

NAME :- ATIF FAROOQ

ID :- 7919

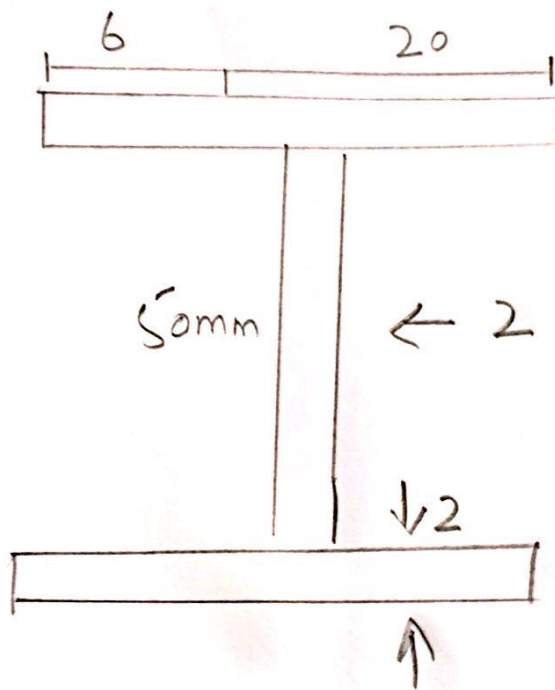
SECTION :- A

SUBJECT :- MOST II

SUBMITTED TO :- ENG SAQIB

Question : 1 (1)

(A) Determine the location of the shear centre of the beam having the cross sectional dimension
..... Centre dimensions



Required:- location of shear centre

Solution:- As we know that
$$e = \frac{t_1 h^2 b^2}{4I}$$

and

$$I_z = 2 \left(\frac{bh^3}{12} + Ay^2 \right) + \left[\frac{bh^3}{12} + Ay^2 \right]$$

$$= \left(2 \left(\frac{26(2)^3}{12} + (20 \times 12)(25)^2 \right) + \left(\frac{2(50)^3}{12} + 0 \right) \right)$$

$$I_z = 50034.66 + 20833$$

$$I = 70867.99 \text{ mm}^4$$

$$e = \frac{2(50)^2(25)^2}{4(70867.99)} = 11.02 \text{ mm}$$

So Shear Centre $e = 11.02 \text{ mm}$



(3)

Q No 1

Part (B)

Data:-

$$H = 26 \text{ ft}$$

$$D = 22 \text{ ft}$$

\Rightarrow tangential stress = ~~6000~~ ⁶⁰⁰⁰ ~~psi~~ ^{psi}

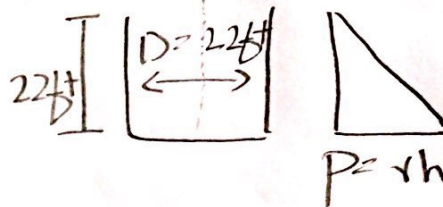
\Rightarrow Specific weight of water tank = 62.4 lb/ft^3

We have to find the thickness = ?

Solution:- The pressure

developed by water $p = \rho h$

$$6t = \frac{PD}{2t}$$

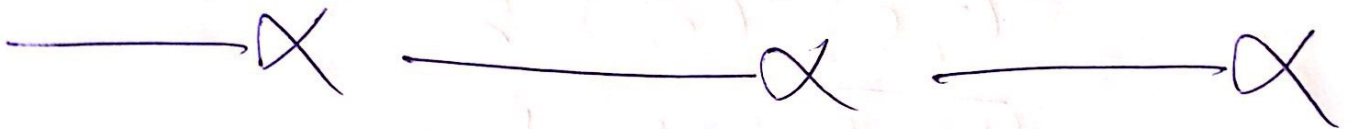


$$6t = \frac{PD}{2t} \Rightarrow \frac{\rho h D}{2t}$$

$$t = \frac{\gamma h D}{64 \times 2} \quad (4)$$

$$t = \frac{\left(\frac{62.4}{(12)^3} \right) \times (26 \times 12) \times (22 \times 12)}{6000 \times 2}$$

