

ID

7904

Name:-

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Section:-

A

Subject:-

MOS II

Teacher:-

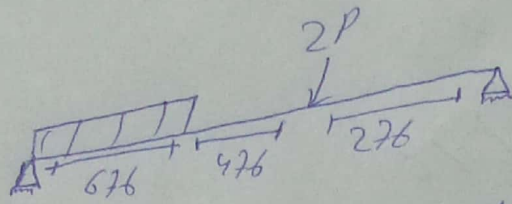
Eng. Sir Saqib

Date:

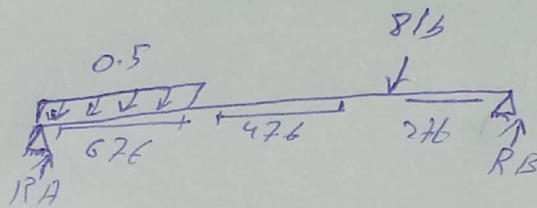
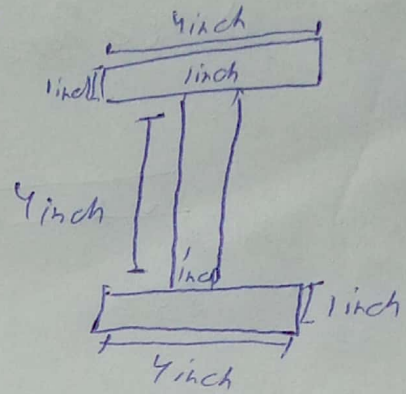
wed April 2020

QUESTION :-

Construct the Mohr's circle diagram and the Principle stress and maximum plane shear stress for the stress state of a point C located at the center of Uniformly distributed load and 1 inch below the top fiber of beam cross section shown in figure. However to construct the Mohr's circle it is necessary to draw the shear stress variation and flexural stress diagram for maximum shear force and bending moment respectively. Compare the results obtained from the Mohr's circle with the stress transformation equations.



Here my class ID number is 7904
 So $2P = 2 \times 04 = 816$



Support Reaction

$$R_A + R_B = 8.5$$

$$\sum M = 0 \quad \uparrow + \curvearrowright$$

$$R_B \times 12 - 8 \times 10 - 3 \times 3$$

$$12R_B = 80 + 9$$

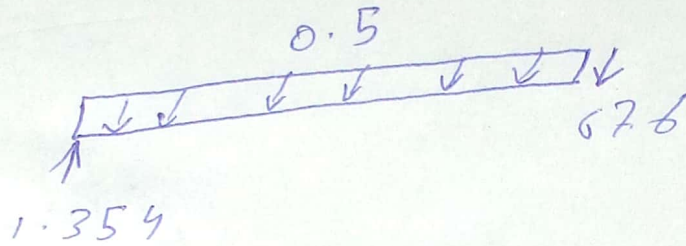
$$\frac{12R_B}{12} = \frac{89}{12}$$

$$R_B = 7.416$$

$$R_A + R_B = 8.5$$
$$R_A + 7.146 = 8.5$$
$$R_A = 8.5 - 7.146$$

$$R_A = 1.354$$

Now Shear force



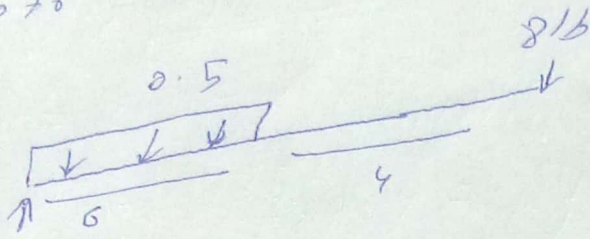
$$\sum F_y = 0 \quad \uparrow + \quad \downarrow -$$

