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Subject: Intro to structural dynamics
and earthquake engg.

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Q# 1

Sol: ~~The~~

Beam is pulled 0.5 inch downwards

$$E = 29,000 \text{ Ksi}$$

$$I = 150 \text{ in}^4$$

st = Deflection due to static load.

$$Ib = 7168$$

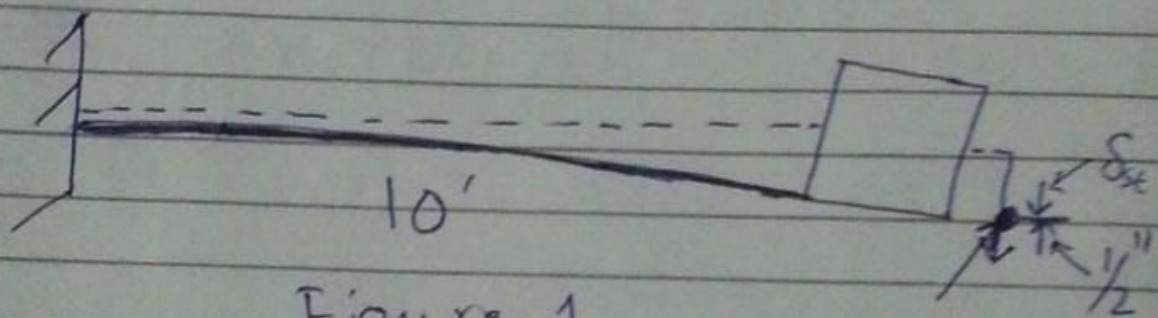


Figure-1

Sol: The general E.O.M for SDOF system is

$$Ku + c\dot{u} + m\ddot{u} = P(t)$$

In our case system is undamped ($c=0$)
undergoing free vibration ($P(t)=0$)

Hence general EOM becomes $Ku + m\ddot{u} = 0$... ①

$$K = \frac{3EI}{L^3}$$

$$= \frac{3 \times 29000 \frac{\text{K}}{\text{in}^2} \times 150 \text{ in}^4}{(16 \times 12 \text{ in})^3}$$

$$K = 7.55 \text{ k/in}$$

In order to eliminate the chances of mistake during calculation, it is more appropriate to use fundamental units like lb, ft sec or kg, m, sec

$$K = 7.55 \text{ k/in} = 90625 \text{ lb/ft}$$

$$m = \frac{7168 \text{ lbsec}^2}{32.2 \text{ ft}}$$

$$m = 222.60 \text{ slug}$$

$$\omega_n = \sqrt{K/m} = \sqrt{\frac{90625}{222.60}}$$

$$\omega_n = 20.177 \text{ rad/sec}$$

$$T_n = 2\pi/\omega_n = \frac{2\pi}{20.177}$$

$$T_n = 0.311 \text{ sec}$$

Substituting the corresponding values in eq. (1)

