

NAME :

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ID :

7956

SECTION :

B

SEMESTER :

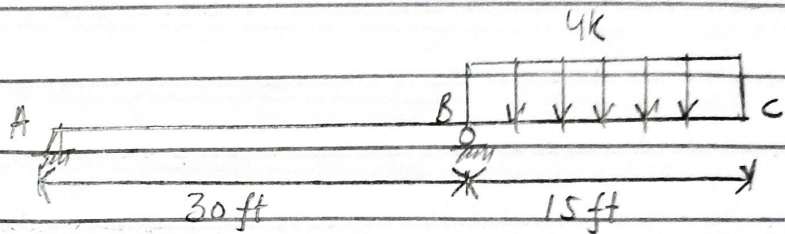
4th

ASSIGNMENT NO; 03

①

Q: No: 01

" DETERMINE THE SLOP AND DISPLACEMENT at C. USE MOMENT AREA THEORM.



To find;

Slop (θ_c) = ?

Displacement Δ_c = ?

Solution:

Firstly we have to draw M/EI diagram

So;

$$\sum \curvearrow + \epsilon M_A = 0$$

$$V_B \times 30 + (4 \times 15) \times 37.5 = 0$$

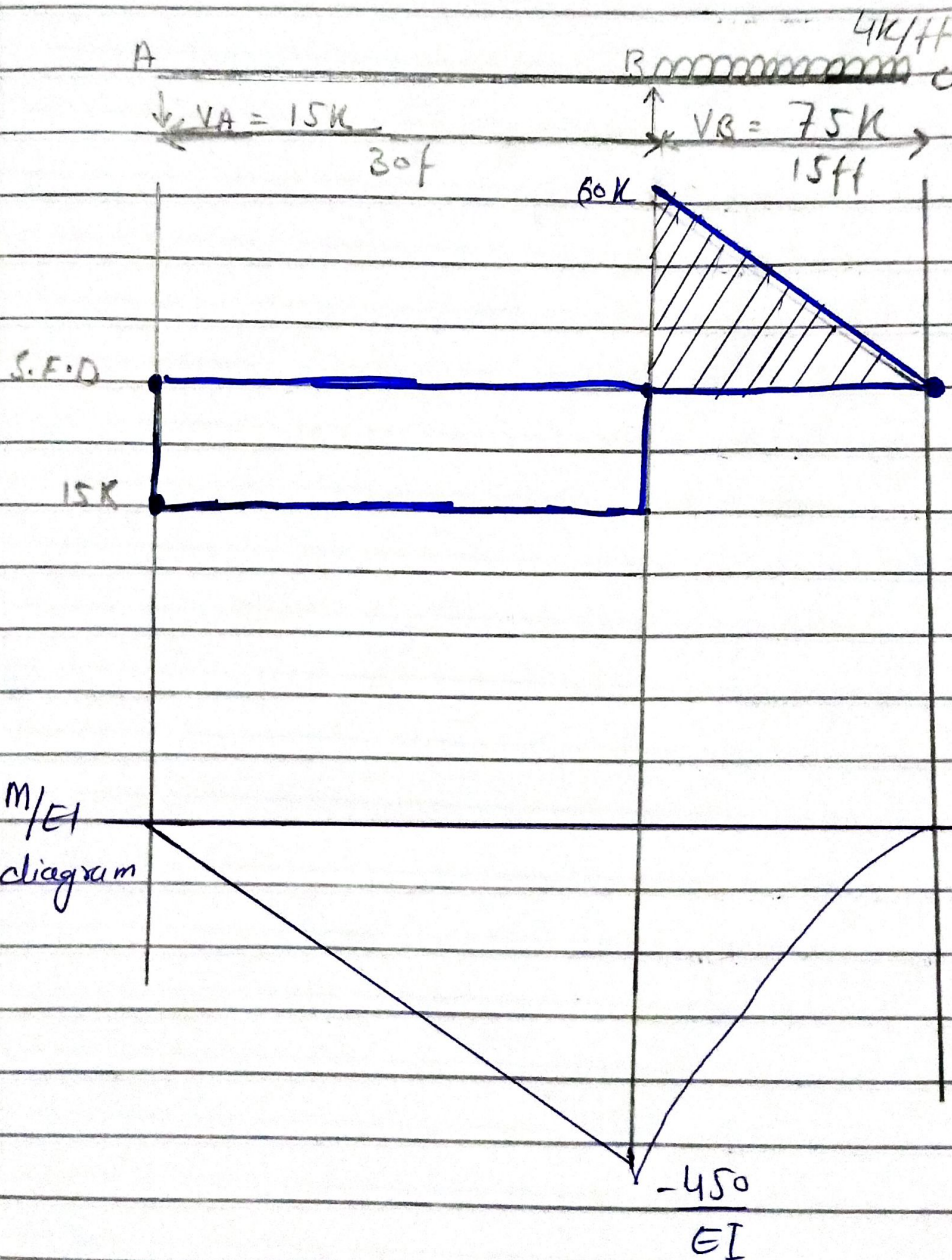
$$V_B = 75k$$

②

$$\sum M_B = 0$$

$$V_A \times 30 + (4 \times 15) \times 7.5 = 0$$

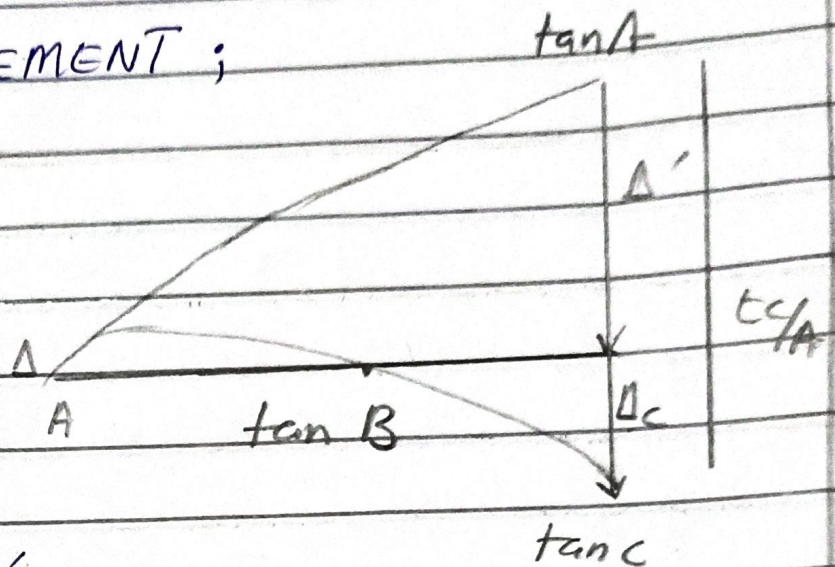
$$\rightarrow V_A = -15K$$



→ Thus M/EI diagram consists of triangular and parabolic segments.

③

FOR DISPLACEMENT;



$$t_{C/A} = \Delta_c + \Delta'$$

$$\Delta_c = t_{C/A} - \Delta'$$

By proportionality of hangers.