$$-V_{6.76} + 1.354 - 0.5 \times 6 = 0$$

$$V_{6.76} = 1.354 - 3$$

$$V_{6.76} = -1.646$$

shear force at 1026
 $V_{1026} = ?$



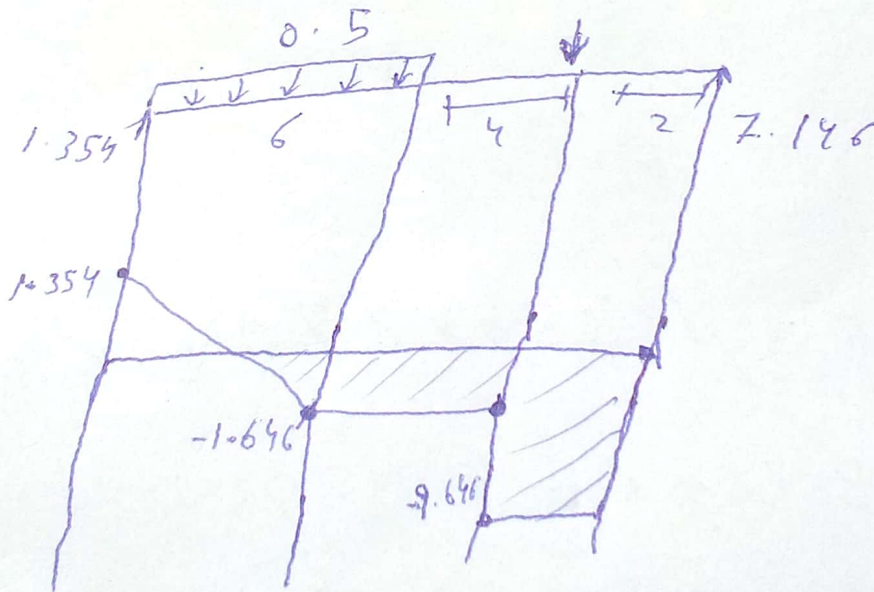
1.354

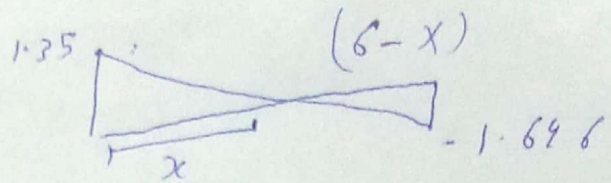
$$\sum F_y = 0 \quad \uparrow^+ \downarrow^-$$