$$90625u + 222.60\ddot{u} = 0$$

Where "k" is in lb/ft and 'm' is in lb sec/ft²

General solution to the EOM for undamped free vibration is,

$$u(t) = u(0) \cos(\omega_n t) + \frac{\dot{u}(0)}{\omega_n} \sin(\omega_n t)$$

$$u(0) = \frac{1}{4} = \frac{1}{48} \text{ ft and } \dot{u}(0) = 0$$

$$u(t) = \left(\frac{1}{48}\right) \times \cos(20.177t) + 0 =$$

$$\left(\frac{1}{48}\right) \times \cos(20.177t)$$

Equivalent static force at any time 't' is

$$f_s(t) = k \cdot u(t) = \frac{90625 \times \cos(22.177t)}{48}$$

$$f_s(t) = 1888 \cos(22.177t)$$

Amplitude of dynamic displacement

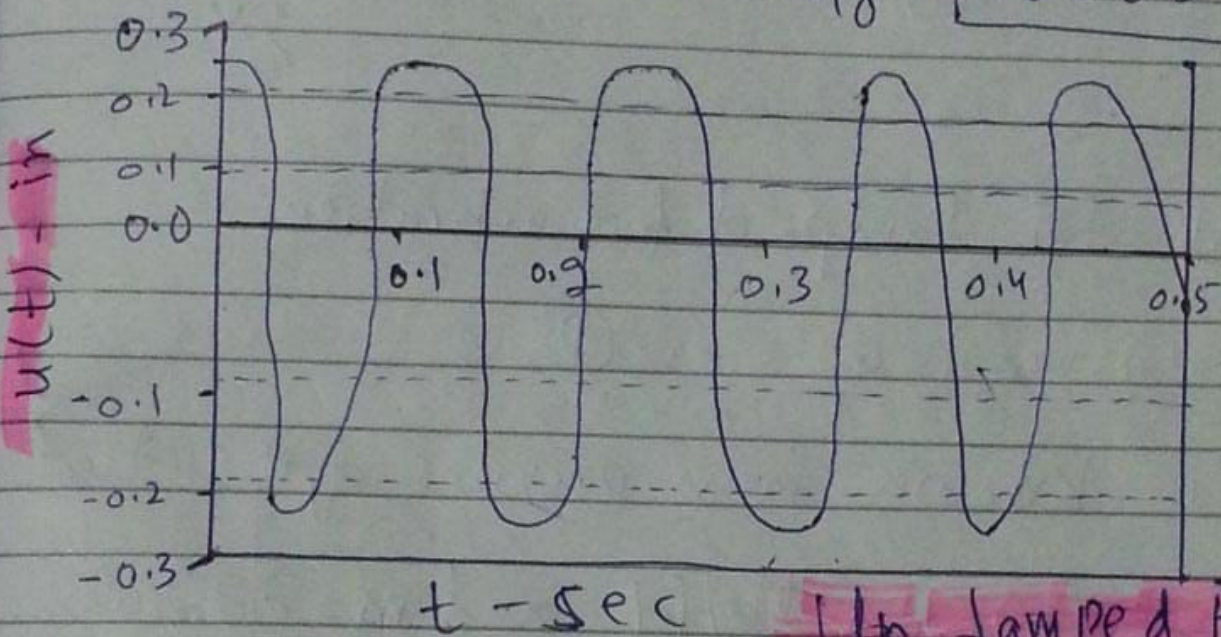
u_0 for undamped free vibration

is

$$u_0 = \sqrt{[u(0)]^2 + (\dot{u}(0)/\omega_n)^2} = \sqrt{50} \left[\left(\frac{1}{48} \right)^2 + 0 \right] = \frac{1}{48} \text{ ft}$$