$$\frac{\Delta'}{45} = \frac{t_{B/A}}{30}$$

$$\Delta' = \frac{3}{2} t_{B/A}$$

eq (i) \rightarrow

$$\Delta_c = t_{C/A} - \frac{3}{2} t_{B/A}$$

(5) (4)

or $t_{C/A}$

$$t_{C/A} = \left[-\frac{450 \times 30 \times \frac{1}{4}}{EI} \right] \times \left[15 \times \frac{1}{3} \times 3 \right] + \left[\frac{3}{4} \times 15 \right] \times \left[\frac{1}{3} \times \frac{150 \times 15}{EI} \right]$$

$$t_{C/A} = -\frac{168750}{EI} - \frac{25312.5}{EI}$$

$$t_{C/A} = -194062.5/EI$$

Now FOR $t_{B/A}$;

$$t_{B/A} = \left[-\frac{450 \times 30}{EI} \times \frac{30}{2} \right] \times \left[\frac{1}{3} \times 30 \right]$$

$$t_{B/A} = -675000/EI$$

Putting the values in eq (ii)

$$\Delta_c = \frac{-194062.5}{2} - \left(\frac{675000}{EI} \right) \times 3/2$$

(5)

$$\Delta_c = \frac{295312.5}{EI} \text{ K-ft}^3$$

" Now for slope at point B "

