

NAME ZARAK REHMAN

SECTION "C"

ID # 7666

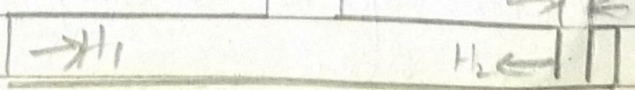
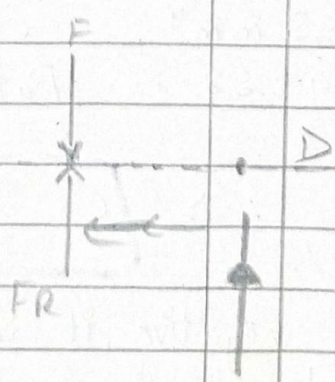
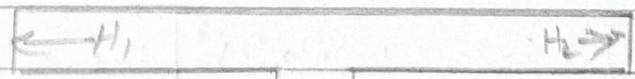
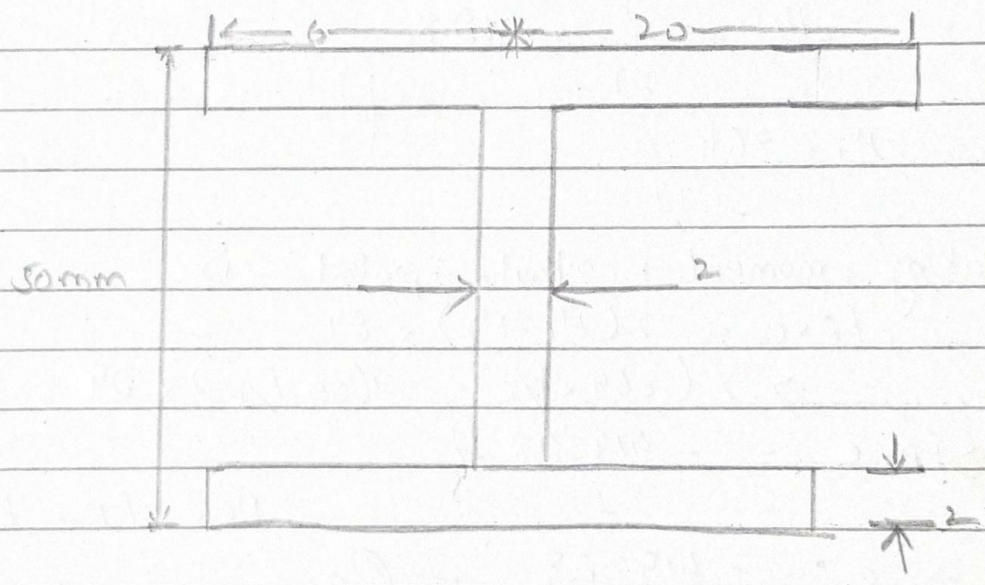
SUBJECT MECHANICS OF  
SOLID 2

SIR ENG MUHAMMAD SAQIB

Question # 1 (a)

Determine the location of the shear centre for the beam's having the cross sectional dimension's shown in the figure 1. All members are to be considered thin walled and calculation should be based on the center line dimension.

Given data :-



$$H_2 = \int q dA = \int \frac{F A \bar{y}}{I \cdot t} dA$$

$$\Rightarrow \frac{F}{I \cdot t} \int \bar{A} \bar{y} dA$$

$$\Rightarrow \frac{F}{I \cdot t} \int_0^{20} 2(20-x) \times (24 \times 2) dx$$

$$\Rightarrow \frac{F}{I \times 2} [19200]$$

$$\Rightarrow 9600 \frac{F}{I}$$

$$H_1 = \frac{F}{2I} \int_0^6 2(6-x) \times 24 \times 2 dx$$

$$H_1 = \frac{F}{2I} \times 1728$$

$$H_1 = 864 \frac{F}{I}$$

Taking moment about point D

$$FR \times e = 2(H_1 - H_2) \times 24$$

$$\Rightarrow 2(864 \times F/I - 9600 \cdot F/I) \times 24$$

$$\Rightarrow FR \times e = -\frac{419328}{I} F$$

Here  $FR = F$

$$\Rightarrow e = -\frac{419328}{I} \quad \text{--- (A)}$$

$$I = 2 \left[ \frac{26 \times 2^3}{12} + 52 \times 25^2 \right] + 2 \times \frac{46^3}{12}$$

$$I = 86561.33 \text{ mm}^4$$

$$\Rightarrow e = \frac{-419328}{I}$$

Putting value

$$e = \frac{-419328}{86561.33}$$

$$\Rightarrow e = -4.84 \text{ mm}$$

As the value is negative, it indicates that our assume direction of  $e$  is wrong and pt is 4.84 mm to the right side.