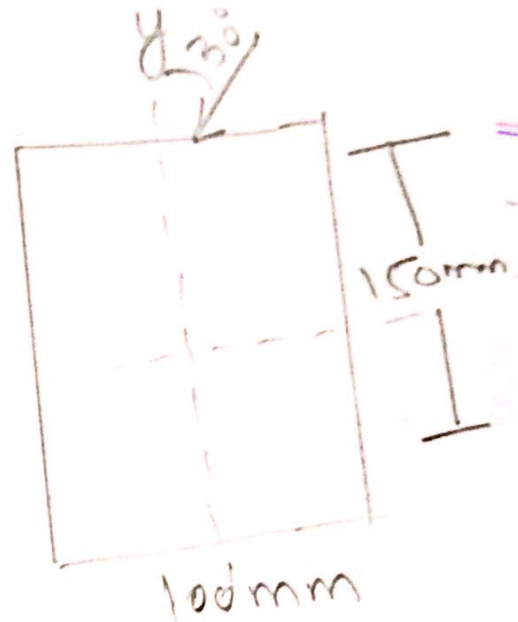
$$t = 0.24''$$



Q No 2 (5)

PART (A) :-

Given data
 $w = 4 \text{ kN/m}$
 $L = 3 \text{ m}$



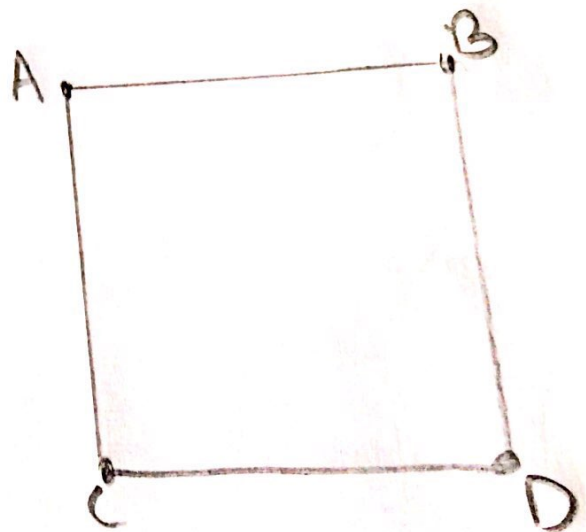
Required :-

Maximum Bending

Stress = ?

Solution :- As moment is extremes so stresses at (as shown)

The bending maximum at we would find A, B, C, & D



As we know ⁽⁶⁾

$$\bar{I} = \frac{mxy}{Ix} + \frac{myx}{Iy}$$

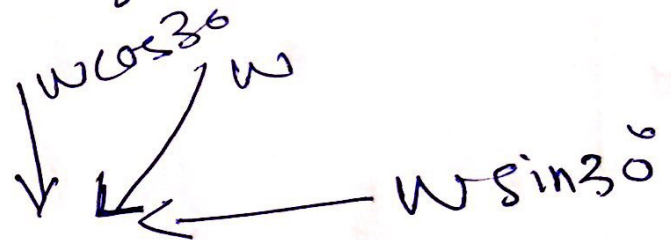
we have to find m_x & m_y

As per Question the m_x & m_y should be found at the mid.

As for simply supported we have

$$m_{mid} = \frac{wL^2}{8} \rightarrow \textcircled{1}$$

Now we have to find the exponents of w in x and y direction



$$\text{So } m_x = \frac{(w \cos 30) \times 12}{18}$$

$$\Rightarrow M_x = (7) (4 \cos 30) \times 3^2$$

$$M_x = 3.9 \text{ KN-m}$$

Now

$$M_y = \frac{(4 \sin 30) \times 3^2}{8}$$

$$M_y = 2.25 \text{ KN-m}$$

M_x is causing compression at A and B tension at C & D. M_y is causing compression at B and D tension at A and C

