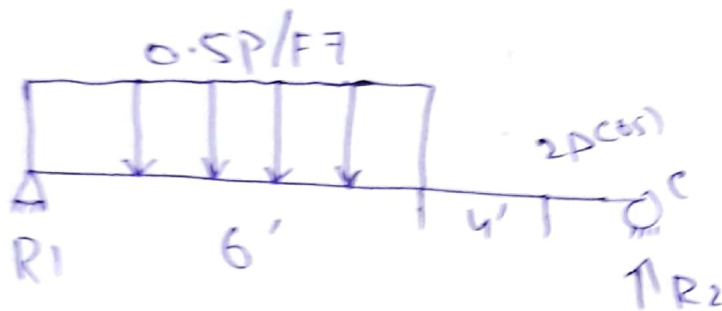
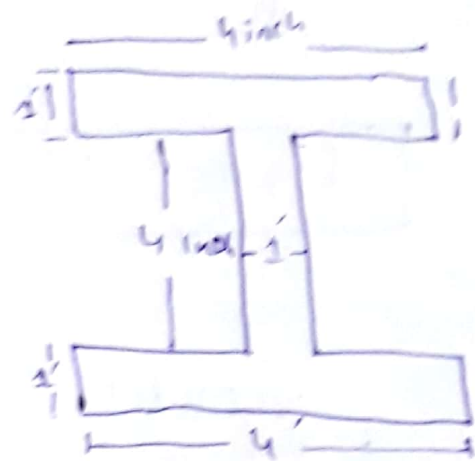
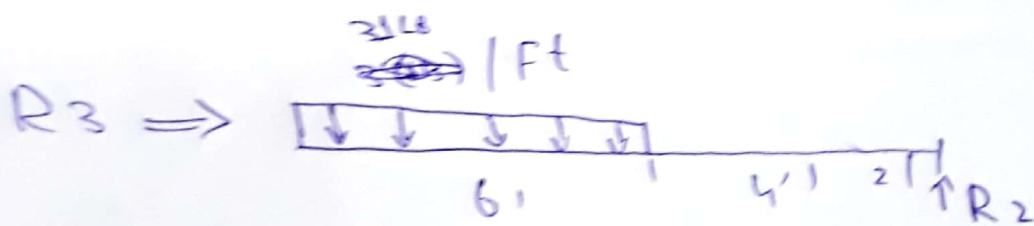


Name	Tariq Khem
ID	7962
Section	B
exam	mid term
Assignment	MOS = 2

P①



Now put the value of  $P = 62$



To find unknown Reaction at the Support Apply equilibrium eq

$$\sum F_x = 0 \quad \text{i.e. } R_3 = 0$$

$$\sum F_y = 0 \quad \uparrow, \downarrow$$

$$R_1 + R_2 = (31)(6) \text{ Lb} + 124 \text{ Lb}$$

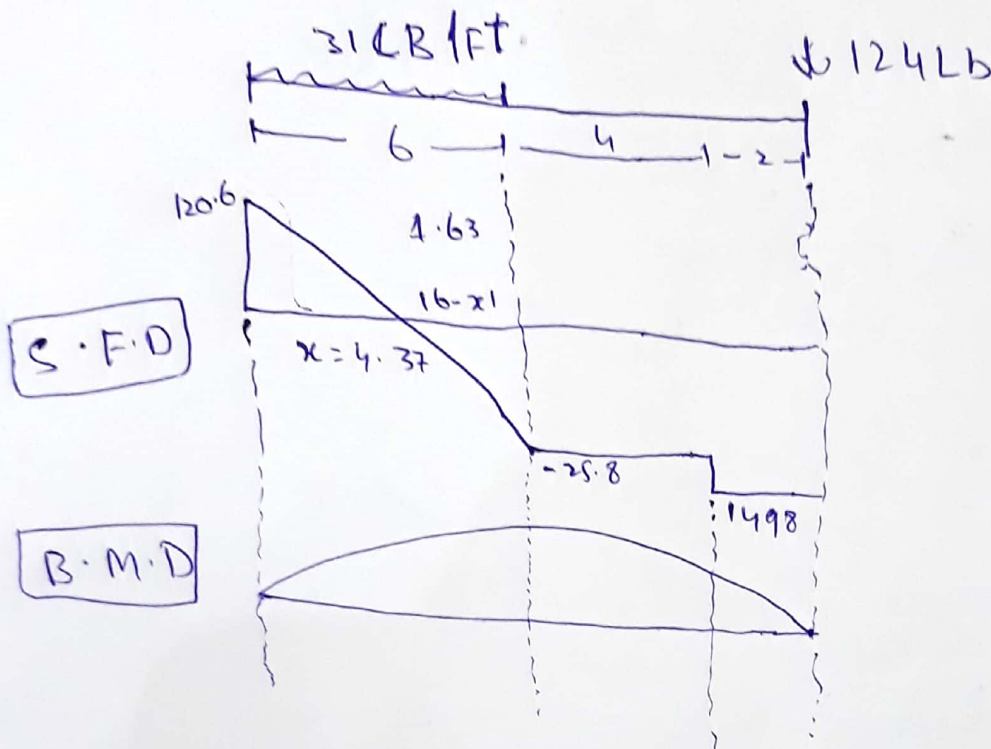
$$R_1 + R_2 = 310 \quad \rightarrow \textcircled{1}$$