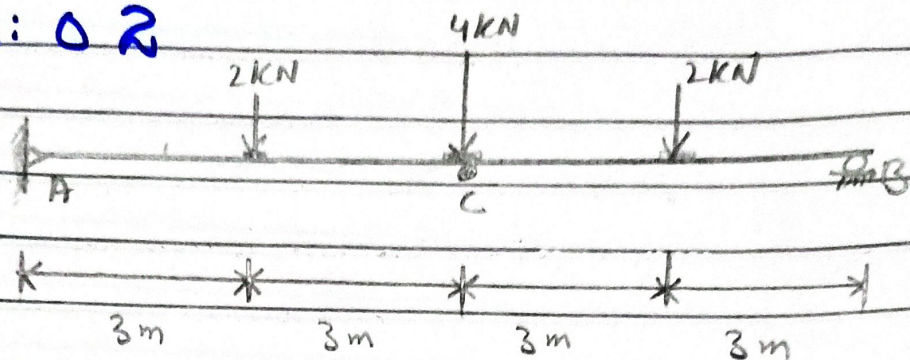
$$\begin{aligned} \theta_B &= \frac{\Delta_c}{15} \\ &= \frac{295312.5}{EI} // 15 \end{aligned}$$

$$\theta_B = \frac{19687.5}{EI} \text{ K-ft}^3$$

→ " Slope of the free end e.g. at point C is nearly equal to zero.

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Q: No: 02



GIVEN DATA:

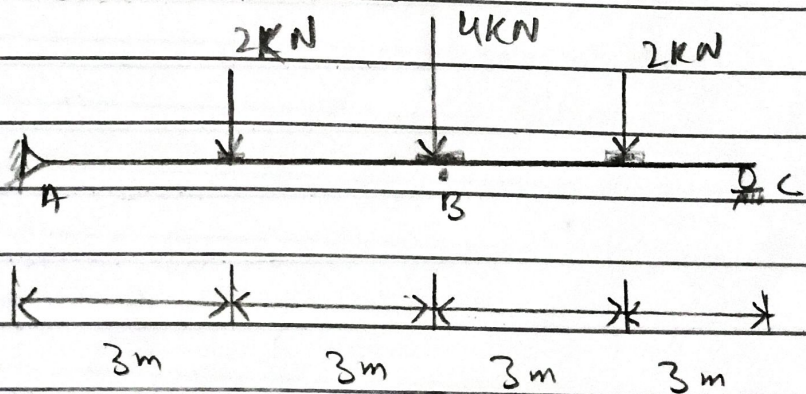
$$E = 200 \text{ GPa}$$

$$I = 8 \times 10^6 \text{ mm}^4$$

REQUIRED;

Slope at point (A) \Rightarrow
Deflection at point (D) \Rightarrow

Solution;



