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o Section :-

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o Subject :-

"Hydraulic Engineering"

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o Submitted to :-

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Q :- 1 :- Given Data:-

Velocity of paraffin = 3 m/s

Pressure in model = 60 kPa

Density of paraffin = 800 kg/m<sup>3</sup>

Viscosity of paraffin = 0.002 kg/m.s

Kinematic viscosity of water =  $1.0 \times 10^{-6}$  m<sup>2</sup>/s

Q Solution:-

The pressure drop  $\Delta p$  is expected to depend upon the gate opening  $h$ , the overall depth  $d$ , velocity  $v$ , density  $\rho$ , viscosity  $\mu$ .

\* List the relevant variables.

$\Delta p, h, d, v, \rho, \mu$ .

\* Dimensions are;

$$\Delta p \rightarrow ML^{-1}T^{-2}$$

$$h \rightarrow L$$

$$d \rightarrow L$$

$$v \rightarrow LT^{-1}$$

$$\rho \rightarrow ML^{-3}$$

$$\mu \rightarrow ML^{-1}T^{-1}$$

\* number of variable,  $n = 6$

\* number of independent = 3 (M, L and T)

\* number of non-dimensional group =  $n - m = 3$

Choose  $m (= 3)$  scaling variable:

geometric ( $d$ ); kinematic/time dependant ( $v$ ),

dynamic/mass dependent ( $\rho$ )

\* form dimensionless group by non-dimensionalizing variable;  
 $\Delta p, h, \mu$ .

$$\pi_1 = \Delta p d^a v^b \rho^c$$

$$\begin{aligned} M^1 L^0 T^0 &= (M L^{-2}) (L)^a (L T^{-1})^b (M L^{-3})^c \\ &= M^{1+c} \cdot L^{-1+a+b-3c} \cdot T^{-2-b} \end{aligned}$$

$$M: 0 = 1+c \quad \Rightarrow c = -1$$

$$T: 0 = -2-b \quad \Rightarrow b = -2$$

$$L: 0 = -1+a+b-3c \quad \Rightarrow a = 1+3c-b = 0$$

$$\Rightarrow \pi_1 = \Delta p v^{-2} \rho^{-1} = \frac{\Delta p}{\rho v^2}$$

$$\pi_2 = \frac{h}{d} \quad (\text{by inspection, since } h \text{ is a length})$$

$$\pi_3 = \mu d^a v^b \rho^c \quad (\text{probably obviously by now but here goes any way.})$$

$$\begin{aligned} M^0 L^0 T^0 &= (M L^{-1} T^{-1}) (L)^a (L T^{-1})^b (M L^{-3})^c \\ &= M^{1+c} \cdot L^{-1+a+b-3c} \cdot T^{-1-b} \end{aligned}$$

$$M: 0 = 1+c \quad \Rightarrow c = -1$$

$$T: 0 = -1-b+0 \quad \Rightarrow b = -1$$

$$L: 0 = -1+a+b-3c \quad \Rightarrow a = 1+3c-b = -1$$

$$\Rightarrow \pi_3 = \mu d^{-1} v^{-1} \rho^{-1} = \frac{\mu}{\rho v d}$$

Recognition of reynold's number suggest that we replace

$$\pi_3 \text{ by } \pi'_3 = (\pi_3)^{-1} = \frac{\rho v d}{\mu}$$

hence dimensional analysis yield.

$$\pi_1 = f(\pi_2, \pi_3)$$

i.e

$$\frac{\Delta P}{\rho V^2} = f\left(\frac{h}{d}, \frac{\rho V d}{\mu}\right)$$

(a) Dynamic similarity requires that all non-dimensional groups be the same in model and prototype

$$\pi_1 = \left(\frac{\Delta P}{\rho V^2}\right)_p = \left(\frac{\Delta P}{\rho V^2}\right)_m$$

$$\pi_2 = \left(\frac{h}{d}\right) = \left(\frac{h}{d}\right)_m$$

{ automatic similar shape }  
{ geometric similarity }

$$\pi_3 = \left(\frac{\rho V d}{\mu}\right)_p = \left(\frac{\rho V d}{\mu}\right)_m$$

from the last we have a velocity ratio

$$\frac{V_p}{V_m} = \frac{(\mu/\rho)_p \cdot d_m}{(\mu/\rho)_m \cdot d_p} = \frac{0.002/800}{1 \times 10^{-6}} \times \frac{1}{5} = 0.5$$

