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Sec = A

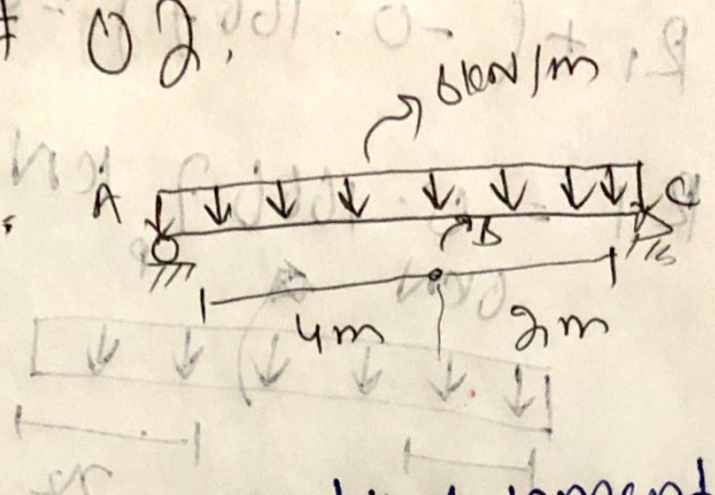
Date = 26 - 6 - 2026

Sir = Amjad gislam.

Paper = Structure Analyst

Question # 02.

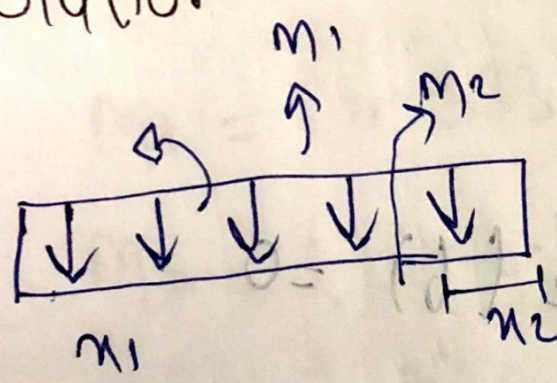
Given data:



Required:

Slope and displacement at point B

Solution:-



$$\Sigma F = 0$$

$$0 = \Sigma M + \dots$$

$$R_1 + R_2 = 0 \quad \text{--- (1)}$$

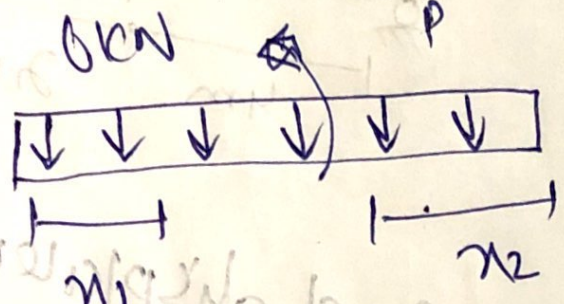
$$\Sigma M_A = 0$$

$$1 + R_2(6) = 0$$

$\Rightarrow -0.16607$ put in eq (1)
P.T.O

$$R_1 + (-0.1667) = 0$$

$$R_1 = 0.16667 \text{ kN}$$



(b)

$$R_1 + R_2 = 1$$

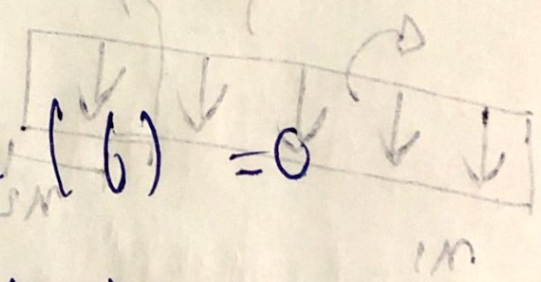
$$\sum M_A = 0$$

$$-(1)(4) + R_2(6) = 0$$

$$R_1 = 0.6667 \text{ kN}$$

$$R_2 = 1 - 0.6667 \text{ kN}$$

$$R_2 = 0.333 \text{ kN}$$



$$0 = \sum R + 1R$$

$$0 = \sum M_A$$

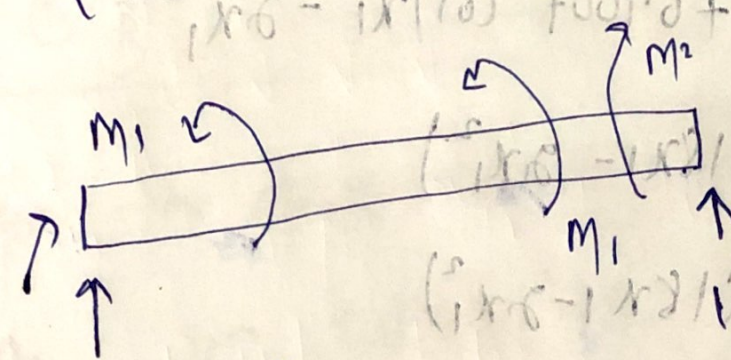
$$0 = \sum R + 1R$$

$$0 = \sum M_A$$

(2)

