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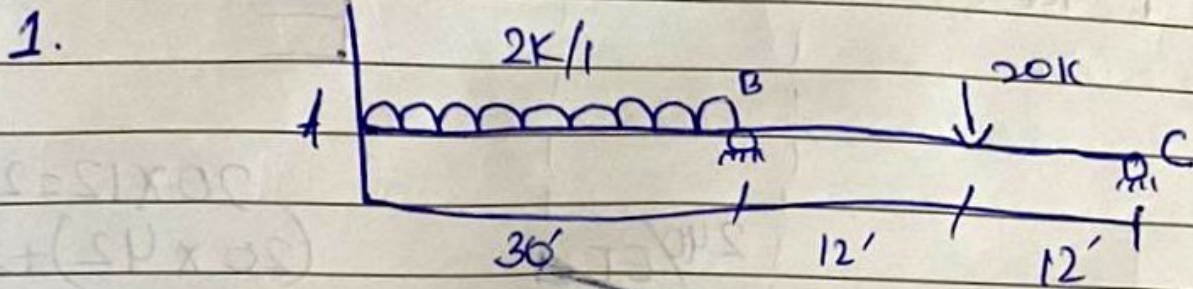
Section \Rightarrow B

Subject \Rightarrow Structural Analysis - II

Teacher \Rightarrow Engr. Adeed Khan

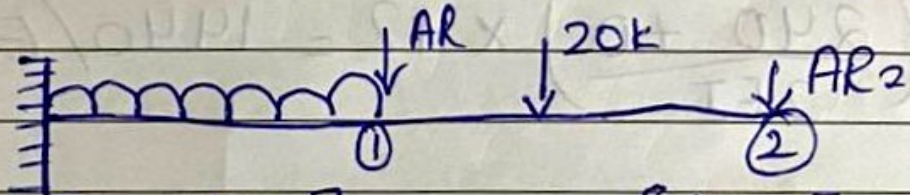
①

Q NO 1



Sol:- EI constant
 $S.I = 2'$

Step 01: Select redundant actions

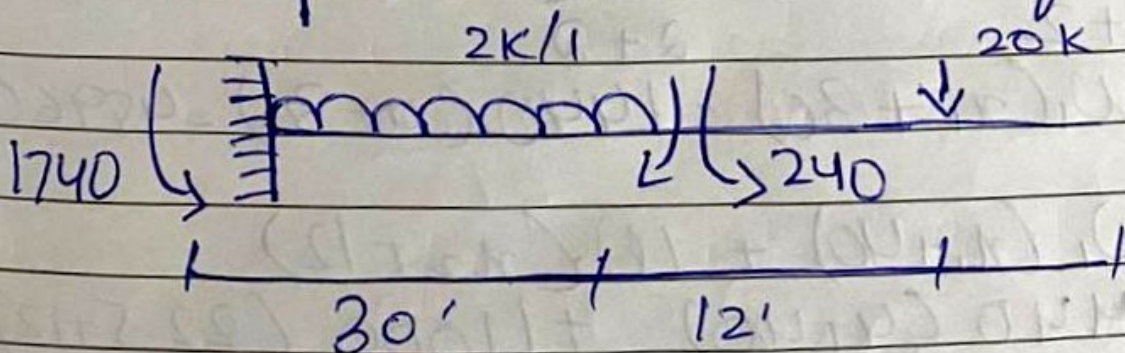


$$\begin{bmatrix} DRS_1 \\ DRS_2 \end{bmatrix} = \begin{bmatrix} ? \\ ? \end{bmatrix}, \quad \begin{bmatrix} AR_1 \\ AR_2 \end{bmatrix} = \begin{bmatrix} ? \\ ? \end{bmatrix}$$

$$[DRS] = [DRL] + F + AR$$

Step #02

Complete the values of $[DRL]$



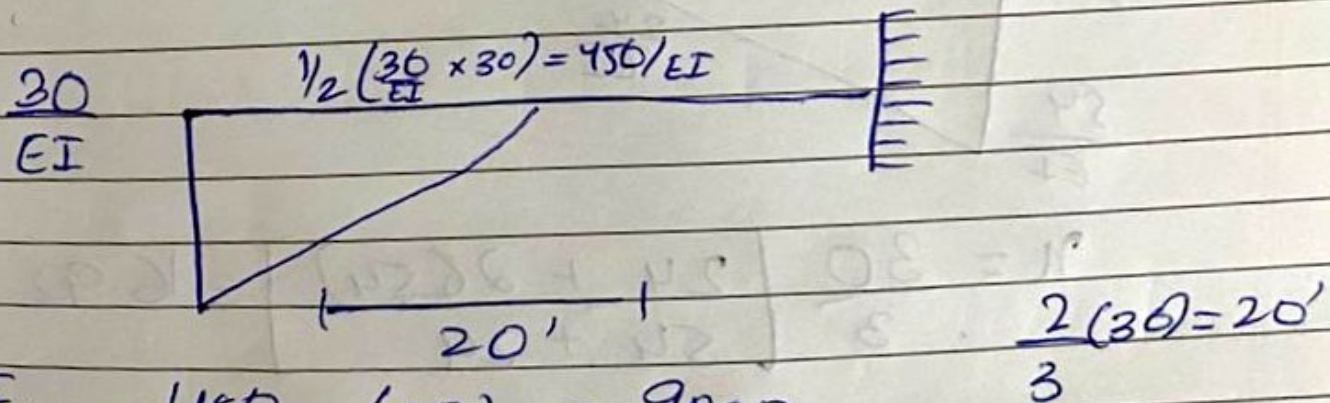
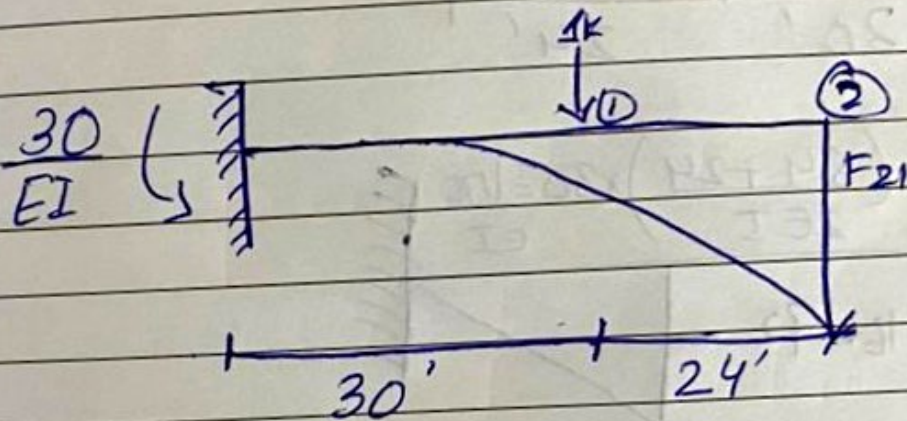
(3)