hence

$$V_m = \frac{V_p}{0.5} = \frac{3}{0.5} = 6 \text{ m/s}$$

(b) The ratio of quantity of flow

$$\frac{Q_p}{Q_m} = \frac{(\text{Velocity} \times \text{Area})_p}{(\text{Velocity} \times \text{Area})_m} = \frac{V_p}{V_m} \left(\frac{d_p}{d_m}\right)^2$$

$$\frac{\phi_p}{\phi_m} = 0.5 \times 5^2 = 12.5$$

(c) finally for pressure drop

$$\pi_1 = \left( \frac{\Delta P}{\rho V^2} \right)_p = \left( \frac{\Delta P}{\rho V^2} \right)_m$$

$$\Rightarrow \frac{(\Delta P)_p}{(\Delta P)_m} = \frac{\rho_p}{\rho_m} \left( \frac{V_p}{V_m} \right)^2$$

$$\pi_1 = \frac{800}{1000} \times 0.5^2 = 0.2$$

$$\Delta P_p = 0.2 \Delta P_m$$

$$\Delta P_p = 0.2 \times 60$$

$$\Delta P_p = 12.0 \text{ kPa}$$



Q: 02 :- Given Data:-

H<sub>w</sub>, maximum depth of water in Reservoir = 75 m

Specific gravity of dam material,  $G = 2.567$

Allowable Compressive Strength for Dam masonry =  $754 \text{ T/m}^2$

Height of wave,  $H_{\text{wave}} = 1.355 \text{ m}$

$$\mu = 0.75$$

No uplift pressure,  $U = 0$

Solution:-

$$\begin{aligned} \text{(i) } H_{\text{limiting}} &= \frac{C_{all}}{\gamma_w (G - C_u + 1)} \\ &= \frac{754 \times 1000}{1000 (2.567 - 0 + 1)} \\ &= \frac{754000}{1000 (2.567 + 1)} \\ &= 211.38 \text{ m} > 75 \text{ m} \end{aligned}$$

So it is low gravity dam.

(ii) Top width "a"

$$\begin{aligned} \text{free board} &= 1.5 \times h_{\text{wave}} \\ &= 1.5 \times 1.355 \end{aligned}$$

$$F.B = 2.03 \text{ m}$$

$$\text{height of dam} = H + F.B = 75 + 2.03$$

$$H.D = 77.03 \text{ m}$$

$$a = 14\% \text{ of H.D}$$

$$a = 14\% \times 77.03$$

$$a = \frac{14}{100} \times 77.03$$

$$\boxed{a = 10.78 \text{ m}}$$

(3) Base width "b" with cutoff set

(i) for no sliding criteria:-

$$b = \frac{H_w}{\mu G} = \frac{75}{0.75 \times 2.567} \approx 39 \text{ m}$$

(ii) for no tension criteria:-

$$b = \frac{H_w}{\sqrt{G}} = \frac{75}{\sqrt{2.567}}$$

$$b = 46.8 \approx 47 \text{ m}$$

Use  $\boxed{b = 47 \text{ m}}$

(4) Depth of vertical portion on ups side :-

$$h' = 2a \sqrt{G - c_u}$$

$$h' = 2 \times 10.78 \sqrt{2.567 - 0}$$

$$\boxed{h' = 34.5 \text{ m}}$$

(5) Upstream off-set =  $\frac{q}{16}$

$1.0 = 7544$

$$\frac{q}{16} = \frac{10.78}{16}$$

$$\frac{q}{16} = 0.673 \text{ m}$$

(6) Depth below the water level to the end of inclined portion in ups =  $3.14 a \sqrt{C_u}$

$$= 3.14 \times 10.78 \sqrt{2.567}$$

$$= 54.3 \text{ m}$$

(7) Total width of the base of the dam

$$b = b' + \frac{q}{16} = 47 + 0.673$$

$$b = 47.673 \text{ m}$$

(8)  $\tan \theta = \frac{b'}{H}$

$$\tan \theta = \frac{b'}{H} = \frac{47}{75}$$

$$\theta = \tan^{-1} \left( \frac{47}{75} \right) = 32.07^\circ$$

(9) Depth of vertical portion on D/s (from WL on ups side)