$$M_1 = (18 + 0.1667 M_1) x_1 - 2x_1^2$$

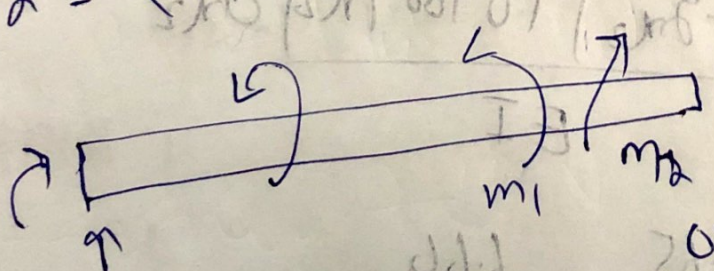
$$M_2 = (18 - 0.1667 M_1) x_2 - 2x_2^2$$



$$18 + 0.1667$$

$$M_1 = (0.3333 P + 18) x_1 - 2x_1^2$$

$$M_2 = (18 - 0.1667 M_1) x_2 - 2x_2^2$$



$$0.3333P + 18$$

The displacement shown in the figure "a" above.

(3)

9

$\frac{\delta m_1}{\delta m_1} = 0.1667 \lambda_1$ and $\frac{\delta m_2}{\delta m_2} = 0.1667 \lambda_2$

set $m' = 0$ then

$m_1 = (18 + 0.1667 \omega) \lambda_1 - 2 \lambda_1$

$\rightarrow m_1 = (18 \lambda_1 - 2 \lambda_1^2)$

$\rightarrow m_2 = (18 \lambda_2 - 2 \lambda_2^2)$

$\delta M = 0 + 21$

$\delta B = \int_0^1 \left(\frac{\delta m_1}{\delta m_1} \frac{\delta m_1}{\delta x_1} \right) dx_1 = \int_0^1 \frac{(18 \lambda_1 - 2 \lambda_1^2) (0.1667 \lambda_1)}{EI} dx_1$

$\int_0^1 \frac{(18 \lambda_2 - 2 \lambda_2^2) (0.1667 \lambda_2)}{EI} dx_2 = \delta M$

$\delta B = \frac{42.65}{EI} + \frac{6.66}{EI}$

$\delta B = \frac{49.31}{EI}$

$$\textcircled{1} \quad \beta = \frac{49.31}{(200 \times 10^6 \text{ kPa})(0.0006)} = 0$$

$$\boxed{\textcircled{1} \quad \beta = 0.4411 \text{ rad}}$$

→ for the displacement function are shown in figure "b"

$$\frac{\partial m_1}{\partial m_p} = 0.333 x_1 \quad \text{and} \quad \frac{\partial m_2}{\partial p} = 0.6667 x_2 \quad \text{at } p=0$$