$$\text{Next } \sum MA = 0 \quad \uparrow + \circlearrowleft$$

$$R_2 \times 12 - 10 \times 124 - (31 \times 6) \times 6/2 = P(2)$$

$$12R_2 = 1240 + 558$$

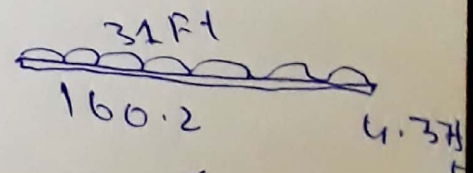
$$R_2 = 149.8 \text{ LB}$$



$$\frac{x \cdot 120.6}{1706} = \frac{6 \cdot x}{25.8}$$

$$x = 4.37$$

Now determine the value of moment at 4.37 ft



$$M \quad 4.37 \text{ ft} - 160.2 \times 4.37 + (31 \times 4.37) \times \left(\frac{4.37}{2}\right) = 0$$

$$M = 4.37 \text{ ft} - 700.0 \text{ lb} + (135.47 \text{ lb})(2.185) \text{ ft} \quad P \text{ (3)}$$

$$M = 4.37 \text{ ft} = 404.072 \text{ LBft}$$

For Shear Stress we have

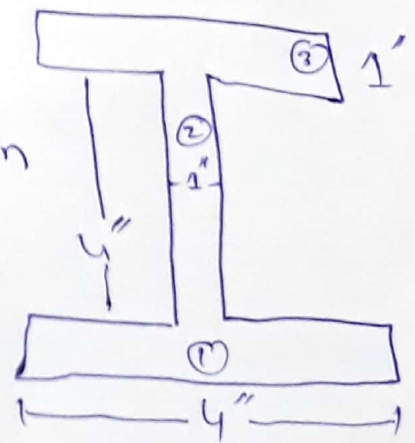
$$\tau = \frac{VQ}{It}$$

$$I = ?$$

As the given figure is symmetrical along the both axis.

$$\text{So } \bar{x} = 4/2 = 2 \text{ in } \Rightarrow \bar{y} = 6/2 = 3 \text{ in}$$

ie  $(\bar{x}, \bar{y})$  (center of gravity)



$$\text{Area point (1)} = 4 \times 1 = 4 \text{ in}^2$$

$$\text{Area point (2)} = 4 \times 1 = 4 \text{ in}^2$$

$$\text{Area point (3)} = 4 \times 1 = 4 \text{ in}^2$$

Moment of inertia about x-axis (centroidal)  $I_{xx}$

Determine distance b/w c.g. of the whole section and the corresponding parts.

