

Mechanics Of Solid ...

Name : Afrasiyab

ID : 7899

Sec : A

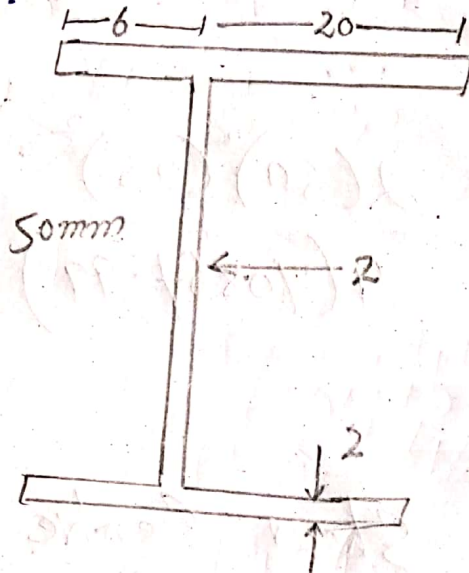
Submitted to : Engr Saqib

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(1)

Q No 1 (part-a)

Ans:



Required: Location of shear centre?

Solution: As we know that

$$e = \frac{h^2 b^2}{4I}$$

and
$$I = 2 \left(\frac{bh^3}{12} + Ay^2 \right) + \left(\frac{bh^3}{12} + Ay^2 \right)$$

$$\Rightarrow 2 \left(\frac{25(2)^3}{12} + (20 \times 2)(25)^2 \right) + \left(\frac{2(50)^3}{12} + 0 \right)$$

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$$I = 50034.66 + 20833$$

$$I = 70867.99 \text{ mm}^4$$

$$e = \frac{2(50)^2(25)^2}{4(70867.99)} = 11.02 \text{ mm}$$

So Shear Centre

$$e = 11.02 \text{ mm}$$

Ans

(3)

QNo1 (part B).

Ans ::

Given data

$$\text{Height} = 26 \text{ ft}$$

$$\text{Tangential stress} = 6000 \text{ psi}$$

$$\text{Specific weight of water} = 62.4 \frac{\text{lb}}{\text{ft}^3}$$

Required data

Thickness of wall of water tank = $t = ?$

Solution ::

$$P = \gamma h$$

$$6t = \frac{PD}{2t} = \frac{\gamma h \times D}{2t}$$

$$t = \frac{\gamma h D}{26t}$$

putting values

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$$t = \frac{62.4 \times 26 \times D}{(12)^3 \times 2(6000)}$$

$$t = \frac{62.4}{(12)^3} (26 \times 22) (22 \times 12)$$
$$2(6000)$$

~~$t = 0.001721296$~~

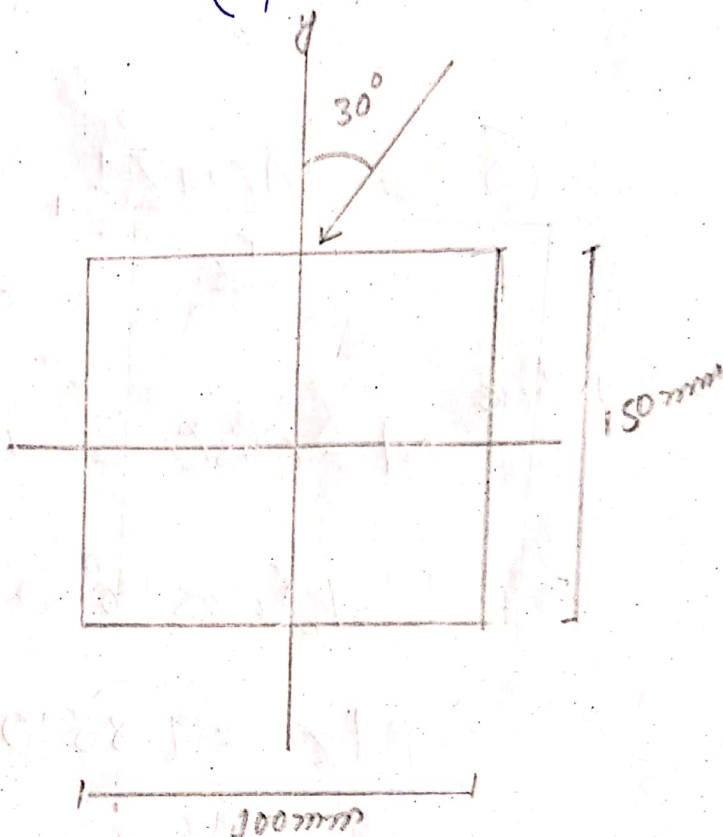
$t = 0.24 \text{ inch}$

Ans

5

Q No 2
x (part - a)