$$\Rightarrow 1.354 - 3 - 8 - V_{1026} = 0$$

$$\Rightarrow V_{1026} = 1.354 - 3 - 8$$

$$V_{1026} = -9.646$$





$$\frac{1.35}{x} = \left(\frac{1.646}{6-x} \right)$$

$$(6-x) 1.35 = 1.646(x)$$

$$8.1 - 1.35x = 1.646x$$

$$8.1 = 1.646x + 1.35x$$

$$\frac{8.1}{2.996} = \frac{2.996x}{2.996}$$

$$x = 2.7036$$

$$E_m = 0 \Rightarrow \uparrow +$$

$$M_{2.7036} + 1.354 \times 2.7036 - 3 \left(\frac{2.7036}{2} \right) = 0$$

$$M_{2.7036} + 3.6606 - 4.0554 = 0$$

$$M_{2.7036} = 4.0554 - 3.6606$$

$$M_{2.7036} = 0.3948$$

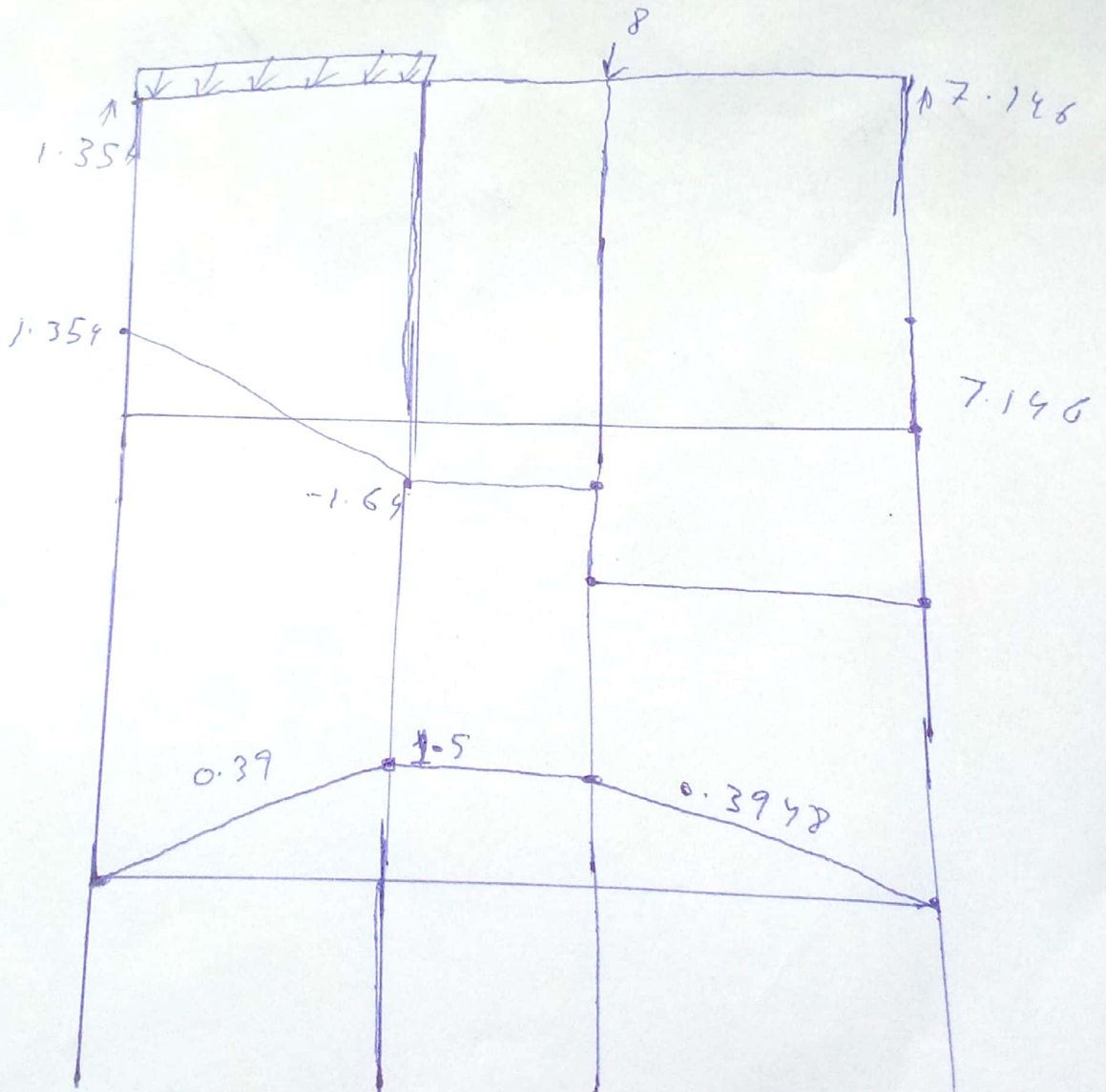
Now

$$M_{676} - 1.75 \times 6 + 0.5 \times 6 \times 3 = 0$$

$$M_{676} - 10.5 + 9 = 0$$

$$M_{676} + 10.5 - 9$$

$$M_{676} = 1.5$$



(7)

Now
shear stress

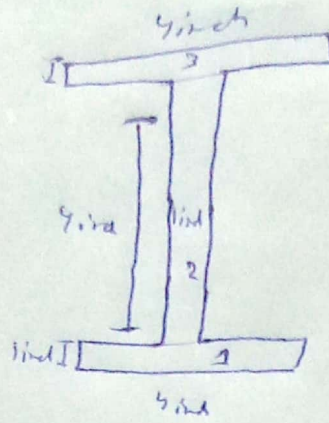
As Question the Maximum
shear stress $\tau = \frac{VQ}{It}$ occurs
where the maximum shear force
lies in above diagram.

So shear force = 7.146

To find the shear stress we
have the following formula

$$\tau = \frac{VQ}{It}$$

we first find the moment of inertia



As we know that to find centroid we have the following formula

$$\bar{y} = \frac{A_1 y_1 + A_2 y_2 + A_3 y_3}{A_1 + A_2 + A_3}$$

$$A_1 = 4 \times 1 = 4$$

$$A_2 = 4 \times 1 = 4$$

$$A_3 = 4 \times 1 = 4$$

$$\bar{y} = \frac{4 \times 0.5 + 4 \times 3 + 4 \times 5.5}{4 + 4 + 4}$$

$$\bar{y} = 3$$

Now method of inertia

$$n \quad \bar{y} \quad \sum (y_i^2) \quad \sum (y_i)$$

$$d = (\bar{y} - y_1)$$

$$(\bar{y} - y_1)(\bar{y} - y_2)$$

$$1) \quad 4 \quad \frac{4 \times (1)^2}{12} = 0.333$$

$$2) \quad 4 \quad \frac{1 \times (4)^2}{12} = 5.333$$

$$3) \quad \frac{4 \times (1)^2}{12} = 0.333$$

(Now d)

