

# Mechanics OF Solid

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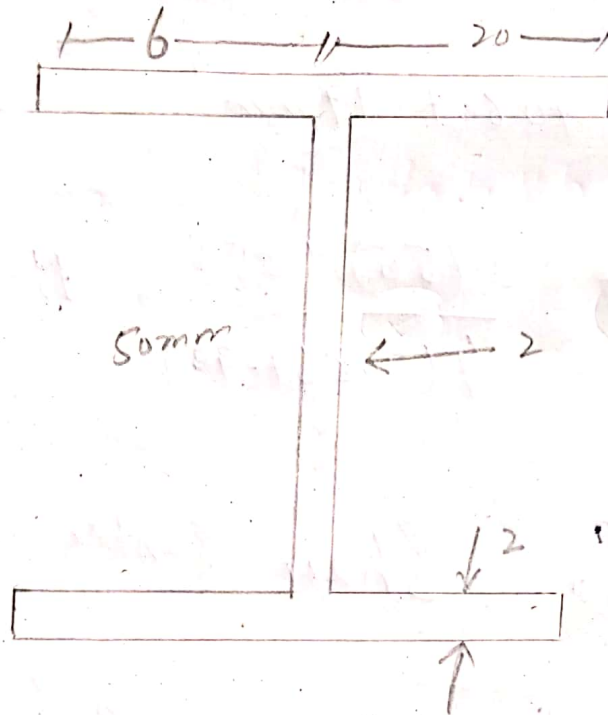
Sec : A

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# Q NO1 (Part-a)

①

Ans.



Required -

Location of Shear Center?

Solution? As we know that

$$Q = \frac{h^2 b^2}{4I}$$

$$\text{and } I = 2 \left( \frac{bh^3}{12} + Ay^2 \right) + \left( \frac{bh^3}{12} + Ay^2 \right)$$

$$\Rightarrow 2 \left( \frac{25(2)^3}{12} + (20 \times 2)(25)^2 \right) + \left( \frac{2(50)^3}{12} + 0 \right)$$

(2)

Q NO 1 (Part B).

Ans:

Given data

$$\text{Height} = 267t$$

$$\text{Tangential Stress} = 6000 \text{ psi}$$

$$\text{Specific weight of water} = 62.4 \frac{\text{lb}}{\text{ft}^3}$$

Required data

Thickness of wall of water tank =  $t = ?$

Solution:

$$P = \gamma h$$

$$\sigma t = \frac{P D}{2t} = \frac{\gamma h \times D}{2t}$$

$$t = \frac{\gamma h D}{26t}$$

Putting values

$$f = \frac{62.4 \times 26 \times D}{(12)^3}$$
$$2(6000)$$

$$f = \frac{62.4 (26 \times 12) (22 \times 12)}{(12)^3}$$
$$2(6000)$$

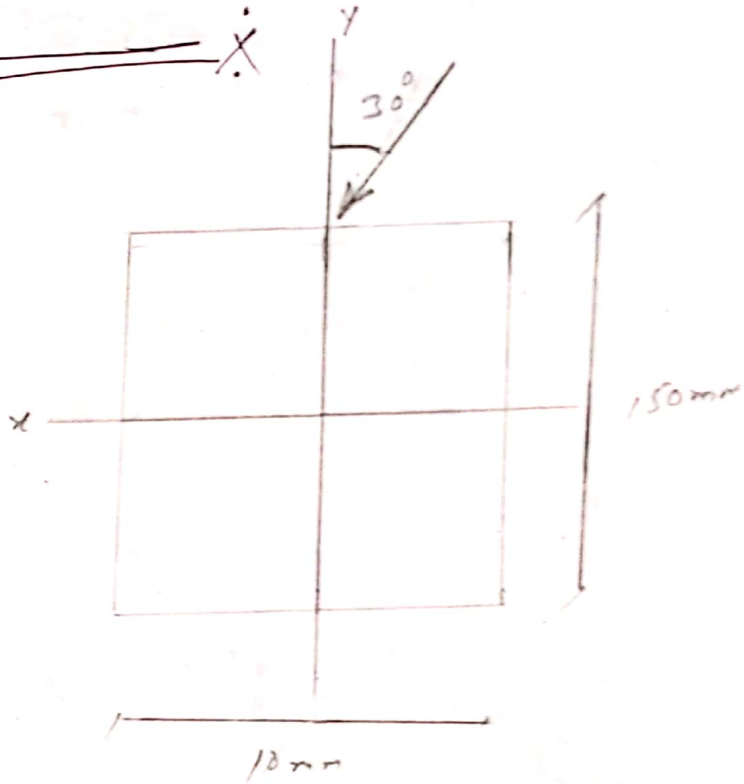
$$f = 0.24 \text{ inch}$$

Ans

④

# Q No 2 (part - a)

Ans:



Moment of Inertia:

$$I_z = \frac{bh^3}{12}$$

$$\Rightarrow 0.1 (0.15)^3 \Rightarrow 2.8125 \times 10^{-5}$$

$$I_z = 2.8125 \times 10^{-5}$$

Now  $I_y = \frac{bh^3}{12} \Rightarrow \frac{0.15 (0.1)^3}{12}$

$$I_y = 1.25 \times 10^{-5}$$

P.T.O

(5)

$$\sigma = \frac{M_z y}{I_z} + \frac{M_y z}{I_y}$$

$$\sigma = \frac{M \cos \theta}{I_z} + \frac{M \sin \theta}{I_y}$$

Where

$$M \cos \theta = M_z$$

$$= 12 \cos 30^\circ$$

$$M_z = 1.8510$$

$$M \sin \theta = M_y = 12 \sin 30^\circ$$

$$M_y = -11.8563$$

$$\sigma = \left( \frac{M_z}{I_z} \right) + \left( \frac{M_y}{I_y} \right)$$

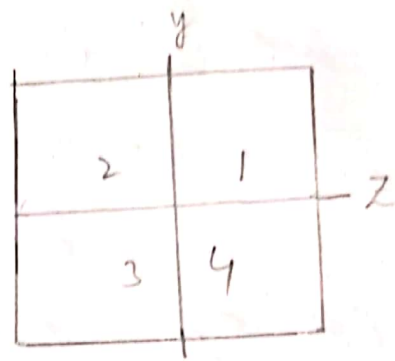
P.T.O

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$$\sigma = \frac{1.851}{2.812 \times 10^{-5}} + \left( -\frac{11.8563}{1.25 \times 10^{-5}} \right)$$

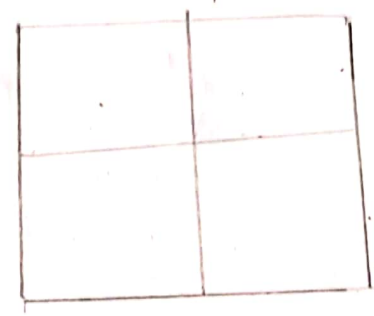
$$\sigma = 882678 \text{ N/m}^2$$

Sign Convention

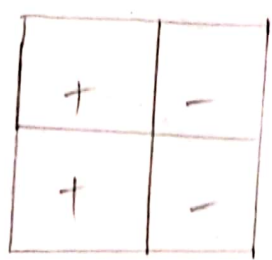


★

If we take compression as negative and tension as positive and the beam is simply supported.

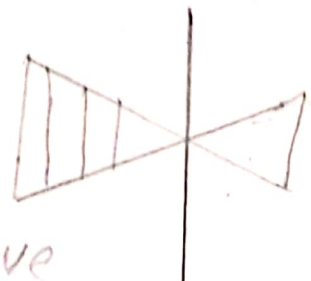


Quadrant 1, 2 -ve  
 " 3, 4 +ve



← P sin θ

Quadrant 1, 4 -ve  
 " 2, 3 +ve



In Case of unsymmetrical loading the Neutral axis lies at an angle of  $\theta$  to the principal axis and the algebraic sum of stress at N.A is zero

$$0 = \frac{M \cos \theta \cdot y}{I_z} + \frac{M \sin \theta \cdot z}{I_y} \quad \text{--- (1)}$$

In this case N.A passes through (2, 4)

$$0 = \frac{M \cos \theta \cdot y}{I_z} + \frac{M \sin \theta \cdot z}{I_y}$$

Let consider a point "A" on N.A lies in Quadrant 2, where

- \* Bending stress due to  $P \cos \theta$  is compressive
- \* Bending stress due to  $P \sin \theta$  is tensile.