Ans:



Moment of Inertia

$$I_x = \frac{bh^3}{12} \Rightarrow \frac{0.1(0.15)^3}{12}$$

$$I_x \Rightarrow 2.8125 \times 10^{-5}$$

Now $I_y = \frac{bh^3}{12} \Rightarrow \frac{(0.15)(0.1)^3}{12}$

$$I_y = 1.25 \times 10^{-5}$$

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$$\sigma = \frac{M_z Y}{I_z} + \frac{M_y Z}{I_y}$$

$$\sigma = \frac{M \cos \theta}{I_z} + \frac{M \sin \theta}{I_y}$$

Where

$$M = P \cos \theta \Rightarrow P \cos \theta = M_z$$

$$= 12 \cos 30^\circ$$

$$M_z = 1.8510$$

$$M \sin \theta = P \sin \theta = M_y$$

$$12 \sin 30^\circ$$

$$M_y = -11.8563$$

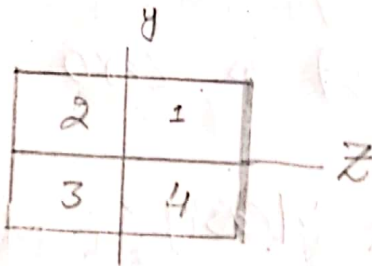
$$\sigma = \left(\frac{M_z}{I_z} \right) + \left(\frac{M_y}{I_y} \right)$$

$$\sigma = \frac{1.851}{2.812 \times 10^{-5}} + \left(\frac{-11.8563}{1.25 \times 10^{-5}} \right)$$

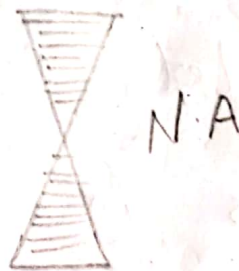
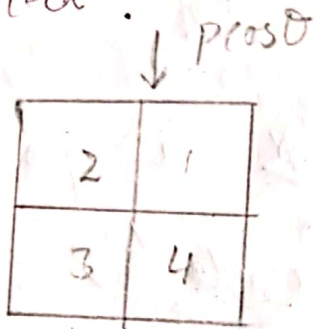
$$\sigma = 882678 \text{ N/m}^2$$

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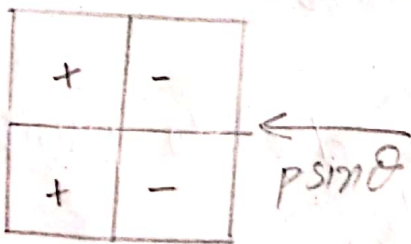
Sign Convention



* If we take compression as negative and tension as positive and the beam is simply supported.



Quadrant 1, 2 - ive
 // 3, 4 +ive
 N.A



Quadrant 1, 4 - ive
 // 2, 3 +ive

In case of Unsymmetrical loading ⑧
The Neutral axis lies of an angle
of " α ", The principle axis and
the algebraic sum of stress
at N.A is zero

$$\sigma = \frac{M \cos \theta \cdot y}{I_z} + \frac{M \sin \theta \cdot z}{I_y} \quad \text{--- (1)}$$

In this case N.A passes through
2, 4

$$\sigma = \frac{M \cos \theta \cdot y}{I_z} + \frac{M \sin \theta \cdot z}{I_y}$$

Let consider a point "A" on NA
lies in Quadrant 2, where

• Bending stress due to $P \cos \theta$ is
Compressive

* Bending stress due to $P \sin \theta$ is
Tensile.

eq (1)