$$1) \quad d = (\bar{y} - y_1) = (3 - 0.5) = 2.5$$

$$2) \quad d = (\bar{y} - y_2) = (3 - 3) = 0$$

$$3) \quad d = (3 - 5.5) = -2.5$$

(Now d^2)

$$1) \quad 4 \times (2.5)^2 = 25$$

$$2) \quad 4 \times (0)^2 = 0$$

$$3) \quad 4 \times (-2.5)^2 = 25$$

$$I_x = I_x + Ad^2$$

(10)

$$① 0.333 + 25 = 25.333$$

$$2) 5.333 + 0 = 5.333$$

$$3) 0.333 + 25 = 25.333$$

Total

$$I = I_{x1} + I_{x2} + I_{x3}$$

$$I = 25.333 + 5.333 + 25.333$$

$$\boxed{I = 55.999 \text{ in}^4}$$

Now shear stress

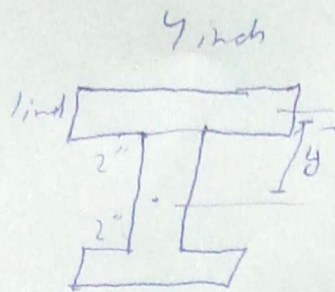
$$\tau = \frac{VQ}{Ib}$$

$$V_{\text{max}} = 7.148$$

$$Q = TA$$

b = width of that fiber

shear stress at point C located
centre of uniformly distributed
load and 1 inch below the top
fiber



$$\bar{y} = 2 + 0.5 = 2.5$$

$$A = 1 \times 4 = 4$$

$$Q = 4 \times 2.5 = 10$$

As we know that

$$t = \frac{VQ}{Ib}$$

$$t = \frac{(-7.746)(10)}{(55.996)(4)}$$

$$t = 0.3190$$

Now Flexural stress Analysis

$$\sigma = \frac{My}{I}$$

where M is maximum moment in BMD

$$M = 1.5$$

$$\sigma = \frac{(1.5)(2.5)}{55.996}$$

$$\sigma = 0.0555$$

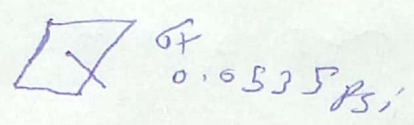
So, shear stress at point 'C' is

$$\tau = 0.475 \text{ Psi}$$

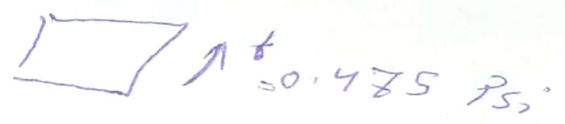
Flexural stress at point 'C'

$$\sigma = 0.0535 \text{ Psi}$$

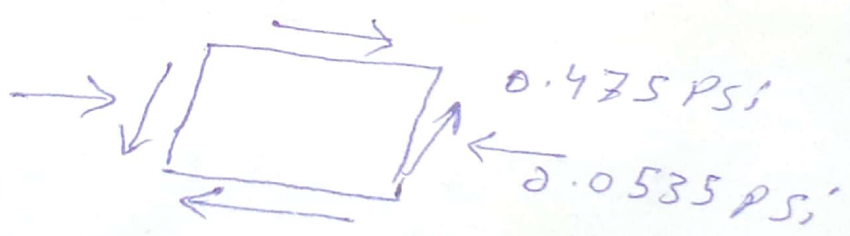
Now consider 'C' is a Planes element



0.0535 Psi is compressive Because point C lies in compression zone of Beam cross Now



Combine stress on 2D ele



N

Now we can find stress state consid of point 'c' at a degree of 20°

clockwise orientation

Solve

Given stress state

$$\sigma_x = -0.0535$$

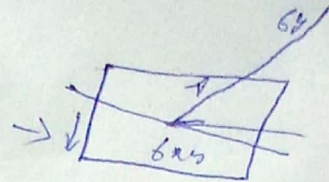
$$\sigma_y = 0$$

$$\tau_{xy} = 0.319$$

$$\sigma_{x'} = ?$$

$$\sigma_{y'} = ?$$

$$\tau_{x'y'} = ?$$



As we derive the following formula equation for stress transformation

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\sigma_{y'} = \frac{\sigma_x + \sigma_y}{2} - \frac{\sigma_x - \sigma_y}{2} \cos 2\theta - \tau_{xy} \sin 2\theta$$