Now I_x and I_y

$$I_x = \frac{bh^3}{12} = \frac{0.1 \times 0.15^3}{12}$$

$$\Rightarrow 2.815 \times 10^{-5} \text{ m}^4$$

$$I_y = \frac{hb^3}{12} = \frac{0.15 \times 0.1^3}{12}$$

$$\Rightarrow 1.25 \times 10^{-5} \text{ m}^4$$

Now Stresser (8) at extreme fibers

$$\bar{\sigma}_x = \frac{Mx}{I_x} = \frac{3.9 \times 0.075}{2.815 \times 10^{-5}}$$

$$\bar{\sigma}_x = 10390.7 \text{ KN/m}^2$$

$$\bar{\sigma}_y = \frac{2.25 \times 0.05}{1.25 \times 10^{-5}}$$

$$\bar{\sigma}_y = 9000 \text{ KN/m}^2$$

Now (taking tension +ve)

$$\text{Stress at A} = \frac{Mx}{I_x} + \frac{My}{I_y}$$

$$= -10390.7 + 9000$$

$$= -1390.7 \text{ KN/m}^2 \text{ (comp)}$$

$$\text{at B} = \frac{Mx}{I_x} + \frac{My}{I_y}$$

$$= -10390.7 - 9000$$

$$\bar{\sigma} \text{ at B} = -19390.7 \text{ KN/m}^2 \text{ Comp}$$

Now

Stress

(9)

$$\text{at } c = \frac{Mx_y}{I_x} + \frac{My_x}{I_y}$$

$$= 10390.7 + 9000$$

$$= 19390.7 \text{ KN/m}^2 \text{ (Tension)}$$

Stresses

$$\text{at } D = \frac{Mx_y}{I_x} + \frac{My_x}{I_y}$$

$$= 10390.7 - 9000$$

$$= 139.7 \text{ KN/m}^2 \text{ (Tension)}$$

So the maximum stresses are at B & C

B is under compression

of 19390.7 KN/m^2 and C is under tension

of the same value

Q No 2 10

PART 8-(b)

Given data

$$L = 167t$$

$$I_x = 112.6 \text{ in}^4$$

$$I_y = 18.7 \text{ in}^4$$

$$E = 12000 \text{ psi}$$

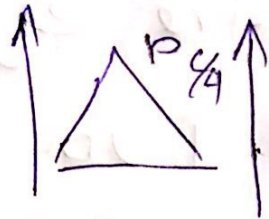
$$G = 5000 \text{ psi}$$

Solution: By looking to the figure we can judge that maximum compression would occur on A and P maximum tension at C and B there will be tension as compression which will reduced that effects of each other calculate stress A and C

$$\sigma_A = \frac{Mx}{I_x} + \frac{My}{I_y} \text{ (Comp)}$$

$$\delta = \frac{M_{xy}}{I_{xx}} + \frac{M_{yx}}{I_{yy}} \quad (\text{Centim})$$

Now M_{xx} & M_{yy}



So $M_{xx} = \frac{P \cos 60 \times (16 \times 12)}{4}$

$$M_{yy} = 48 P \sin 60$$

Now

$$0 = \frac{M_{xy}}{I_{xx}} + \frac{M_{yx}}{I_{yy}}$$

$$1200 = \frac{48 P \cos 60 \times 30.07}{112.6}$$

$$\frac{48 P \sin 60 \times 3}{18.7}$$

Solving the equation Now

$$= \frac{M_{xy}}{I_{yy}} + \frac{M_{yx}}{I_{xx}}$$

$$5000 = 48^{(12)} P \cos 60 \times 5 - 93 + \frac{48 P \sin 60 \times 0.5}{18}$$

