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ID: 7925 Section : A

Subject: MOST II

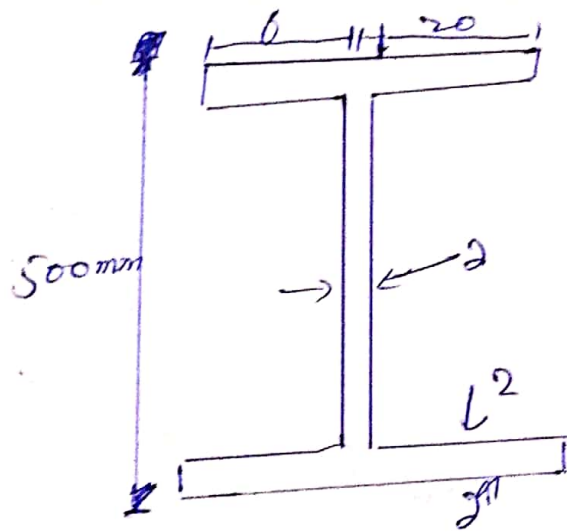
Semester: 4th Final Spring Ex paper

Degree: BE Civil

Instructor: Eng. Saad Shah

Date: 23 June 2020

Q# 1 Part A ① 1



Required location of Centroid:

Solution:

As we know that

$$e = \frac{t b h^2 b^2}{4I}$$

and:

$$I = 2 \left(\frac{b h^3}{12} + A y^2 \right) + \left[\frac{b h^3}{12} + A y^2 \right]$$

① ②

$$= 2 \left[\frac{26(2)^3}{12} + (20 \times 2)(25)^2 \right] + \left[2 \frac{(50)^3}{12} + 0 \right]$$

$$I = 50034.66 + 20833$$

$$I = 70867.99 \text{ mm}^4$$

$$e = \frac{2(50)^2(25)^2}{4(70867.99)} = 11.02 \text{ mm}$$

So shear center $e = 11.02 \text{ mm}$

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Question #1 part (b)

Data

$$\Rightarrow H = 20 \text{ ft}$$

\Rightarrow I assume the diameter

$$D = 22 \text{ ft}$$

$$\Rightarrow \text{tangential stress} = 600 \text{ lb/ft}$$

\Rightarrow specific weight of water tank

$$= 62.4 \text{ lb/ft}^3$$

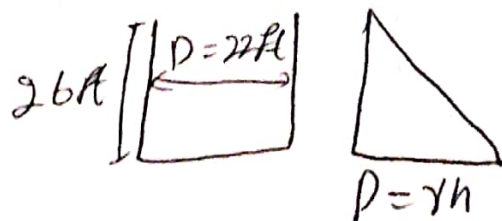
We have to find the thickness = ?

Solution:

The pressure developed by water

$$p = \gamma h$$

$$600 = \frac{pD}{2t}$$



$$bt = \frac{PD}{2t} = \frac{\gamma h D}{2t}$$

$$2t \times bt = \gamma h D$$

$$2t = \frac{\gamma h D}{bt}$$

$$t = \frac{\gamma h D}{bt + 2}$$

$$t = \frac{(62.4) \times (26 \times 12)}{(12)^3} \times 22 \times 12$$

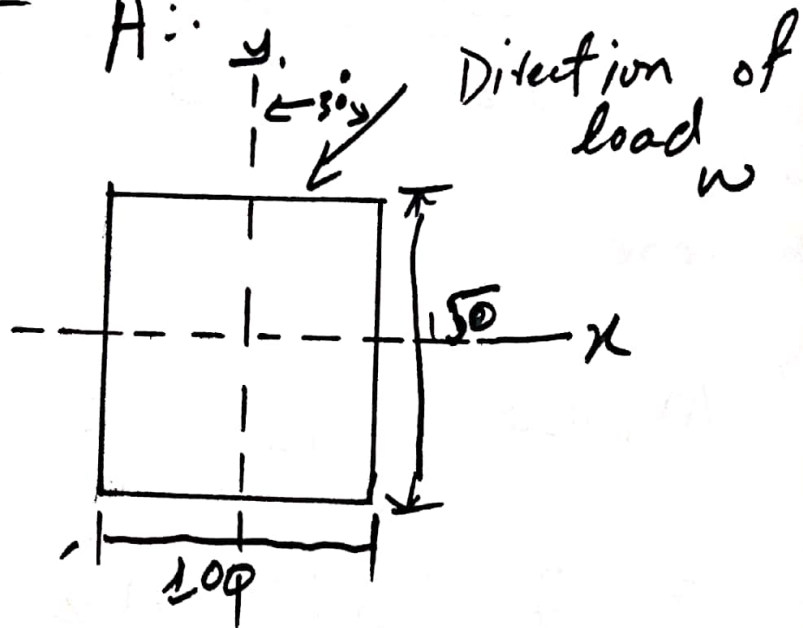
$$600 \times 2$$

$$t = 0.24''$$

⑤

Q# 2 part A:

Solution:



Moment of inertia

$$I_x = \frac{bh^3}{12} = \frac{0.1 (0.15)^3}{12} = I_x = 2.8125 \times 10^{-5}$$

Now

$$I_y = \frac{hb^3}{12} = \frac{0.15 (0.1)^3}{12}$$

$$I_y = 1.25 \times 10^{-5}$$

$$\sigma = \frac{Mz}{Iz} + \frac{My}{Iy}$$

$$\sigma = \frac{M \cos \theta}{Iz} + \frac{M \sin \theta}{Iy}$$

Where

$$M \cos \theta = P \cos \theta = Mz$$

$$= 12 \cos 30^\circ = Mz$$

$$Mz = 1.8510$$

$$M \sin \theta = P \sin \theta = My$$

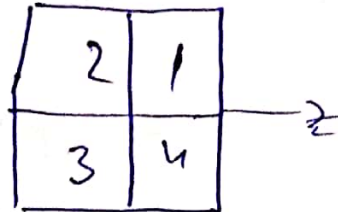
$$My = 12 \sin$$

$$My = -11.8563$$

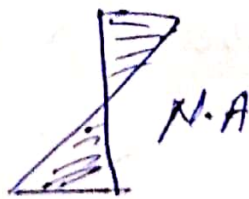
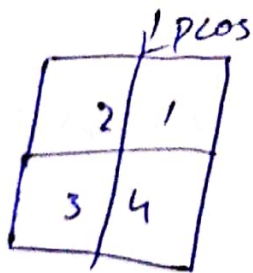
$$\sigma = \left(\frac{Mz}{Iz} \right) + \left(\frac{My}{Iy} \right)$$

$$\sigma = \frac{1.851}{2.812 \times 10^{-5}} + \left(\frac{-11.8563}{1.25 \times 10^{-5}} \right) = 882628 \text{ Nm}^2$$

Sign converting

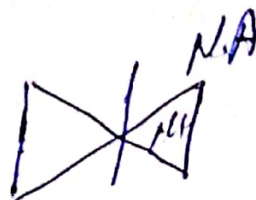
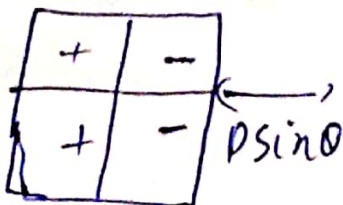


if we take compression as negative and tension as positive and the beam is a simple supported



Quadrant 1, 2 — ive

Quadrant 3, 4 + ive



Quadrant 1, 4 -ive

Quadrant 2, 3 +ive

In case of an symmetrical loading the neutral axis lies of an angle of " θ " the principle axis and the algebraic sum of stress at N.A is zero

$$\sigma = \frac{M \cos \theta}{I_z} y + \frac{M \sin \theta}{I_y} z \rightarrow \textcircled{1}$$