Let  $G_1, G_2, G_3$  be the center of gravity of point ①, ②, ③ and  $k_1, k_2, k_3$  be the distance b/w  $\bar{y}$  and  $y_1, y_2, y_3$  respectively

So

$$k_1 = \bar{y} - y_1 = 3 - 0.5 = 2.5 \text{ in}$$

$$k_2 = \bar{y} - y_2 = 3 - 3 = 0 \text{ in}$$

$$k_3 = \bar{y} - y_3 = 3 - 0.5 = 2.5 \text{ in}$$

So

$$I_{xx} = \frac{b_1 h_1^3}{12} + a_1 k_1^2 + \frac{b_2 h_2^3}{12} +$$

$$a_2 k_2^2 + \frac{b_3 h_3^3}{12} + a_3 k_3^2$$

$$I_{xx} = \frac{(4 \times 2)^3}{12} + 4(2.5)^2 + \frac{(1)(4)^3}{12} + a_2(0) + \frac{4(1)^3}{12} + 4(2.5)^2$$

$$I_{xx} = \frac{4}{12} + 25 + \frac{64}{12} + \frac{4}{12} + 25$$

$$I_{xx} = \frac{4 + 12(25) + 64 + 4 + 12(25)}{12}$$

$$I_{xx} = 56 \text{ in}^4$$

$$\text{Now } I_{yy} = \frac{b_1^3 h_1}{12} + \frac{b_2^3 h_2}{12} + \frac{b_3^3 h_3}{12}$$

$$I_{yy} = \frac{(4)^3 (1)}{12} + \frac{(1)^3 (4)}{12} + \frac{(4)^3 (1)}{12}$$

$$I_{yy} = \frac{64}{12} + \frac{4}{12} + \frac{64}{12}$$

$$I_{yy} = \frac{64 + 4 + 64}{12}$$

$$I_{yy} = 11 \text{ in}^4$$

Find the shear stress at various

Point we know that  $\tau = \frac{VQ}{IT}$

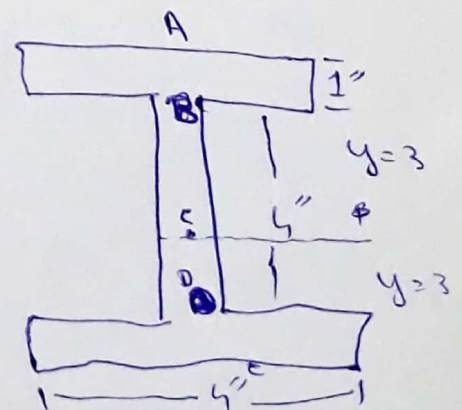
① Shear stress at Point A

at the top fiber

$$\tau = \frac{VQ}{IT}$$

$$\tau = \frac{149.8(0)}{67(4)} = 0 = \bar{y} \cdot A$$

$$\tau = 0$$



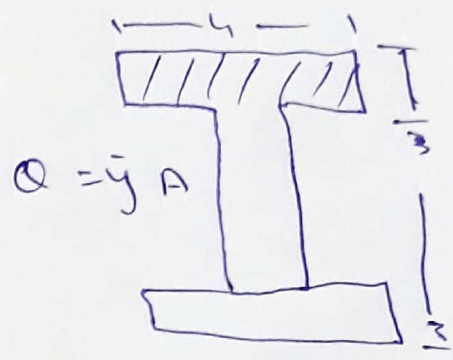
Here  $Q = 0$   
 Because no Area  
 of the section  
 exist above point  
 i.e.  $Q = \bar{y} \cdot A = 0(\bar{y}) = 0$

### Shear stresses at point B

$$\tau = \frac{VQ}{Ib}$$

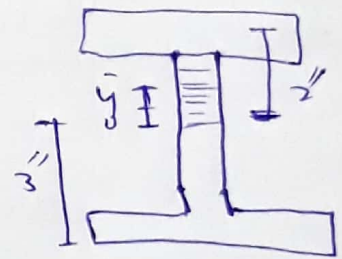
$$\tau = \frac{(149.8) \times (4 \times 1) (3 - 0.5)}{67 \times 4}$$

$$\tau = 5.57 \text{ LB/in}^2$$



### Shear stress at point c i.e at

$$\tau = \frac{VQ}{It}$$



$$= 149.5 \frac{[4 \times 1 (3 - 0.5) + (1 \times 2) (2 \times -)]}{67 \times 1}$$