then $M_1 = (18x_1 - 2x_1^2) \text{ kN}\cdot\text{m}$

$$M_2 = (18x_2 - 2x_2^2) \text{ kN}\cdot\text{m}$$

thus,

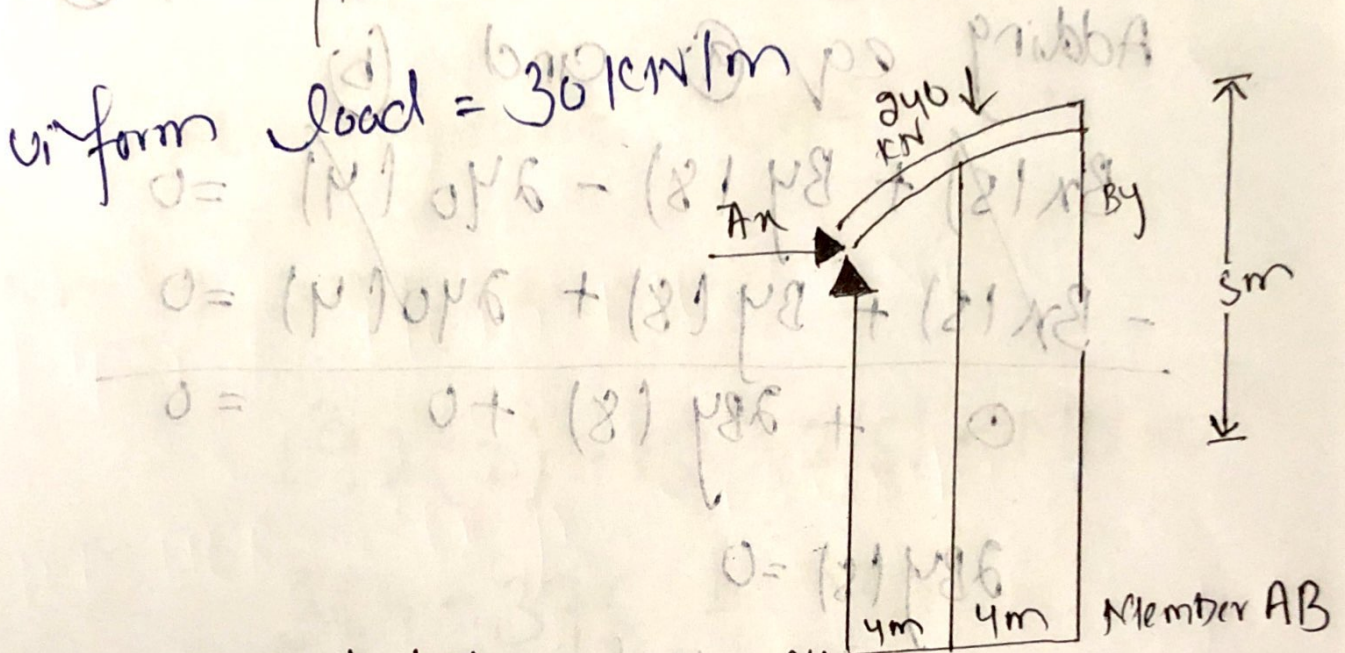
$$\Delta_B = \int_0^L m \left(\frac{\partial m}{\partial p} \right) \frac{dx}{EI}$$

$$\Delta_B = \int_0^4 \frac{(30x_1 - 2x_1^2)(0.333x_1) dx}{EI} + \int_0^2 \frac{(30x_2 - 2x_2^2)(0.6667x_2) dx}{EI}$$

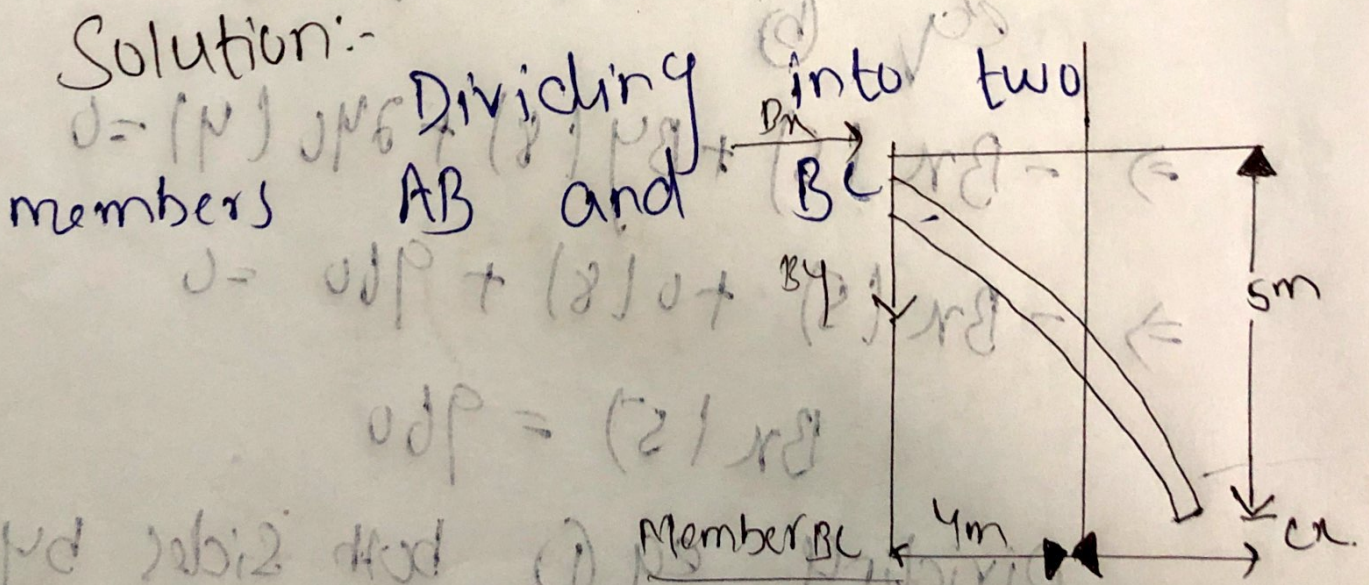
$$\Delta_B = \frac{218.5}{EI} \Rightarrow \frac{218.5}{(200 \times 10^6)(0.0006)} = 0.018 \text{ m} = \boxed{18 \text{ mm}}$$