est 1(b)

Determine the thickness of the wall of a water tank constructed from steel plates joined to a height of 26 ft, the circumferential stress is limited to 6000 psi the specific weight of water is 62.4 lb/ft<sup>3</sup>

**Given data:-**

$$h = 26 \text{ ft}$$

$$\sigma_t \Rightarrow 6000 \text{ psi}$$

$$\gamma_w \Rightarrow 62.4 \text{ lb/ft}^3 \Rightarrow 0.36 \text{ lb/in}^3$$

\*  $\Rightarrow 26 \text{ ft}$  converting in to inch  
 $h = 312 \text{ inch}$

**Rev:-**

thickness =  $t = ?$

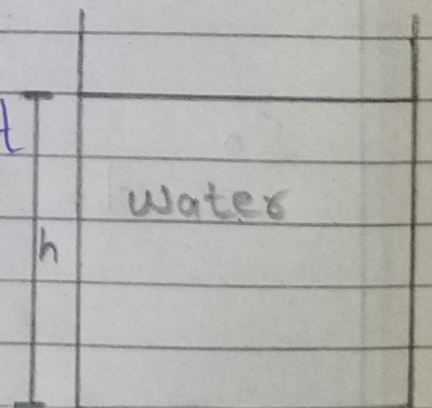
As we know that

$$P = \gamma h \quad (\text{for water})$$

$$\Rightarrow \sigma_t = \frac{P D}{2t}$$

$$\sigma_t = \frac{\gamma h D}{2t}$$

$$\Rightarrow t = \frac{\gamma h D}{2\sigma_t}$$



Putting value's

$$\Rightarrow \frac{62.4 / 12^3 \times (26 \times 12) \times D}{2 \times 6000}$$

$$\Rightarrow t = 9.38 \times 10^{-4} D \quad \text{--- (i)}$$

Since  $D$  is not given in the question, so  $t$  depends on  $D$

for different values of  $D$  we would have different values of  $t$

eg let take's

$$D = 30 \text{ ft}$$

$$\Rightarrow 30 \times 12$$

$$\Rightarrow 360 \text{ inch}$$

So

$$t = 9.38 \times 10^{-4} \times 360$$

$$= 0.337 \text{ in}$$

Also we can take any value of  $D$  and consequently calculate  $t$ .

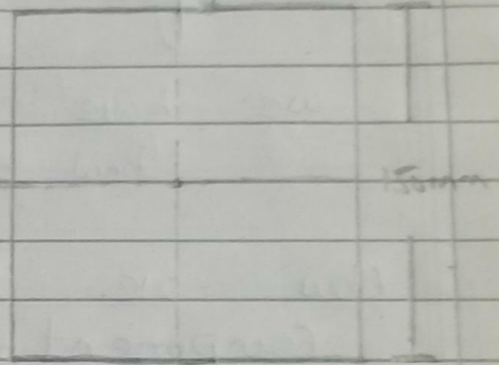
Question # 2(a)

The 100 by 150 mm wooden beam shown in figure 2 is used to support a uniformly distributed load of 4 kN on a simply span of 3m the beam.

Given data:-

$$w = 4 \text{ kN/m}$$

$$L = 3 \text{ m}$$

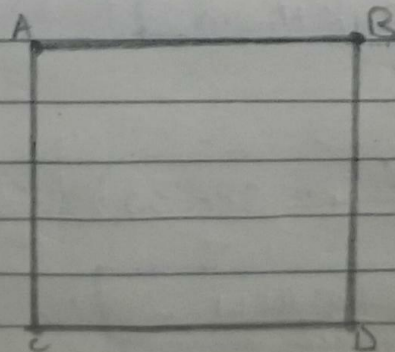
Required:-

Maximum bending stress = ?

Solution:-

As the bending moment is maximum at extremes, so we would find stress at

A, B, C, and D (as shown)



As we know

(6)

$$\sigma = \frac{M_x y}{I_x} + \frac{M_y x}{I_y}$$

we have to find  $M_x$  &  $M_y$

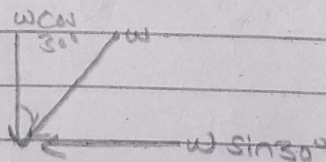
As per Question the  $M_x$  and  $M_y$  should be found at the mid