for $\sigma_{x'}$

$$\sigma_{x'} = -\frac{0.0535}{2} + \frac{-0.0535}{2} \cos(2(-20))$$

$$+ (0.319) \sin(2(-20))$$

$$\sigma_{x'} = -0.02675 - 0.0204 - 0.205$$

$$\sigma_{x'} = -0.435 \text{ Psi (compressive)}$$

For $\sigma_{y'}$

$$= \frac{0.0535}{2} - \frac{(-0.0535)}{2} \cos(2(-20)) - (0.3190) \sin(2(-20))$$

$$\sigma_{y'} = -0.02675 - 0.0204 - 0.205$$

$$\sigma_{y'} = -0.435 \text{ Psi (compressive)}$$

For $\epsilon_{x'y'}$

$$\epsilon_{x'y'} = -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta + \epsilon_{xy} \cos 2\theta$$

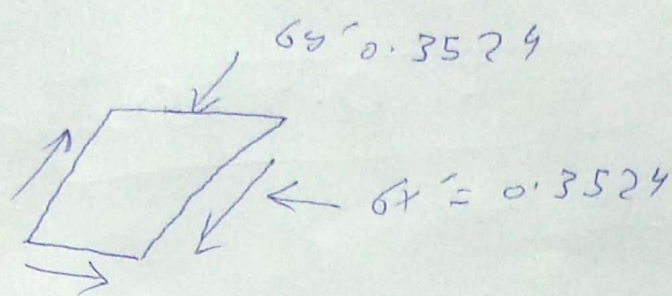
$$= -\frac{(-0.0535)}{2} - 0 \sin(2(-20))$$

$$= +0.3190 \cos(2(-20))$$

$$= -0.01719 + 0.3842$$

$$\epsilon_{x'y'} = 0.3670$$

So, the New stress state after 20° clockwise orientation is shown



Find its Principle stress

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_{1,2} = \frac{-0.0535 + 0}{2} \pm \sqrt{\left(\frac{-0.0535 - 0}{2}\right)^2 + (0.3190)^2}$$

$$\sigma_{1,2} = -0.0267 \pm \sqrt{7.155 + 0.101761}$$

$$\sigma_{1,2} = -0.0267 \pm \sqrt{7.2567}$$

$$\sigma_{1,2} = -0.0267 \pm 2.6938$$

$$\sigma_y = \sigma_1 = -0.0267 + 2.6938 = 2.6671$$

$$\sigma_x = \sigma_2 = -0.0267 - 2.6938 = -2.7205$$

Max in plane shear stress

$$\tau_{xy} = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\tau_{xy} = \sqrt{\left(\frac{-0.0535 - 0}{2}\right)^2 + (0.3190)^2}$$

$$\tau_{xy} = \sqrt{0.00071 + 0.101761}$$

$$\tau_{xy} = \sqrt{0.102471}$$

~~$\tau_{xy} = 0.3201109 \text{ Psi}$~~

$\tau_{xy} = 0.3201109 \text{ Psi}$



To Draw Mohr's Circle for the
Given Problem

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Solve

As we know that to draw the circle we need the coordinates of circle as well as radius

The coordinates of circle can find by this

$$\left(\frac{b_x + b_y}{2}, 0 \right)$$

Centre Coordinates

$$(h, k) = \left(-\frac{0.0535}{2}, 0 \right)$$

$$= (-0.026, 0)$$

Radius of Mohr's circle is

$$r = \sqrt{\left(\frac{b_x - b_y}{2} \right)^2 + Exy^2}$$

$$r = \sqrt{\left(\frac{-0.0535 - 0}{2} \right)^2 + (0.3190)^2}$$

$$r = \sqrt{0.000215 + 0.101781}$$

$$r = \sqrt{0.102476}$$

$$r = 0.3201$$