Q NO 4

Ans: (1) $\sum M_A = 0$
 $\sum M_B = 0$
 $\sum M_C = 0$
 (2) Given data:-



Required data:-
 Internal Moment at D = ?



AB:-
 $\sum M_A = 0$
 $B_x(5) + B_y(8) - 240(4) = 0$

Be:

$$\hookrightarrow + \Sigma M_c = 0$$

$$- B_x (5) + B_y (8) + 240 (4) = 0 \rightarrow (b)$$

Adding eq (a) and (b)

$$B_x (5) + B_y (8) - 240 (4) = 0$$

$$- B_x (5) + B_y (8) + 240 (4) = 0$$

$$0 + 2B_y (8) + 0 = 0$$

$$2B_y (8) = 0$$

$$\Rightarrow B_y = 0 \text{ kN}$$

putting the value of "B_y" in eq (b)

$$\Rightarrow - B_x (5) + B_y (8) + 240 (4) = 0$$

$$\Rightarrow - B_x (5) + 0 (8) + 960 = 0$$

$$B_x (5) = 960$$

Dividing eq (1) both sides by

$$\frac{B_x(5)}{5} = \frac{960}{5}$$

$$B_x = 192 \text{ kN}$$

"At segment DB"

$\sum MD = 0$

$$192(2) - 150(2.5) - MD = 0$$

$$384 - 375 = MD$$

$$MD = 9 \text{ kN}\cdot\text{m}$$

ANS.

$$(P \cdot r \cdot u)$$

$$J_o = FH = MCF = \frac{M \cdot r}{J}$$

(4)

Q NU 3 $\frac{W_0 L^2}{2} = \frac{P L^2}{2}$

Given data: $W_0 =$ uniform load $= 460 \text{ lb/ft}$

$h = 10 \text{ ft}$, $L = 15 \text{ ft}$

Required:

Equation of curve

Reaction force in cable = ?

Solution: we know that

$y = \frac{h}{L^2} x^2$ putting the values

$$y = \frac{h}{L^2} x^2 = \frac{10}{(15)^2} x^2 = \frac{0.44 x^2}{2 \times 10}$$

$$T_0 = FH = \frac{W_0 L^2}{2h} = \frac{460 \times (15)^2}{2 \times 10}$$

Pr. 10

$$I_0 = 4500 \text{ lb} = 4.5 \text{ k}$$

$$I_B = I_{\text{max}} = \sqrt{(FH)^2 + (w_0L)^2}$$

$$= \sqrt{(4500)^2 + (400 \times 15)^2}$$

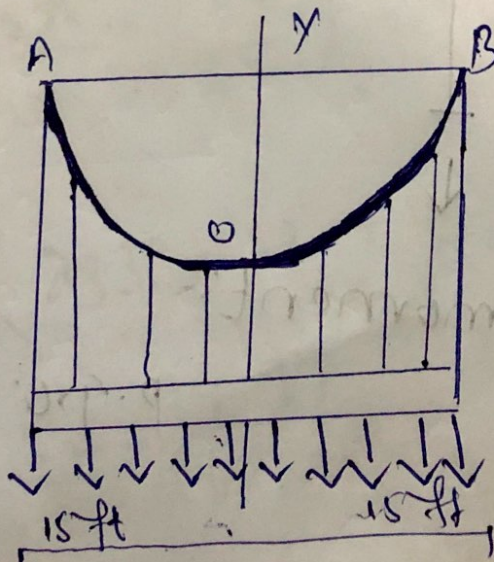
$$I_{\text{max}} = 7500 \text{ lb} = 7.5 \text{ k}$$

Now " I_{max} " by another equation

$$I_B = I_{\text{max}} = w_0L \sqrt{\left(1 + \frac{L^2}{24h}\right)^2} \Rightarrow$$

