = \sqrt{1.1275 + 0.5698}$$

$$r = 1.30280466705$$

(P14)