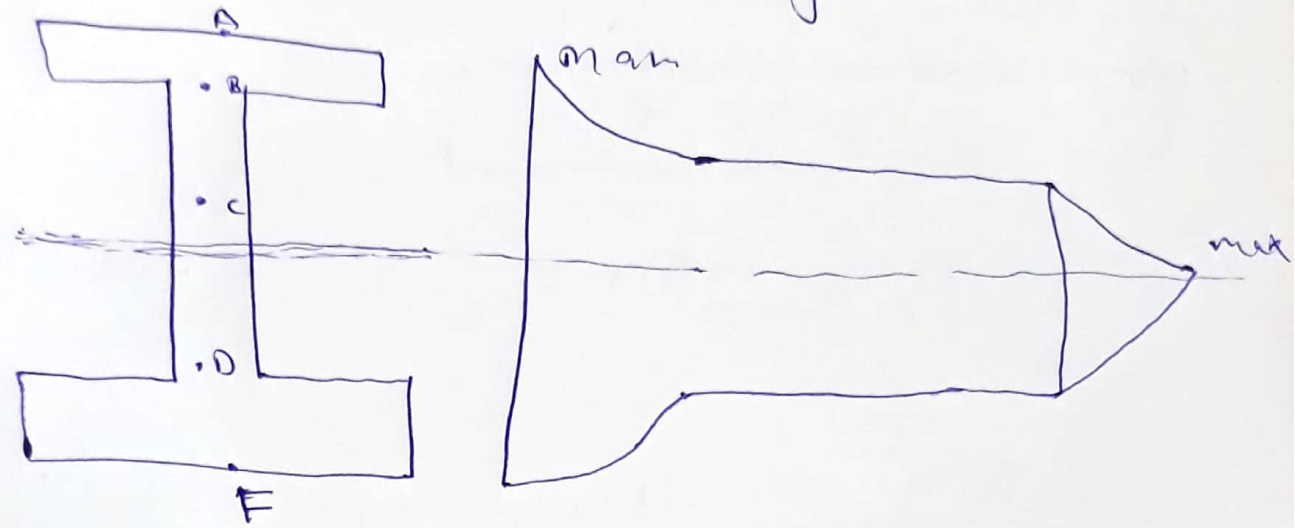
$$\tau = \frac{149.5 \times 12}{67}$$

$$\tau = 26.776 \text{ LB/in}^2$$

(iv) Shear stress at point D and E will be the same because of symmetry.

Note The max shear stress value occurs at the neutral axis and minimum value at the top of the section.

# Shear Stress Diagram.



# Flexural Stress Determination

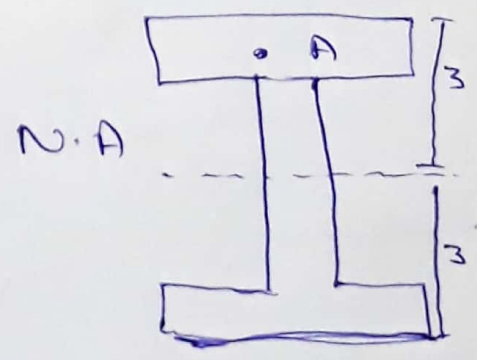
$$S = \frac{My}{I}$$

Flexural stress at the top fiber point (A)