eg ①

$$0 = \frac{-M \cos \theta y_A}{I_z} + \frac{M \sin \theta z_A}{I_y}$$

$$\frac{M \cos \theta y_A}{I_z} = \frac{M \sin \theta z_A}{I_y}$$

$$\frac{y_A}{z_A} = \frac{I_z \sin \theta}{I_y \cos \theta}$$

$$\tan \alpha = \frac{I_z}{I_y} \tan \theta \rightarrow \text{②}$$

Now, put values of  $I_z$ ,  $I_y$  and  $\theta$

In eg ②

$$\tan \alpha = \frac{I_z}{I_y} \tan 30^\circ$$

$$\Rightarrow \frac{2.8125 \times 10^{-5}}{1.25 \times 10^{-5}} \tan 30^\circ$$

P.T.O

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$$\tan \alpha = -14.4129$$

$$\alpha = \tan^{-1}(-14.4129)$$

$$\alpha = 1.5^\circ$$

$$\boxed{\alpha = 1^\circ 36' 5''}$$

Ans

# Q No 3 (Part-a)

Given data :

$$\text{Length} = 10 \text{ ft}$$

$$E = 10.3 \times 10^6$$

$$b = 0.75$$

$$h = 2$$

$$\text{Factor of Safety} = 2$$

Required

a) Safe load at hinged = ?

b) Safe load at fixed

Solution

a) For hinged columns

$$L_e = L$$

$$I = I_x = \frac{(0.75)(2)^3}{12} = 0.5 \text{ in}^4$$

$$P_{cr} = \frac{n^2 E I \pi^2}{L_e^2} = \frac{(1)^2 (10.3 \times 10^6) (0.5) \pi^2}{(10 \times 12)^2}$$

N.A

$$P_{cr} = \frac{50776940}{14400} = 3526.176 \text{ lb}$$

$$P_{\text{safe load}} = \frac{P_{cr}}{\text{Factor of Safety}} = \frac{3526.176}{2}$$

$$1763.088 \text{ lb}$$

(Part - b): Start with columns.

$$L_e = \frac{L}{2} \quad (\text{For fixed ended})$$

$$L_e = \frac{10}{2} = 5 \text{ ft}$$

$$I = I_y = 2 \times \frac{(0.75)^3}{12} = 0.07 \text{ in}^4$$

$$P_{cr} = \frac{n^2 E \pi^2 I}{L_e^2} = \frac{(1)^2 (10.3 \times 10^6) (0.07) (3.14)^2}{(5 \times 12)^2}$$

$$P_{cr} = \frac{7108771.6}{(60)^2}$$

$$P_{cr} = 1974658.03$$

$$P_{\text{safe load}} = \frac{1974.658}{2} = \boxed{987.3293 \text{ lb}}$$

Q No 2 Part - b



Given data

Length of beam = 16 ft

Average of inclination =  $60^\circ$

$$I_x = 102.6 \text{ in}^4$$

$$I_y = 18.7 \text{ in}^4$$

$$\sigma_t = 5000 \text{ psi}$$

$$\sigma_c = 12000 \text{ psi}$$

Required

maximum load = ?

Soln

There will be tension as well as compression

which will reduce the effect of each other.

So we will calculate  
stress at A and C. So

$$\sigma_A = \frac{M_x y}{I_x} + \frac{M_y x}{I_y} \quad (\text{in compression})$$

$$\sigma_C = \frac{M_x y}{I_x} + \frac{M_y x}{I_y} \quad (\text{tension})$$

Now  $M_x$  and  $M_y$

$$M_x = P \cos 60^\circ (16 \times 12)$$

$$M_x = 48 P \cos 60^\circ$$

$$M_y = \frac{P \sin 60^\circ (16 \times 12)}{4}$$

$$M_y = 48 P \sin 60^\circ$$

(14)

$$\sigma_A = \frac{48P \cos 60^\circ \times 3.07 + 48P \sin 60^\circ \times 3.07}{1126 + 18.7}$$

$$P = 1638.6 \text{ lb}$$

Now

$$\sigma_c = \frac{Mx}{I_x} + \frac{My}{I_y}$$

$$5000 = 48P \cos 60^\circ + 593 + \frac{48P \sin 60^\circ \times 0.5}{18.7}$$

$$P = 2104.906$$

Maximum Stress load = 1638.6 lb

Ans