Solving the equation

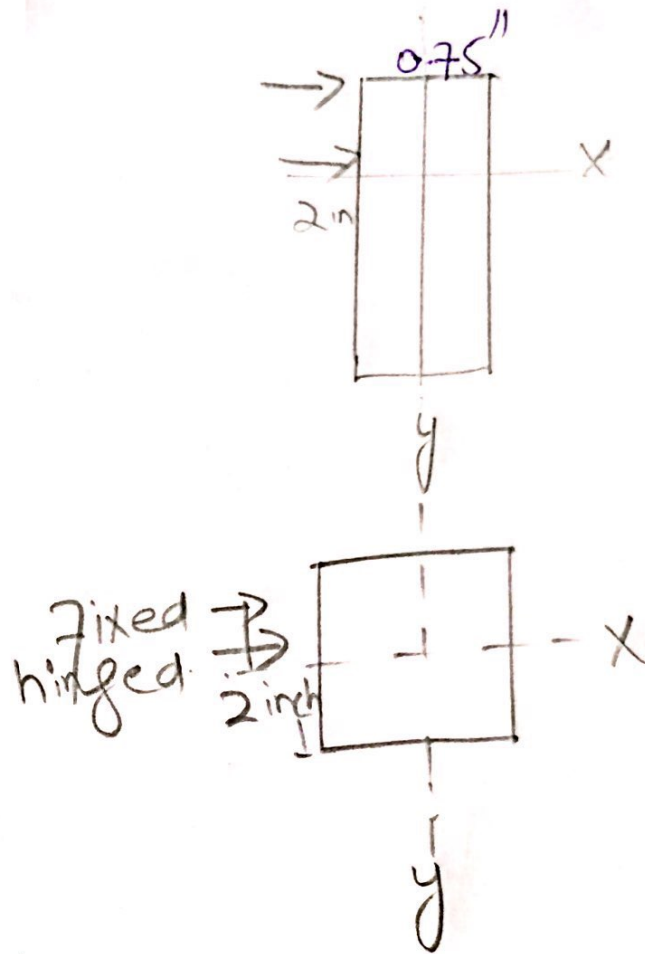
$$P = 2104.9 \text{ lb}$$

So the maximum load P applied should be 1638.6 lb

~~12~~ α α α

Q 3 :-

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Given DATA:

Length, $L = 10 \text{ ft}$
Breath, $b = 0.75 \text{ in}$
height, $h = 2 \text{ in}$

Factor of safety = 2

$E = 10.3 \times 10^6$

Required DATA:

Safe load = P_{safe} ?

Solution 0_

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CASE 1:

Structural column act as a hinged column about an axis perpendicular to the main dimension then

$$I = I_x = \left(\frac{3}{4}\right) (2)^3 = 0.5 \text{ in}^4$$
$$L_e = L \quad (\text{For hinged ended column})$$

$$P_{cr} = \frac{n^2 EI \pi^2}{L_e^2}$$

$$\Rightarrow P_{cr} = \frac{(1)^2 (10.3 \times 10^6) (0.5) (3.14)^2}{(10 \times 12)^2}$$

$$P_{safe} = \frac{P_{cr}}{\text{factor of safety}}$$

$$P_{safe} = \frac{3526.17}{2}$$

$$\Rightarrow P_{safe} = 1763.08$$

CASE II

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Column act as
fixed end about axis
parallel in 2 in ie y axis

$$I = I_y = \frac{(2)(0.75)^3}{12}$$

$$\Rightarrow I_y = 0.07 \text{ in}^4$$

Now for fixed ended

$$L_e = L/2$$

$$P_{cr} = \frac{n^2 EI \pi^2}{L_e}$$

$$\Rightarrow P_{cr} = \frac{(1)^2 (10.3 \times 10^6)(0.07)(3.14)^2}{(120/2)}$$

$$\Rightarrow \boxed{P_{cr} = 1974.65 \text{ lb}}$$

For P safe:

$$P_{\text{safe}} = \frac{P_{cr}}{\text{factor of safety}}$$

$$P_{\text{safe}} = \frac{(16)}{2} \cdot \frac{1974.65}{2}$$

$$\Rightarrow P_{\text{safe}} = 987.32 \text{ lb}$$

In both case ~~with~~ we take smaller value
 P_{safe}

$$P_{\text{safe}} = 987.32 < 1763.07$$