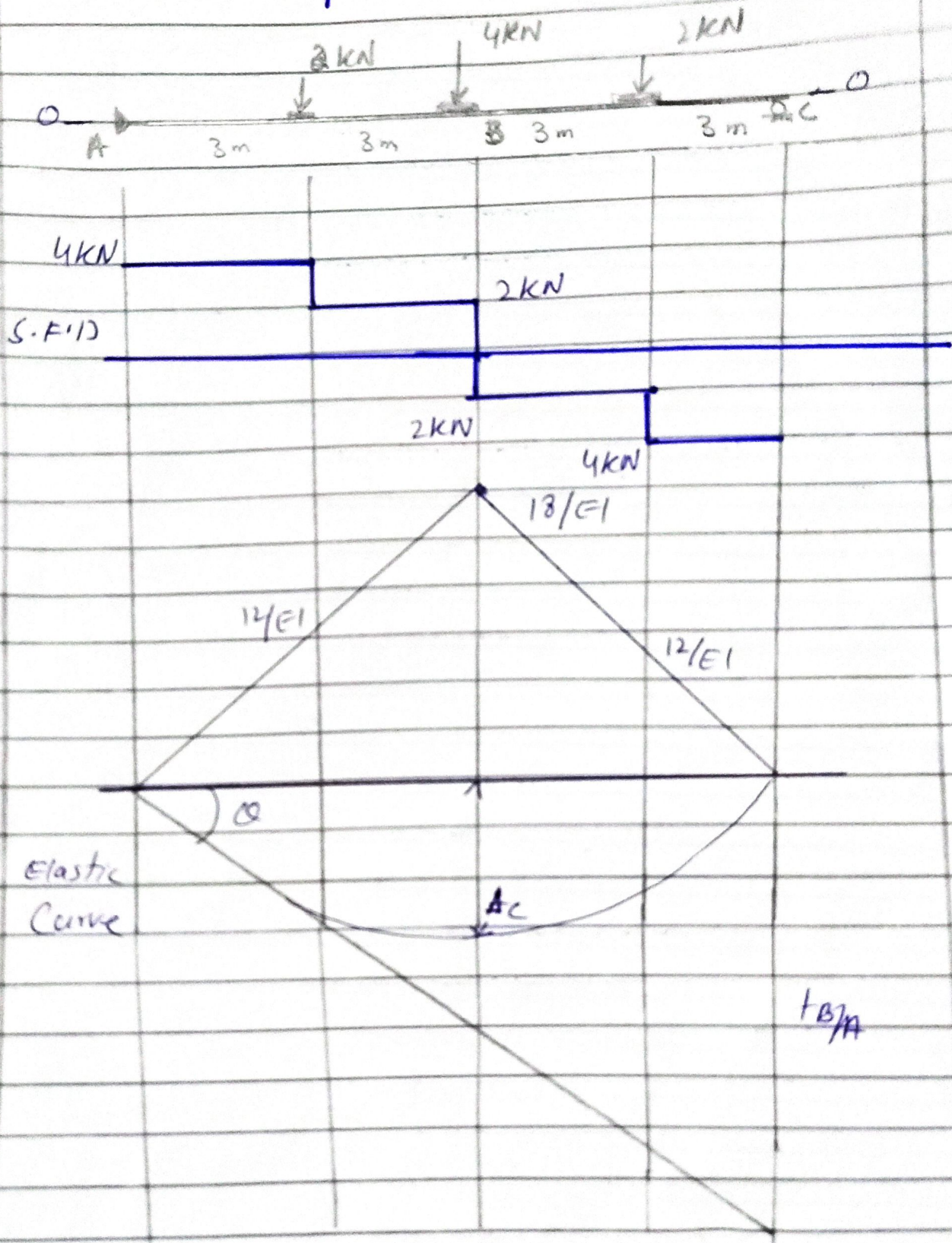
\rightarrow As the beam is symmetrical so,

$$V_A = V_B = (2 + 4 + 2) / 2$$

$$V_A = V_B = 4 \text{ kN}$$

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"FOR M/EI DIAGRAM"



$$\theta_A = \tan B/A / 120 \rightarrow \text{①}$$

(8)

$$\theta_A = \frac{\Delta_c + t_{C/A}}{6}$$

$$\Delta_c = 6\theta_A - t_{C/A} \rightarrow (ii)$$

Now for slope θ_A .

$$t_{B/A} = \int_A^B \frac{M}{EI} \bar{x}$$

$$= \frac{1}{EI} \left[\left(\frac{18 \times 6}{2} \right) \left(\frac{2}{3} \times 6 \right) + \left(\frac{18 \times 6}{2} \right) \times \left(\frac{6 \times 6}{3} \right) \right]$$

$$t_{B/A} = \frac{1}{EI} [216 + 432]$$

$$t_{B/A} = \frac{648}{EI}$$

$$\text{eq (i)} \quad \theta_A = \frac{648}{EI} \times \frac{1}{12}$$

$$\theta_A = 54 \text{ KN-m}^2$$

$$= \frac{200 \times 10^9 \times 6 \times 10^6}{(1000)^4}$$

$$\theta_A = 0.045 \text{ (rod anticlockwise)}$$



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Now for displacement at point

Δ_c ;

$$t_{C/A} = \frac{1}{EI} \left[\left(\frac{18 \times 6}{2} \right) \times \left(\frac{1}{3} \times 6 \right) \right]$$

$$t_{C/A} = \frac{108}{EI} \text{ kN-m}^3$$

eq (ii)

$$\Delta_c = \frac{6 \times 54}{EI} - \frac{108}{EI}$$

$$\Delta_c = \frac{324 - 108}{EI}$$

$$\Delta_c = \frac{216}{EI}$$

$$\Delta_c = \frac{216}{200 \times 10^9 \times 6 \times (30)^6}$$

$$\Delta_c = 0.18 \text{ m}$$

$$\Delta_c = 18 \text{ cm} \downarrow$$