As for simply supported

we have

$$M_{mid} = \frac{wl^2}{8} \quad \text{--- (1)}$$

Now we have to find the component's of  $w$  in  $x$  and  $y$  direction's



$$\text{So } M_x = \frac{(w \cos 30) \times l^2}{8}$$

$$\Rightarrow M_x = \frac{(4 \times \cos 30) \times 3^2}{8}$$

$$M_x = 3.9 \text{ KN-m}$$

Now

$$M_y = \frac{(4 \times \sin 30) \times 3^2}{8}$$

$$M_y = 9.25 \text{ KN-m}$$

(7)

$M_x$  and  $P_c$  causing compression at A and B and tension at C and D

Now  $I_x$  and  $I_y$

$$I_x = \frac{bh^3}{12}$$

Putting value

$$\Rightarrow \frac{0.1 \times 0.15^3}{12} \Rightarrow 2.815 \times 10^{-5} \text{ m}^4$$

$$I_x = 2.815 \times 10^{-5} \text{ m}^4$$

$$I_y = \frac{hb^3}{12}$$

Putting values

$$\Rightarrow \frac{0.15 \times 0.1^3}{12} \Rightarrow 1.25 \times 10^{-5} \text{ m}^4$$

$$I_y = 1.25 \times 10^{-5} \text{ m}^4$$

Now stress at extreme fibres

$$\sigma_x = \frac{M_y}{I_x} \Rightarrow \frac{3.9 \times 0.075}{2.815 \times 10^{-5}}$$

$$\sigma_x = 10390.7 \text{ KN/m}^2$$



$$\sigma_y = \frac{2.25 \times 0.05}{1.25 \times 10^{-5}}$$

$$\boxed{\sigma_y = 9000 \text{ KN/m}^2}$$

Now (taking tension  $\uparrow$ )

$$\text{Stress at A} = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$= -10390.7 + 9000$$

$$= -1390.7 \text{ KN/m}^2 \text{ (Comp)}$$

$$\text{at B} \Rightarrow \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$\Rightarrow -10390.7 - 9000$$

$$\boxed{\sigma \text{ at B} = -19390.7 \text{ KN/m}^2 \text{ (Comp)}}$$

Now

$$\text{Stresses at C} = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$\Rightarrow 10390.7 + 9000$$

$$\Rightarrow 19390.7 \text{ KN/m}^2 \text{ (Tension)}$$

$$\text{Stresses at D} = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$\Rightarrow 10390.7 - 9000$$

$$1390.7 \text{ KN/m}^2 \text{ (Tension)}$$

$$\sigma \text{ at } \text{D} = 1390.7 \text{ KN/m}^2 \text{ (Tension)}$$

So the maximum stresses are on B and C.

B is under compression of  $19390.7 \text{ KN/m}^2$  and C is under tension of the same value.

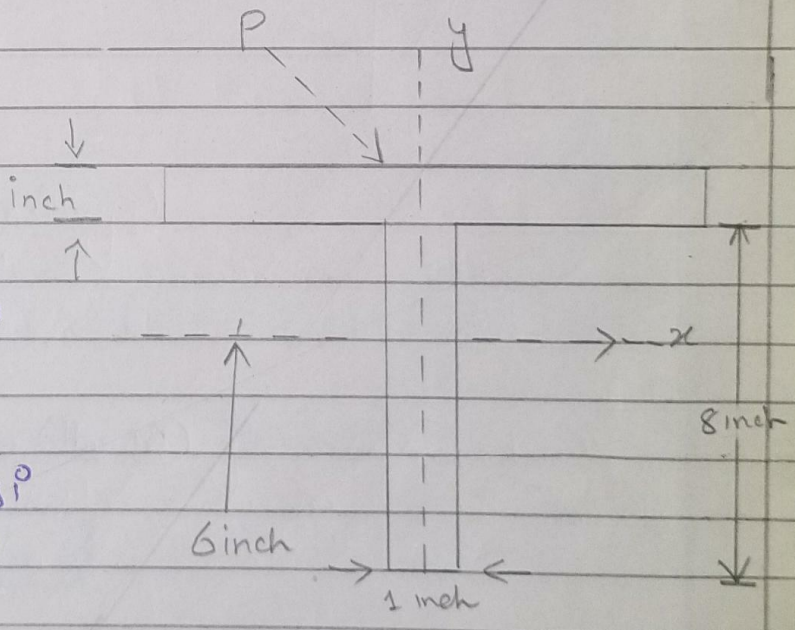
### Question 2(b):-

The T Section in figure 3 is the cross section of a simply supported beam uniformly loaded. The beam?