$$S = \frac{My}{I}$$

$$S = \frac{404.072 \times 3}{67}$$

$$S = 18.09 \text{ Lb/in}^2$$



Flex stress at point B

$$S = \frac{My}{I}$$

S/F MP

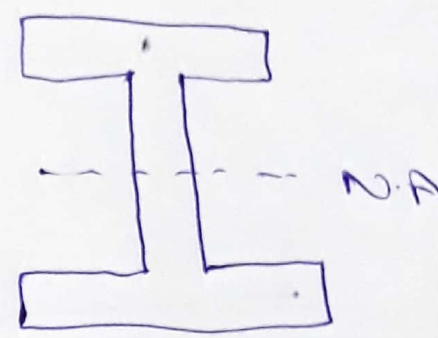


$$S = \frac{404.072 \times (3-1)}{67}$$

$$S = 1206 \text{ L B/in}^2$$

Flexural stress at Neutral axis

$$S = \frac{m y}{I} = \frac{404.072 \times 6}{70}$$



$$S = 0 \text{ Lb/in}$$

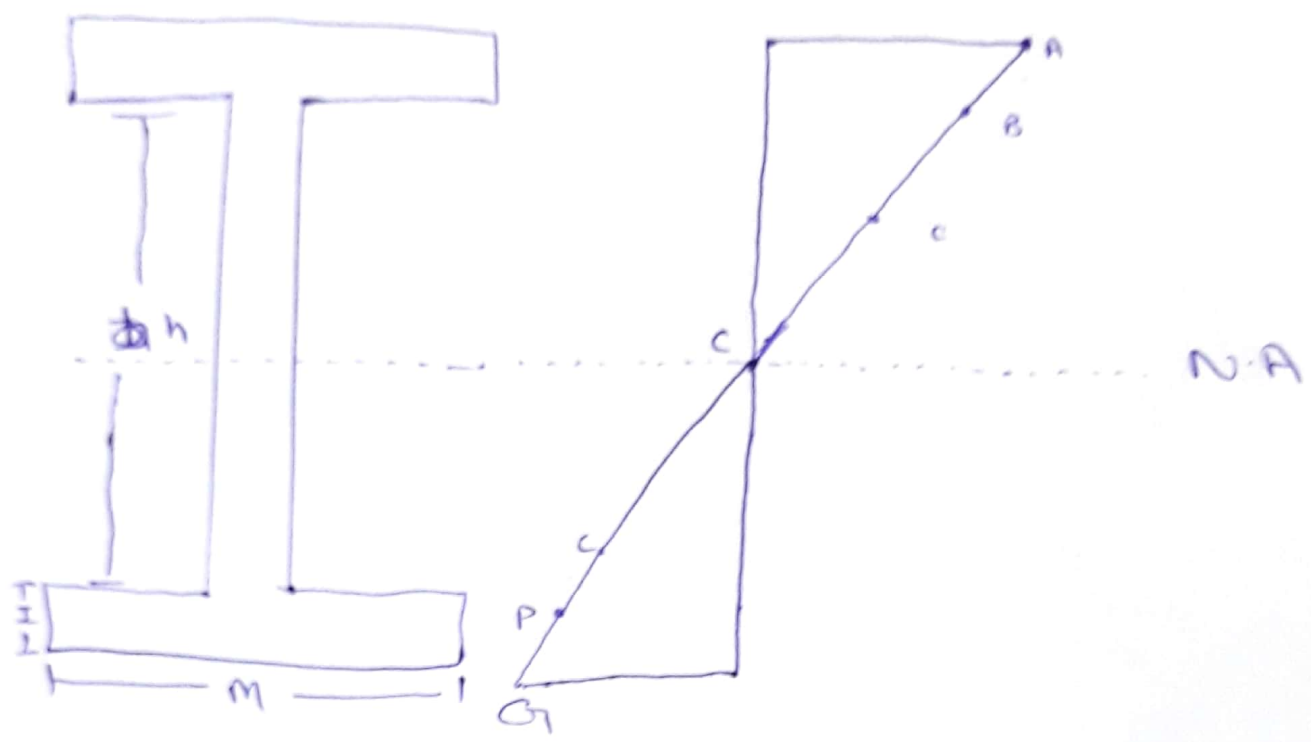
Flexural stress value at point E, F and G

Remain the same because of symmetry

The upper portion above N.A shows tension and below the N.A shows compressions

Note

The flexural stress values is maximum at extreme top and bottom fiber at zero at N.A



Stress State

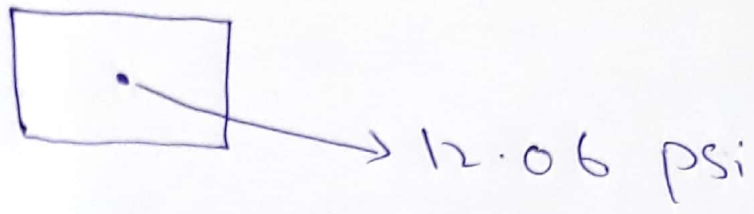
Find stress state of  
 a point c  
 3/4 h left support 1 inch below  
 from top fiber  
 point c  
 stress at