In this case N.A passes through 2, 4

$$\sigma = \frac{M \cos \theta}{I_z} y + \frac{M \sin \theta}{I_y} z$$

Let consider point A on N.A lies in Quadrant where

• Bending stress due to $P \cos \theta$ is compressive

• Bending ~~stress~~ ^{and stress} due to $P \sin \theta$ is tensile

$$\text{eq (i)} \Rightarrow 0 = \frac{-m \cos \theta y_A}{I_z} + \frac{m \sin \theta z_A}{I_y}$$

$$\Rightarrow 0 = \frac{-m \cos \theta}{I_z} y_A + \frac{m \sin \theta}{I_y} z_A$$

$$\Rightarrow \frac{m \cos \theta y_A}{I_z} + \frac{m \sin \theta}{I_y} z_A$$

$$\frac{y_A}{z_A} = \frac{I_z}{I_y} \frac{\sin \theta}{\cos \theta} \Rightarrow \tan \alpha = \frac{I_z}{I_y} \tan \theta$$

↘ (ii)

Now put the value of

I_z , I_y , and θ in eq (ii)

$$\text{Qw } \tan \theta = \frac{I_2}{I_1} \tan 30$$

$$\Rightarrow \tan \theta = \frac{2.8125 \times 10^{-5}}{1.25 \times 10^{-5}} (\tan 30^\circ)$$

$$\Rightarrow \tan \theta = -14.4129$$

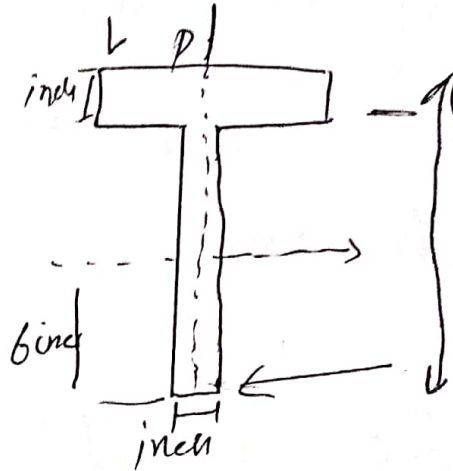
$$\theta = \tan^{-1}(-4.4129)$$

$$\theta = 1.5^\circ$$

$$\theta = 1^\circ 30' 5''$$

Question # 2

part B

Given

$$L = 16 \text{ ft}$$

$$I_x = 112 \cdot 6 \text{ inch}^4$$

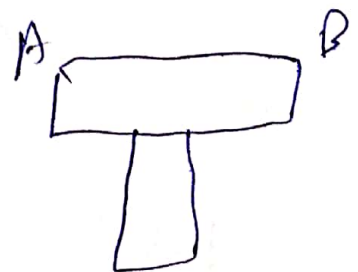
$$I_y = 18.7 \text{ inch}^4$$

$$\sigma_c = 12000 \text{ PSI}$$

$$\sigma_t = 5000 \text{ PSI}$$

Solution :

By looking figure we



we can judge that maximum compression would

will occur on a and maximum tension C at B. There

will tension as well a compression which effected effect of each other

So we will calculate stress at A and C

So

$$\sigma_A = \frac{M_x y}{I_x} + \frac{m_y x}{I_y} \text{ (Compression)}$$

$$\sigma_C = \frac{M_x \times y}{I_x} + \frac{m_y x}{I_y} \text{ (Tension)}$$

Now M_x and m_y

uclis

666

13

So

$$M_x = \frac{P \cos 60^\circ \times (16 \times 12)}{4}$$

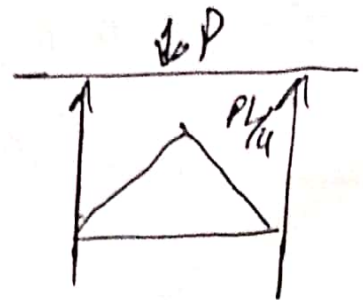
$$M_x = 48 P \cos 60^\circ$$

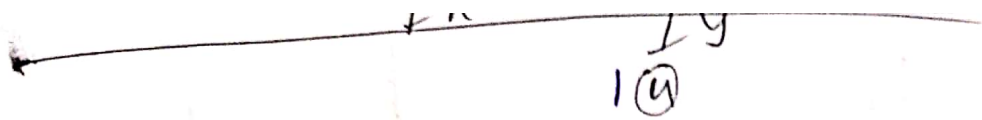
$$M_y = \frac{P \sin 60^\circ (16 \times 12)}{4}$$

$$M_y = 48 P \sin 60^\circ$$

Now

$$\delta A = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$





$$1200 = \frac{48P \cos 60^\circ \times 3.07}{112.6} + \frac{48P \sin 60^\circ \times 3}{18.7}$$