### Given data:-

$L = 16 \text{ ft}$   
 $I_x = 112.6 \text{ in}^4$   
 $I_y = 18.7 \text{ in}^4$

$\sigma_c = 12000 \text{ psi}$   
 $\sigma_t = 5000 \text{ psi}$



### Sol:-

By looking to the figure we can judge that maximum compression would occur on A and maximum tension at C at B there will be tension as well as compression, which will reduce the effect of each other. So we will

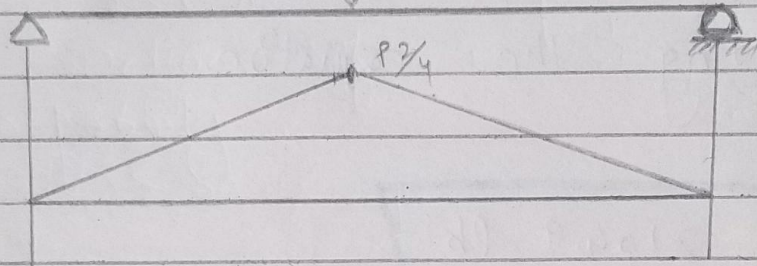
Calculate stress at A and C

So

$$\sigma_A = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y} \quad (\text{Comp})$$

$$\sigma_C = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y} \quad (\text{Tension})$$

Now  $M_x$  and  $M_y$



So

$$M_x = \frac{P \cos 60 \times (16 \times 12)}{4}$$

$$M_x = 48 P \cos 60^\circ$$

$$M_y = \frac{P \sin 60 \times (16 \times 12)}{4}$$

$$M_y = 48 P \sin 60^\circ$$

Now

$$\sigma_A = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$\Rightarrow 12000 = \frac{48 P \cos 60^\circ \times 3.07}{119.6} + \frac{48 P \sin 60^\circ \times 3}{18.7}$$

Solving the equation

$$\Rightarrow P = 1638.6 \text{ lb}$$

Now

$$\sigma_c = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$5000 = \frac{48 P \cos 60 \times (5.93)}{112.6} + \frac{48 P \sin 60 \times 0.5}{18.7}$$

Solving the equation.

$$P = 2104.9 \text{ lb}$$

So the maximum load P applied should be 1638.6 lb.

### Question 3 :-

A 10 ft long stout based on the middle has a rectangular section of 0.75 in by 2 in. A bolt through

$$E = 10.3 \times 10^6$$

### \* Given data:-

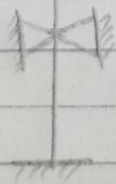
$$L = 10 \text{ ft}$$

$$E = 10.3 \times 10^6$$

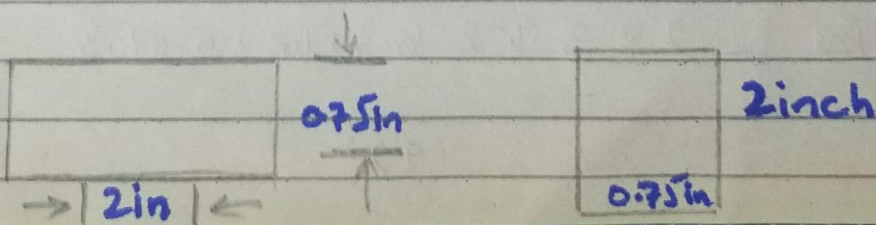
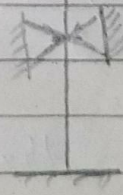
### • Sol:-

According to the given data and condition of the supports it is not clear that in which direction the column will buckle. So we will analyse both cases.

Case 1



Case 2



For Case I

$$P_{ex} = \frac{n\pi^2 EI}{L_e^2}$$

Here for Case I

$$n = 2, \quad E = 10.3 \times 10^6 \text{ psi}$$

$$I = \frac{0.75 \times 2^3}{12}$$

$$I = 0.5 \text{ in}^4$$

$$L_e = 0.5L \Rightarrow 0.5 \times 16 \times 19$$

$$\Rightarrow 96 \text{ FT}$$

$$\Rightarrow P_{ex} = \frac{2 \times 3.14^2 \times 10.3 \times 10^6 \times 0.5}{96^2}$$

$$P_{ex} = 11019.3 \text{ lbs} = 11.01 \text{ kip}$$

Now for case - 2

$$n = 1, \quad E = 10.3 \times 10^6 \text{ psi}$$

$$I = \frac{2 \times 0.75^3}{12} = 0.0703 \text{ in}^4$$

$$L_e = L = 16 \times 19 = 192$$

$$\Rightarrow P_{ex} = \frac{2 \times 3.14^2 \times 10.3 \times 10^6 \times 0.0703}{192^2}$$

$$P_{ex} = 387.8 \text{ lbs} = 0.387 \text{ Kips}$$

So

$$P_{\text{ajce load}} = \frac{0.387}{2} = 0.2 \text{ kip}$$