$$\tan \theta = \frac{q}{d'} = \frac{10.78}{d'}$$

$$d' = \frac{10.78}{\tan 32.07^\circ} = 17.2 \text{ m}$$

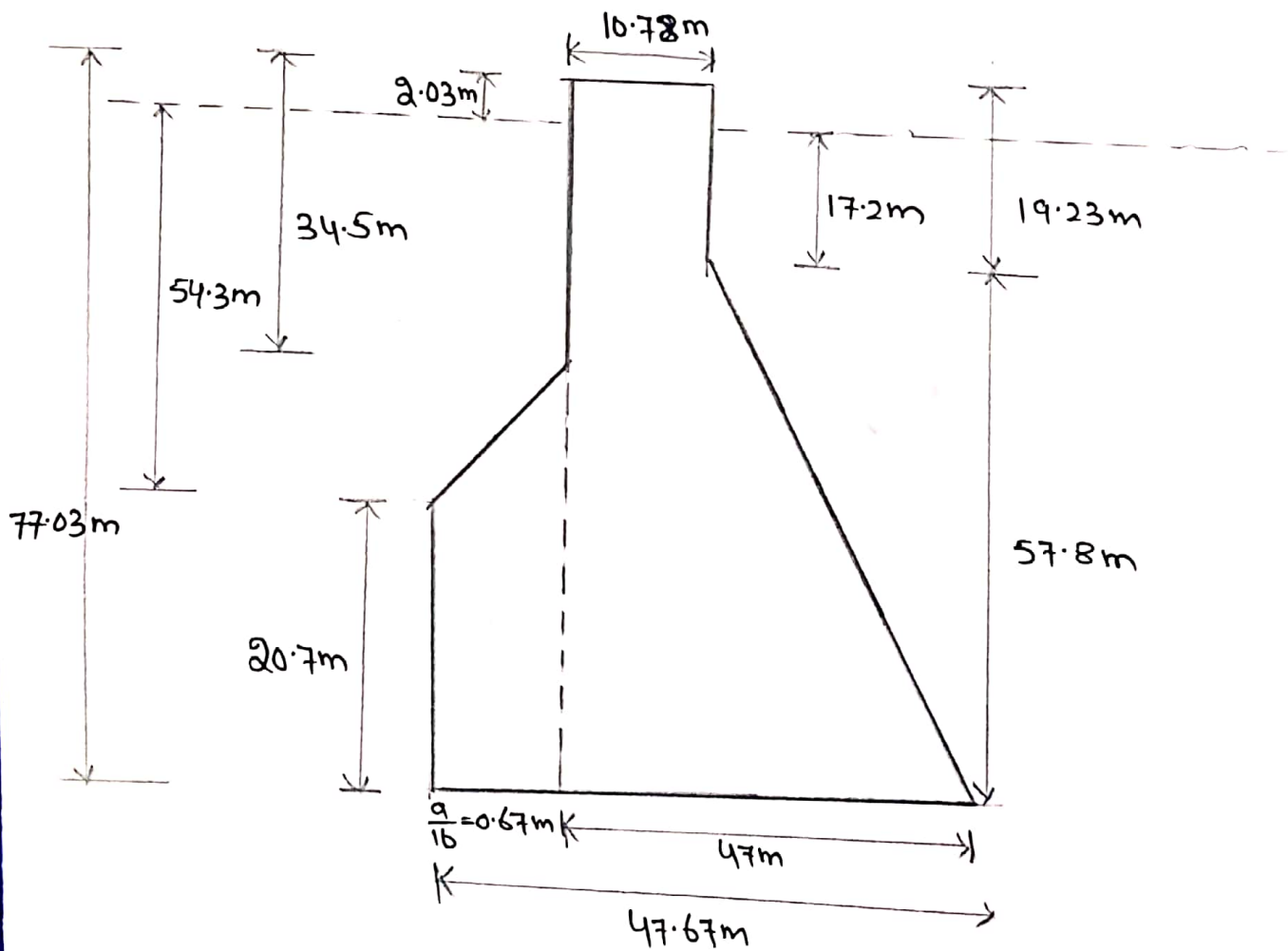


Depth of vertical portion

$$d = d' + f \cdot B$$

$$= 17.2 + 2.03$$

$$d = 19.23 \text{ m}$$



## Q 3:- Similitude and Dimension Analysis

### o Analysis to turbomachines:-

\* Pumps (Centrifugal, axial flow)

\* Turbines (impulse, reaction)

⇒ Dimensional analysis useful to make generalization about turbomechanics or distinguish between them.