$$G = 12.06 \text{ PSI}$$

S. stress at point c

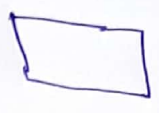
$$\bar{\tau} = 20.776$$

consider point c is a  
 planar element

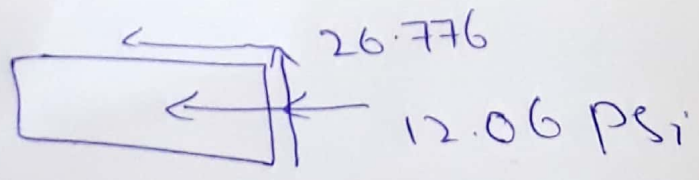
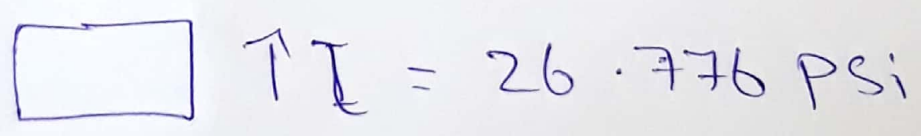


$\tau = 26.28$  psi is compressive

Because Point c lies in compression  
 zero  $\sigma$  Beam cross section



97 Point c lies below the centroid then stress would be tensile



Combine stress on 2-D element

Find its principle stress.

we have also find

$\sigma_x = 12.06$

$$b_y = 0$$

P (11)

$$\tau_{xy} = 26.776$$

Principle stress equation.

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\frac{(\sigma_x - \sigma_y)^2}{4} + \tau_{xy}^2}$$

$$\sigma_{1,2} = \frac{12 \cdot 06 + 0}{2} \pm \sqrt{\frac{(12 \cdot 06 + 0)^2}{4} + (26.776)^2}$$

$$= -6 \cdot 03 \pm \sqrt{36 \cdot 36 + 716.9}$$

$$= -6 \cdot 03 \pm 27.44$$

$$= -6 \cdot 03 - 27.44 = -33.47$$

$$= -6 \cdot 03 + 27.44 = 21.41$$

$\alpha_p = ?$

$$\tan 2\alpha = \frac{\tau_{xy}}{(\sigma_x - \sigma_y)/2}$$

$$\tan 2\alpha_p = \frac{26.776}{(12 \cdot 06 - 0)/2}$$

$$\tan 2\alpha_p = \frac{26.77}{6 \cdot 03}$$

$$2\alpha_p = \tan^{-1} 4.43$$

$$\sigma_p = \frac{77.30}{2}$$

P12

$$\sigma_p = 38.65$$

Put in general equation.

$$\sigma_{max} = \frac{(-12.06 + 0)}{2} + \left( -\frac{12.06}{2} \cos(2) (38.65) \right)$$

$$\begin{aligned} \sigma_{max} &= -6.03 - 6.03(0.229) \\ &= -6.03 - 1.382 \end{aligned}$$

$$\sigma_{max} = -7.412 = \sigma_x = 12.06$$

Max in plane shear stress.

In this case

$$\tan 2\theta_s = \frac{(\sigma_x - \sigma_y)/2}{\tau_{xy}}$$

$$\tan 2\theta_s = \frac{(-12.06 - 0)/2}{26.776}$$

$$\tan 2\theta_s = \frac{6.03}{26.776}$$

$$+ \sin^{-1}(0.22)$$

$$\boxed{Q_s = 6.20}$$

Put in these general equation

$$\bar{z} x' y' = - \left[ \frac{bx}{2} - by \right] \sin 2\alpha + \bar{z} xy \cos 2\alpha$$

$$\bar{z} x' y' = - \left( \frac{12 \cdot 06 - 0}{2} \right) \sin 2(12 \cdot 00)$$

$$26.776 \cos 2(12 \cdot 06)$$

$$= -6.03 \sin 2(12 \cdot 06) + 26.776 \cos 2(12 \cdot 06)$$

$$\boxed{\bar{z} x' y' = 320.18}$$

To draw Mohr's circle  
center coordinate.

$$\begin{aligned} (h, k) &= (bx, by, 0) \\ &= \left( -\frac{12 \cdot 06}{2}, 0, 0 \right) \end{aligned}$$

$$= (6.04, 0)$$

$$r = \frac{(b_x \cdot b_y)^2}{2} + \overline{xy} =$$

$$r = \sqrt{\frac{(-12 \cdot 06 - 0)^2}{2} + (26 \cdot 776)^2}$$

$$r = \sqrt{7533}$$

$r = 27.44$
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