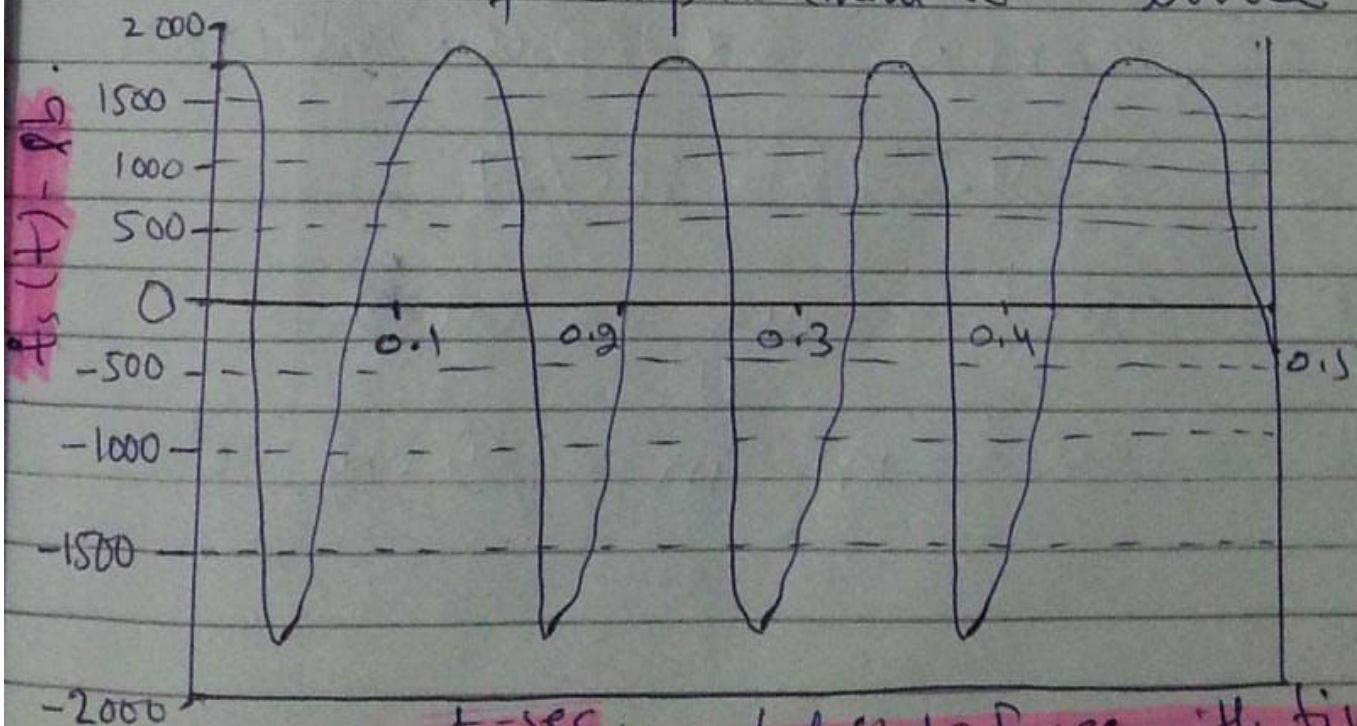
Amplitude of equivalent static force,

$$K u_0 = 90625 \times \frac{1}{48} = \boxed{1888 \text{ lb}}$$



Undamped Free Vibration

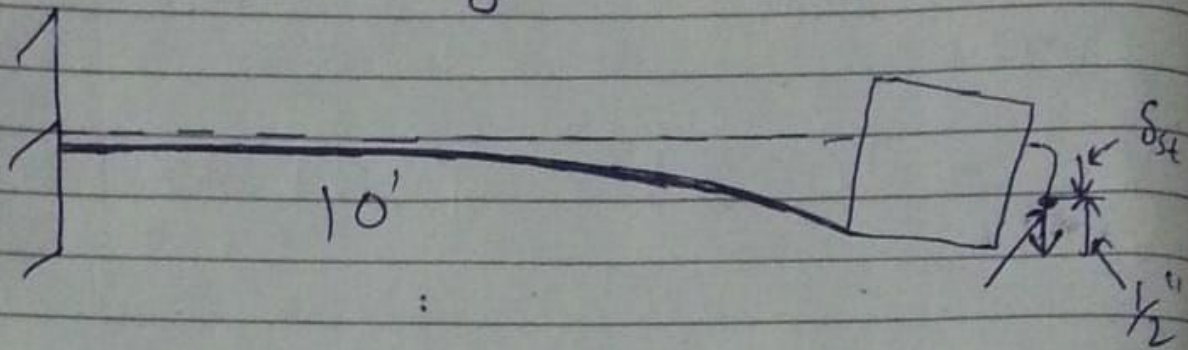
Variation of displacement with time



Variation of Equivalent Static Forces with time

Q# 2. For the beam's data given in question # 1 develop and solve the equation of motion for vibration resulting at free end.

Sol:- Beam given in q# ①



E.O.M for damped free vibration is

$$kx + c\dot{x} + m\ddot{x} = 0 \quad \text{--- ①}$$

It is known from previous q# 1 that

$$k = 90625 \text{ lb/ft} \quad \text{and} \quad m = 222.60 \text{ lb}\cdot\text{sec}^2/\text{ft}$$

$$c = \zeta \times 2m\omega_n = 2 \times 222.60 \times 20.177 \times 0.025$$

$$c = 224.57 \text{ lb}\cdot\text{sec}/\text{ft}$$

By substituting values of k , c and m in eq ① we get

$$90625u + 224.57\dot{u} + 222.60\ddot{u} = 0$$

Solution to the E.O.M for damped free vibration is:

$$u(t) = e^{-\zeta\omega_n t} \left[u(0) \cos(\omega_D t) + \frac{1}{\omega_D} [\dot{u}(0) \zeta \omega_n] \sin(\omega_D t) \right]$$

$$\omega_D = 20.177 \text{ rad/sec}$$

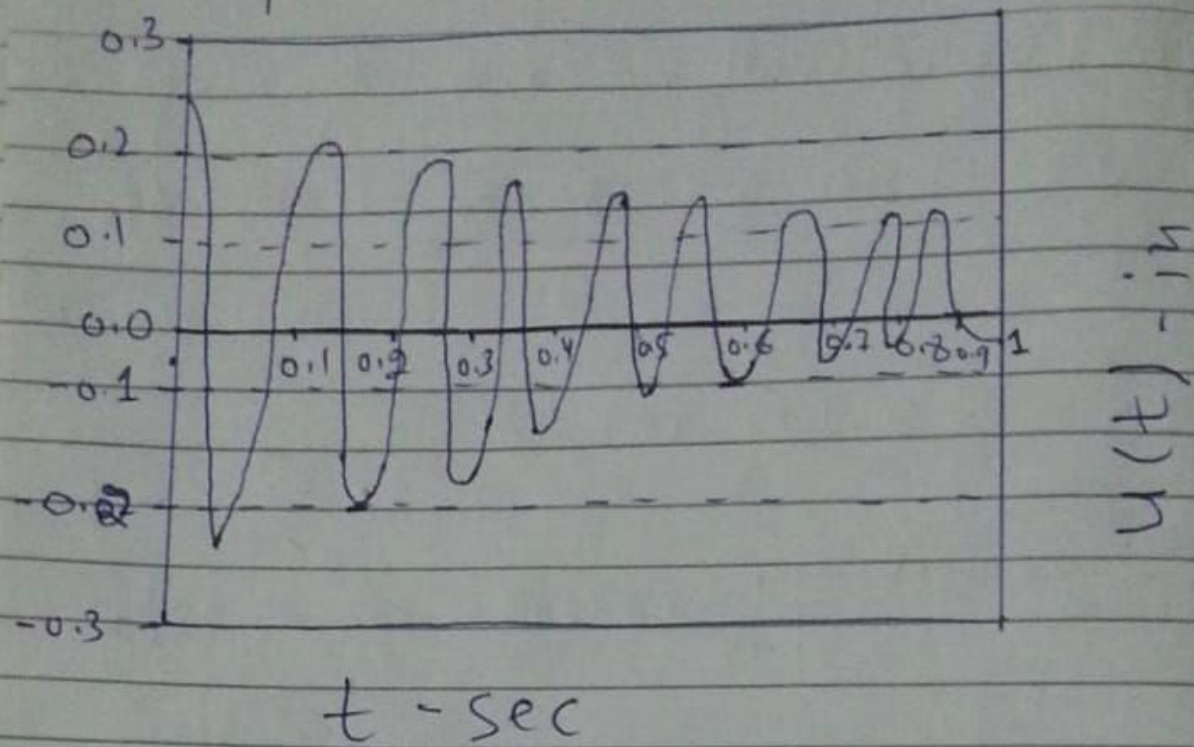
$$u(t) = e^{-0.025 \times 20.177 t} \left[\frac{1}{48} \times \cos(20.177 t) + \frac{1}{20.177} \left[0 + \frac{1}{48} \times 0.025 \times 20.177 \times \sin(20.177 t) \right] \right]$$

$$u(t) = e^{-0.504 t} \left[0.0208 \times \cos(20.177 t) + 5.2083 \times 10^{-4} \times \sin(20.177 t) \right]$$