Relevant variables with reference to power (P)

\* Impeller dia (D)

\* Rotational speed (N)

\* Flow (Q)

\* Energy added or subtracted (H)  $[H] = \text{Nm/kg} = \text{m}^2/\text{s}^2$

\* Fluid property such as viscosity ( $\mu$ ), density ( $\rho$ ), Elasticity.

### "Dimensional analysis for turbomechanics"

\* Assume the following relationship among variables

$$f\{P, D, N, Q, H, \mu, \rho, E\} = 0$$

### Buckingham's Theorem:-

Three fundamental dimension (M, L, T) and 8 variables imply that  $8 - 3 = 5$   $\Pi$  term can be formed.

\* select,  $\rho$ ,  $D$ , and  $N$  as variables containing the 3. fundamental dimensions to be combined with the remaining 5 variables ( $P$ ,  $Q$ ,  $H$ ,  $\mu$  and  $E$ ).

\* possible to use other variables combination that contains the fundamental dimensions

$\rho$ ,  $D$ ,  $N$  combined with  $\mu$  yields.

$$\Rightarrow \pi_1 = \mu^a \rho^b D^c N^d$$

Solving the dimensional equation gives

$$\pi_1 = \frac{\rho N D^2}{\mu} = Re$$

\* Derive other  $\pi$ -terms in the same manners

$$\Rightarrow \rho, D, N \text{ combined with } E \rightarrow \pi_2 = \frac{\rho N^2 D^2}{E} = \frac{N^2 D^2}{Q^2} = M^2$$

$$\Rightarrow \rho, D, N \text{ combined with } P \rightarrow \pi_3 = \frac{P}{\rho N^3 D^5} = C_p$$

$$\Rightarrow \rho, D, N \text{ combined with } Q \rightarrow \pi_4 = \frac{Q}{N D^3} = C_q$$

$$\Rightarrow \rho, D, N \text{ combined with } H \rightarrow \pi_5 = \frac{H}{N^2 D^2} = C_h$$

\* Summarizing the results;

$$\frac{P}{\rho N^3 D^5} = f\{C_q, C_h, Re, M^2\}$$

OR;

$$\Rightarrow \frac{Q}{ND^3} = f'' [C_p, C_H, Re, M]$$

$$\Rightarrow \frac{H}{N^2 D^2} = f'' [C_p, C_H, Re, M]$$

Previous analysis:

form a new  $\Pi$ -term

$$\Pi'_3 = \frac{P}{\rho Q H} = \frac{C_p}{C_H} f'' [C_p, C_H, Re, M]$$

= incompressible flow with  $C_p$  &  $C_H$  held constant

$$\frac{P}{\rho Q H} = f'' [Re] = \eta_H$$

$\eta_H \Rightarrow$  hydraulic efficiency.

### Alternative approach:-

Assume that relationship b/w  $P$  and  $f, Q, H$  is known, that  $\mu$  includes both  $Re$  and mechanical effects

Assume relationships :- (incompressible flow)

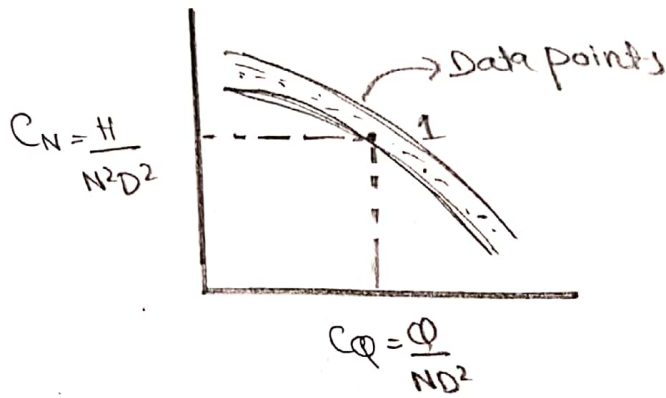
$$f\{D, N, Q, H, \mu, \gamma\} = 0$$

Alternative:

$$\frac{H}{N^2 D^2} = f' \left\{ \frac{Q}{ND^3}, \eta \right\}$$



## Typical plot of Experimental Data:-



$$H \sim D^2$$

$$Q \sim D^3$$

## Alternative Dimensionless term:-

Specific speed pumps:-

$$N_s = \frac{N \sqrt{Q}}{H^{3/4}}$$

(represent actual speed when machine operates under unit head in unit flow)

\* Common to relate  $\eta$  to  $N_s$

\* Characterize classes of pumps

\* Specific speed (turbines)

$$N_s = \frac{N \sqrt{P}}{\sqrt{P} H^{5/4}}$$



## \* Application of Dimensional analysis to pipe friction :-

Assume the wall shear stress ( $\tau_0$ ) depends on

\* mean velocity

\* Diameter (D)

\* mean height of roughness projection.