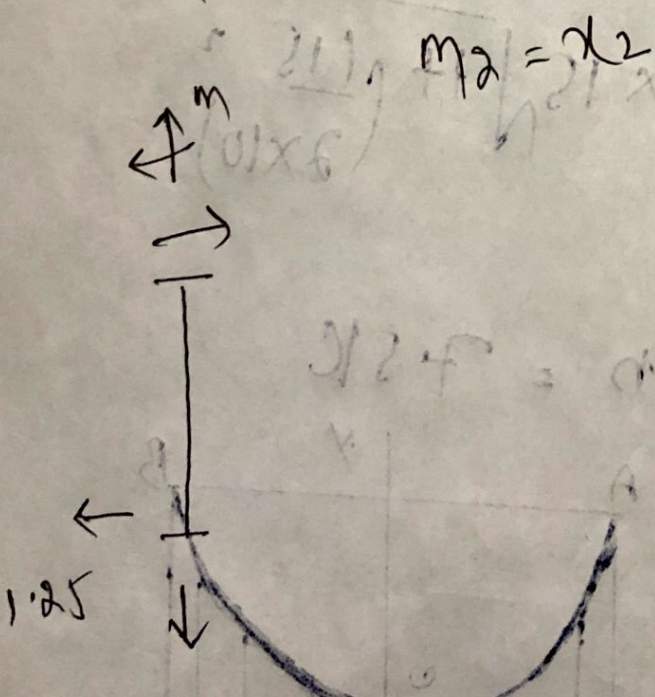
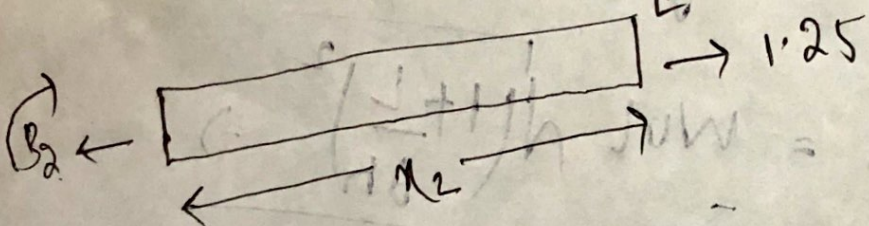
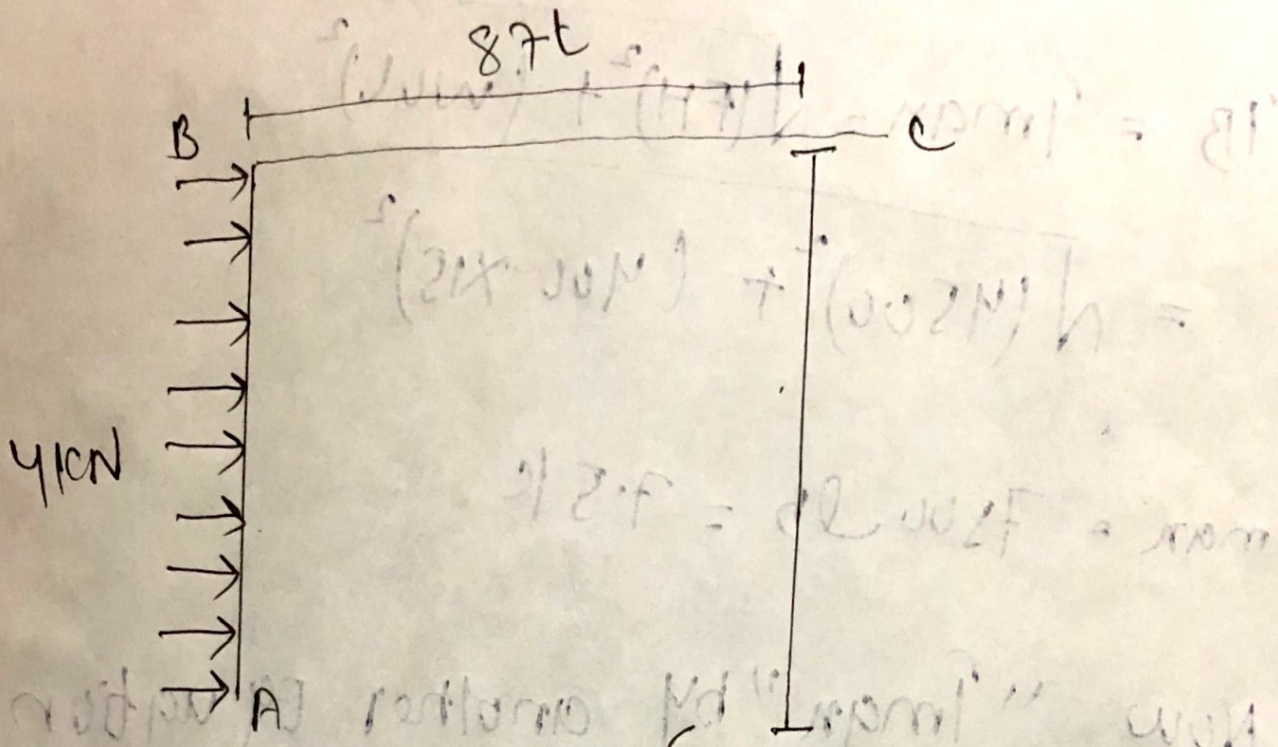
$$= 400 \times 15 \sqrt{1 + \frac{(15)^2}{2 \times 10}}$$