Solving the equation

$$\Rightarrow P = 1638.6 \text{ lb}$$

now

$$\delta C = \frac{mxy}{Lx} + \frac{myk}{Ly}$$

$$5300 = 48P \cos 60^\circ \times (593) + \frac{48P \sin 60^\circ \times 5}{18.7}$$

Solving the equation

$$P = 2104.916$$

So the maximum load

P applied should 1638.6 lb

Q# 03

Answer:

Given Data

Length 'L' = 10 ft

As both sides are hinged

So $L_e = L$ $E = 10.3 \times 10^6$

Factor of safety = 2

 $b = 0.75$ inch $h = 2$ inch

Required Data

17

②

- ① Determine safe load = ?
- ② safe load at fixed = ?

Solution: As we know that

$$P_{cr} = \frac{\pi^2 EI}{L_e^2}$$

As we know that $I = Ay^2$

$$I = Ay^2 \Rightarrow$$

$$y^2 = I/A$$

Now square roots
on both side

$$y = \sqrt{I/A}$$

$$y = \sqrt{\frac{hb^3}{12}} \Rightarrow \sqrt{\frac{b^2}{12}}$$

$$y = \frac{b}{2\sqrt{3}} \Rightarrow \frac{0.75}{2\sqrt{3}}$$

for hinged column, 18

$$L_e = L$$

$$I = \frac{1}{12} \times 2 (0.75)(2)^3 = 0.5 \text{ in}^4$$

$$P_{cr} = \frac{\pi^2 E I \pi^2}{L_e^2} = \frac{(1)^2 (10.3 \times 10^6) (0.5) (3.14)^2}{(10 \times 12)^2}$$

$$P_{cr} = \frac{50776940}{14400} = 3526.176 \text{ lb}$$

$$P_{\text{safe load}} = \frac{P_{cr}}{\text{factor of safety}}$$

$$= \frac{3526.176}{2} = 1763.089 \text{ lb}$$

$$P_{\text{safe load}} = 1763.089 \text{ lb}$$

$$\gamma = 0.216 \text{ inch}$$

$$P_{cr} = \frac{\pi^2 EA}{\left(\frac{Le}{\gamma}\right)^2} \Rightarrow \frac{(3.14)^2 (10-3 \times 10^6) (1.5)}{\left(\frac{10}{0.216}\right)^2}$$

$$P_{cr} = 853.8343$$

$$\text{Safe load} = \frac{\text{Crippling Load}}{\text{factor of safety}}$$

$$\Rightarrow \frac{853.834}{2}$$

$$\text{Safe Load} = 426.917$$

(*) For fixed ended column ²⁰

$$L_e = \frac{L}{2} = \frac{10}{2} \Rightarrow L_e = 5 \text{ ft}$$

$$P_{cr} = \frac{\pi^2 EA}{(L_e/r)^2} \Rightarrow \frac{(3.14)^2 (10.3 \times 10^6) (1.5)}{\left(\frac{60}{0.216}\right)^2}$$

$L_e = 5 \text{ ft}$
Converted
to inch
 $5 \times 12 = 60 \text{ inch}$

$$P_{cr} = \frac{15233082}{77160.502}$$

$$P_{cr} = 1974.207218$$

$$P_{cr} = 1974.207218$$

$$\text{Safe Load} = P_{cr}$$

Factor of Safety

$$\text{Safe load} = \frac{1974.207218}{2}$$

$$\text{Safe Load} = 987.103609$$

Required
Answer