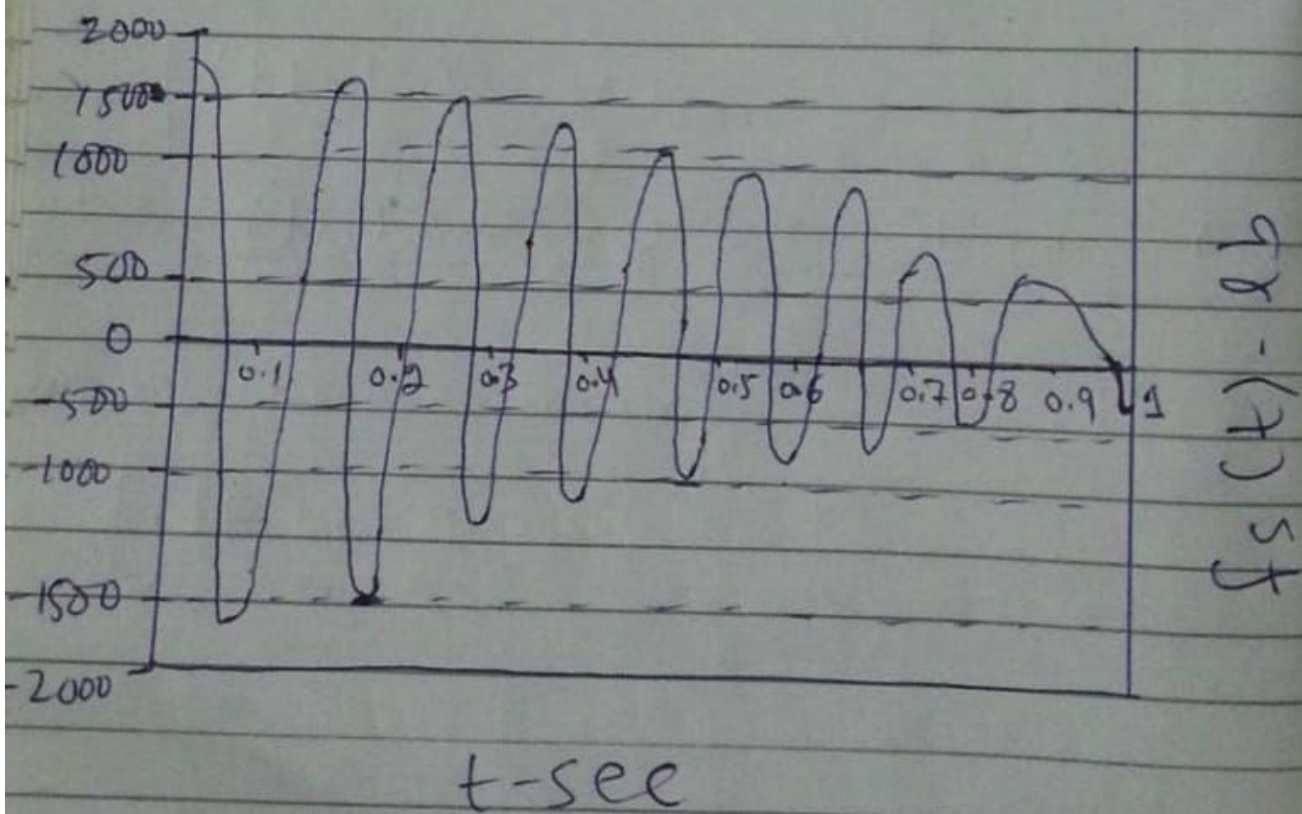
$$f_s(t) = k \cdot u(t) = 90625 \times u(t)$$

$$f_s(t) = e^{-0.504 t} \left[0.0208 \cos(20.177 t) + 0.0005208 \times \sin(20.177 t) \right]$$

Damped Free Vibration.