$$I_{\text{max}} = 7500 \text{ lb} = 7.5 \text{ k}$$



P. 1.6)

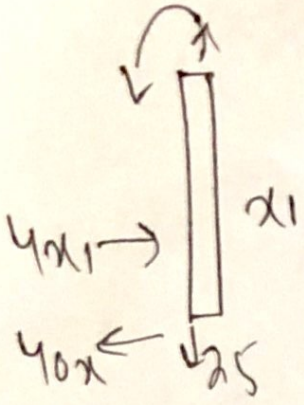
Question # 1



Real moment:-

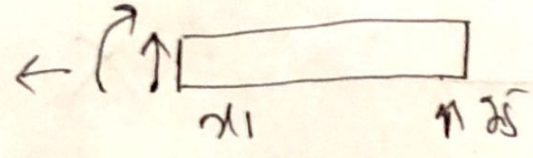
P.T.O

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Virtual work = EI



$$m_2 = 25x_2$$

$$M = \frac{y_0x_1 - \frac{1}{2}x_1(x_2)}{y_0x_1 - 2x^2}$$

Now put virtual work

$$1 \cdot \Delta C = \int_0^2 m \frac{\delta m}{\delta} dx$$

$$= \int_0^{16} 1x_1 \left(\frac{y_0x_2 - 2x^2}{\delta} \right) dx + \int_0^8 \frac{(1 \cdot 25x_1)(25x_2)}{EI} dx_2$$

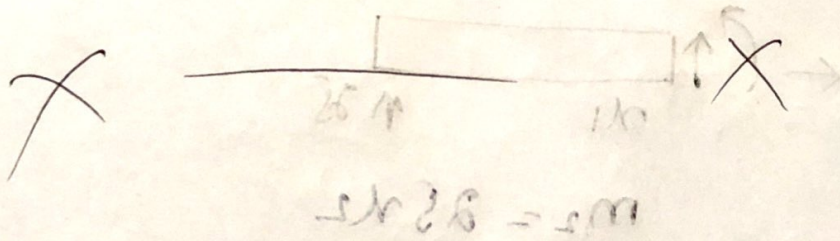
$$\Delta C = \frac{1}{EI} \left(\frac{y_0x^3}{3} - \frac{2x^3}{4} \right) \int_0^{16} + \left(\frac{31 \cdot 25x_2^3}{3} \right) \int_0^8$$

$$\Rightarrow \Delta C = \frac{1}{EI} (23333.33 + 10666.66)$$

$$\Delta C = \frac{33999.99}{(200) \cdot (60 \times 10^6)}$$

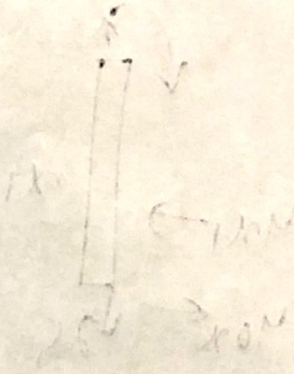
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$$\Delta \bar{I} = 2.833 \times 10^6 \text{ inch}^4$$



0

8



$$\int_{-10}^{25} p \cdot T \cdot x \, dx = 0$$

$$\int_{-10}^{25} p \cdot T \cdot x \, dx = 0$$

Now for mirror work

$$\int_{-10}^{25} p \cdot T \cdot x \, dx = 0$$

$$\int_{-10}^{25} p \cdot T \cdot x \, dx = 0$$

$$\int_{-10}^{25} p \cdot T \cdot x \, dx = 0$$