\* fluid density.

\* fluid viscosity.

⇒ Relation should hold:

$$f(\tau_0, v, d, e, \rho, \mu) = 0$$

### Buckingham's Theorem:-

⇒  $v, d, \rho$  combine with  $\tau_0$  yields

$$\pi_1 = \tau_0^a v^b d^c \rho^d$$

⇒ solving dimensional equation gives

$$\pi_1 = \frac{\tau_0}{\rho v^2}$$

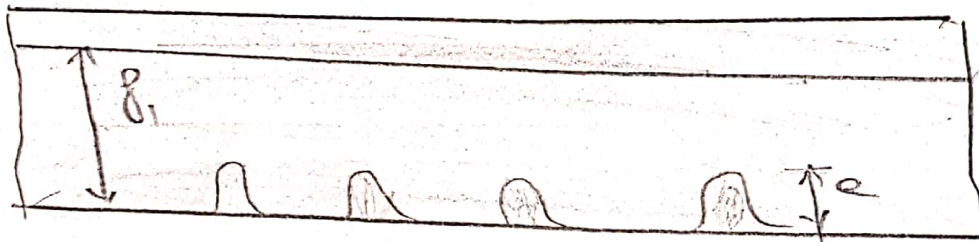
further analysis gives.

$$\pi_2 = \frac{e}{d}, \quad \pi_3 = \frac{v d \rho}{\mu}$$

⇒ following relation may derive.

$$\frac{\tau_0}{\rho v^2} = f\left(\frac{v d \rho}{\mu}, \frac{e}{d}\right) = f\left(Re, \frac{e}{d}\right)$$

$\frac{e}{d}$  = relative roughness



Hydraulically  
Smooth



Rough  
flow

Darcy - Weisbach friction formula:-

frictional losses in pipe

$$h_L = f \frac{L}{d} \frac{V^2}{2g}$$

Energy equation:-

$$h_L = \frac{f L}{\rho g R_h}$$

$$f = f'' \left[ Re, \frac{e}{d} \right]$$

# Q4:- Factor affecting fall velocity of sediments:-

When a grains falls down in stillwater it obtain a constant velocity when the upward fluid drag force on the grain is equal to the downward submerged weight of grain. This constant velocity is defined as the fall velocity.

factors affect this velocity are.

- \* Particals dia
- \* Particles density
- \* Particle concentration
- \* Shape of particle
- \* viscosity of water.
- \* Turbulency.

## (i) particles diameter:-

The particles with greater dia will settle more quickly than the particles of smaller dia. Hence the fall gravity velocity of sediments is directly proportional diameter of particles. as usually greater the mass greater will be velocity.

$$V \propto d^2$$



ii) Particles density:-

The term density is defined as mass per unit volume. When mass increase density will also increased. So sediment of higher density attains more velocity than low densed sediment. So density is directly proportional to fall velocity.

\* In some case particle density is less than dispersed medium that particles floats.

iii) Particle concentration:-

Concentration of particles will considerably effect its fall velocity as the section having greater concentration will be settled down at the place thus causing more fall velocity compare with section of low concentration.

iv) Particle Shape:-

It has significance effect on the fall velocity. Particles having regular shape tends to be effected more than irregular shape. Since regular shaped particle have even surfaces which offer very

little or low friction while particle with smaller surface area, are more lightly to be effected due to less resistance.

(5) Viscosity of water:-

The fluid shear viscosity and elasticity both seem to have significant effect on the particle settling velocity. hence in such cases increasing the fluid elasticity can help to reduce particle settling velocity even at lower shear viscosity value.

(6) Turbulency of water:-

Turbulency of water depends upon the different factor such as velocity. It will effect the fall velocity because of its zig-zag motion thus the velocity varies at every point which is why it effect the fall velocity.