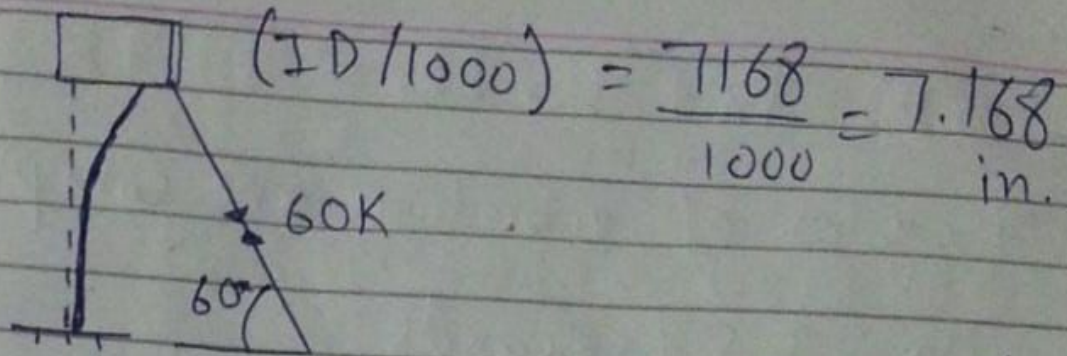


Variation of displacement with time



Variation of Equivalent Static Forces with time

Q#3.



Required Data:-

- Damping Ratios
- Natural Period of un-damped vibration
- Stiffness of structures
- Weight of tank
- Damping Co-efficient
- Number of cycles to reduce the displacement amplitude to 0.5"

Sol:-

$$u_1 = 7.168 \text{ in}$$

$$\text{After } j=7, u_{j+1} = u_6 = 0.9$$

$$a) \zeta = \text{Damping Ratio} = ?$$

$$j = \frac{1}{2\pi\zeta} \ln \left[\frac{u_1}{u_{j+1}} \right]$$

$$\Rightarrow 7 = \frac{1}{2\pi\zeta} \ln \left[\frac{7.168}{0.9} \right]$$

$$\Rightarrow \zeta = 0.181 = 1.81\%$$

$$b) T_n = ?$$

7 cycles of vibrations are completed
in 3.57 seconds

\Rightarrow Time required to complete one cycle

$$= \frac{3.57}{7}$$

$$T_D = 0.51 \text{ sec}$$

Now

$$\omega_D = \omega_n \sqrt{1 - \zeta^2}$$

$$\frac{2\pi}{\omega_D} = \frac{2\pi}{(\omega_n \sqrt{1 - \zeta^2})}$$

$$= T_D = \frac{T_n}{\sqrt{1 - \zeta^2}}$$

$$\Rightarrow T_n = T_D \times \sqrt{1 - \zeta^2}$$

$$\Rightarrow T_n = 0.51 \times \sqrt{1 - (0.181)^2}$$

$$\Rightarrow T_n = 0.5015 = 0.51 \text{ sec}$$

$$c) \quad k = ?$$

$$k = \frac{60 \times \cos 60^\circ}{2} = 15 \text{ k/in}$$

$$k = 18000 \text{ lb/ft}$$

d) Weight of tank, $W = ?$

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{k}{\left(\frac{W}{g}\right)}} = \sqrt{\frac{k \times g}{W}}$$

$$\Rightarrow \omega_n^2 = k \times g / W$$

$$W = k \times \frac{g}{\omega_n^2}$$

$$\text{Also } \omega_n = \frac{2\pi}{T_n}$$

$$\Rightarrow W = \frac{k g}{\left(\frac{4\pi^2}{T_n^2}\right)} = k g \times \frac{T_n^2}{4\pi^2}$$

$$W = \frac{18000 \text{ lb}}{\text{ft}} \times \frac{32.2 \text{ ft}}{\text{sec}^2} (0.5 \text{ sec})^2$$

$$4\pi^2$$

$$W = 74875 \text{ lb} = 74.9 \text{ K}$$

e) Damping Co-efficient, $c = ?$

It is known that $\zeta = \frac{c}{2m\omega_n}$

$$\Rightarrow c = \zeta \times 2m\omega_n = \zeta \times 2m \times (2\pi/T_n)$$

$$c = \frac{0.181 \times 4 \times \pi \times \left(\frac{74875}{32.2}\right)}{0.51}$$

$$c = 10370.4 \text{ lb}\cdot\text{sec}/\text{ft}$$

f) No. of cycles to reduce the displacement amplitude to 0.5" in

$$J = ?$$

$$J = \frac{1}{2\pi\zeta} \ln \left[\frac{u_1}{u_{j+1}} \right]$$

$$\Rightarrow J = \frac{1}{2 \times \pi \times 0.181} \ln \left[\frac{7.168}{0.5} \right]$$

$j = 12.605$ or 13 cycles