(9)

$$0 = \frac{-M \cos \theta y_A}{I_z} + \frac{M \sin \theta z_A}{I_y}$$

$$\frac{M \cos \theta y_A}{I_z} = \frac{M \sin \theta z_A}{I_y}$$

$$\frac{y_A}{z_A} = \frac{I_z \sin \theta}{I_y \cos \theta}$$

$$\tan \alpha = \frac{I_z}{I_y} \tan \theta \quad \text{--- (2)}$$

Now put values of I_z , I_y and θ in eq (2)

$$\tan \alpha = \frac{I_z}{I_y} \tan 30^\circ$$

$$\Rightarrow \frac{2.8125 \times 10^{-5}}{1.25 \times 10^{-5}} \tan 30^\circ$$

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$$\tan \alpha = -14.4129$$

$$\alpha = \tan^{-1}(-14.4129)$$

$$\alpha = 1.5^\circ$$

$$\alpha = 1^\circ 30' 5''$$

Ans.

Q No 2 (part b)

(11)

Ans:

Given data:

Length of beam = 16 ft

Angle of inclination = 60°

$$I_x = 112.6 \text{ in}^4$$

$$I_y = 18.7 \text{ in}^4$$

$$S_x = 5000 \text{ Psi}$$

$$S_c = 12000 \text{ Psi}$$

Required data:

Maximum Load ?

Solution:

There are two forces
Compression as well as tensile
which will reduce the effect of

(12)

of each other. So we will calculate stress at A and C So,

$$\sigma_A = \frac{M_x y}{I_x} + \frac{M_y x}{I_y} \text{ (compression)}$$

$$\sigma_C = \frac{M_x y}{I_x} + \frac{M_y x}{I_y} \text{ (Tension)}$$

Now M_x and M_y

$$M_x = P \overset{=48}{\cos 60} (16 \times 12)$$

$$M_x = 48 p \cos 60$$

$$M_y = P \sin \theta (60)$$

$$M_y = 48 \sin 60$$

$$\sigma_A = \frac{M_x y}{I_x} + \frac{M_y x}{I_y}$$

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$$\sum A = \frac{48 \cos 60^\circ \times 3.07 + 48 \sin 60^\circ \times 3.07}{1126 + 18.7}$$

$$P = 1638.6 \text{ lb}$$

Now

$$S_c = \frac{M_x y}{I_x} + \frac{M_y x}{I_y}$$

$$5000 = 48 \cos 60^\circ + 593 + \frac{48 \sin 60^\circ \times 0.5}{18.7}$$

$$P = 2104.9 \text{ lb}$$

Max Stress load \Rightarrow 1638.6 lb

Q No 3

14

Ans: Given data

$$L = 10 \text{ ft}$$

$$E = 10.3 \times 10^6$$

$$b = 0.75$$

$$\text{Factor of safety} = 2$$

$$h = 0.75$$

Required data

- 1) Safe load at hinged ?
- 2) " " at fixed ?

Solution

1) for hinged
 $l_e = L$

$$I = I_x = \frac{(0.75)(2)^3}{12} = \underline{\underline{0.5 \text{ in}}}$$

$$P_{cr} = \frac{\pi^2 EI n^2}{L e^2}$$

$$\Rightarrow \frac{(1)^2 (10.3 \times 10^6) (0.5) (3.14)^2}{(10 \times 12)^2}$$

$$P_{cr} = 3526.176 \text{ lb}$$

P safe load $\Rightarrow \frac{P_{cr}}{\text{factor of safety}}$

$$\Rightarrow \frac{3526.176}{2}$$

$$\Rightarrow 1763.088 \text{ lb}$$

2) For fixed $\rightarrow l_e = \frac{l}{2} = 5 \text{ ft}$

$$I = I_y = \frac{2 \times (0.75)^3}{12} \Rightarrow 0.07 \text{ in}^4$$

$$P_{cr} = \frac{n^2 EI \pi^2}{L^2}$$

Putting values

$$P_{cr} = \frac{(1)^2 (10.3 \times 10^8) (0.07) (3.14)^2}{(5 \times 12)^2}$$

$$P_{cr} = 1974.658 \text{ lb}$$

Safe load $\Rightarrow \frac{1974.658}{2}$

$$\Rightarrow 987.3293 \text{ lb}$$

Ans