Step 03 =

Construct flexibility coefficient matrix

$$F_{2 \times 2} = \begin{bmatrix} F_{11} & F_{12} \\ F_{21} & F_{22} \end{bmatrix}$$

a) Apply a unit value of AR at reference point
i- Compute the value of F_{11} , F_{12} , F_{21} , F_{22} .

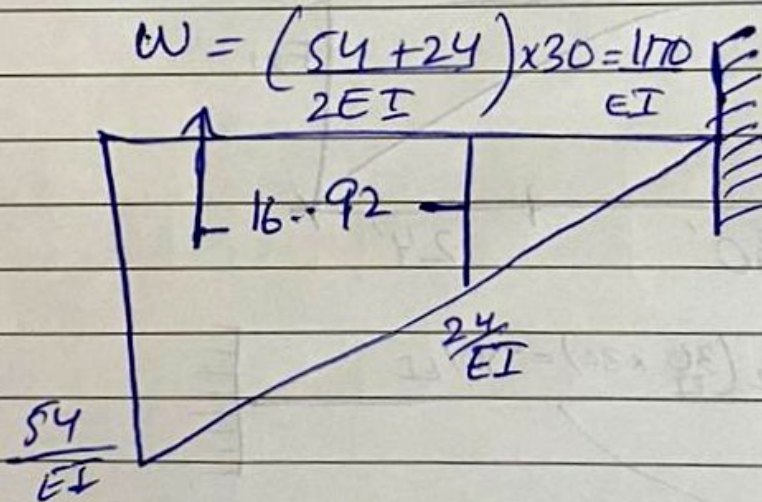
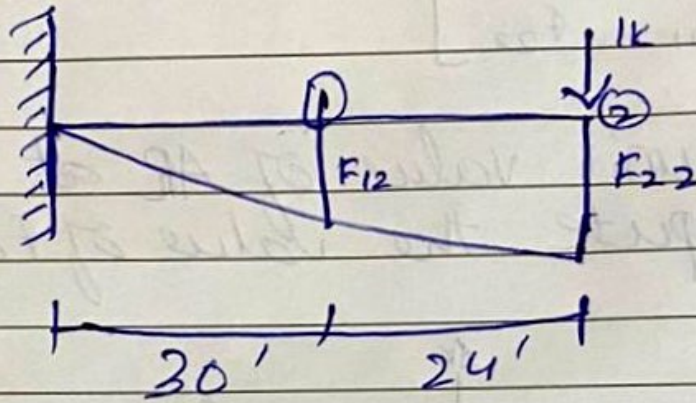


$$F_{11} = \frac{450}{EI} (20) = \frac{9000}{EI}$$

$$F_{21} = \frac{450}{EI} (20 + 24) = \frac{19800}{EI}$$

(4)

- b. Apply a unit of Ak_2 at reference point (2)
ii. Compute the value of F_{12} & F_{22} .



$$\eta = \frac{30}{3} \left[\frac{24 + 2(54)}{54 + 24} \right] = 16.92$$

$$F_{12} = \frac{1170}{EI} \times 16.92 = \frac{19800}{EI}$$

$$F_{22} = \frac{1}{2} (54 \times 54) \times \frac{1}{3} (30) + 24 = \frac{49572}{EI}$$

①

QNO#2

In force method, we assume forces and moments as unknowns and solve for them. Then we calculate displacements and rotations from forces and moments.

This is better if static indeterminacy is less than kinematic indeterminacy.

Ex: three moment equation etc.

In displacement based method, we assume displacements and rotations as unknowns and solve them. Then we calculate forces and ~~solve them~~ moments from them.

This method is better if kinematic indeterminacy is less than static indeterminacy.

Ex: slope deflection method, moment distribution method etc.

(2)

Force Method

Displacement Method

$$D_s < D_k$$

$$D_s > D_k$$

Forces are
redundant or unknowns

Displacements are
redundant or unknowns

Starts with
equilibrium of
forces

Starts with
compatible deformations

Forces found by
compatibility eqns
of displacements

displacements found
by equilibrium eqns
for forces

no. of redundants = D_s
not suitable for
computer

no. of redundants = D_k
not suitable for
trees.

(3)

Difference between force and displacement method are:-

Force Method :-

1. Assumed force as Unknown.
2. Preferable when structure has less static indeterminacy.
3. Known as Flexibility Method e.g. Consistent Method of Deformations.

Displacement Method :-

1. Assumed Displacement as Unknown.
2. Preferable when structure has less kinematic indeterminacy.
3. Known as stiffness Method. e.g. slope Displacement Method and Moment Distribution Method.

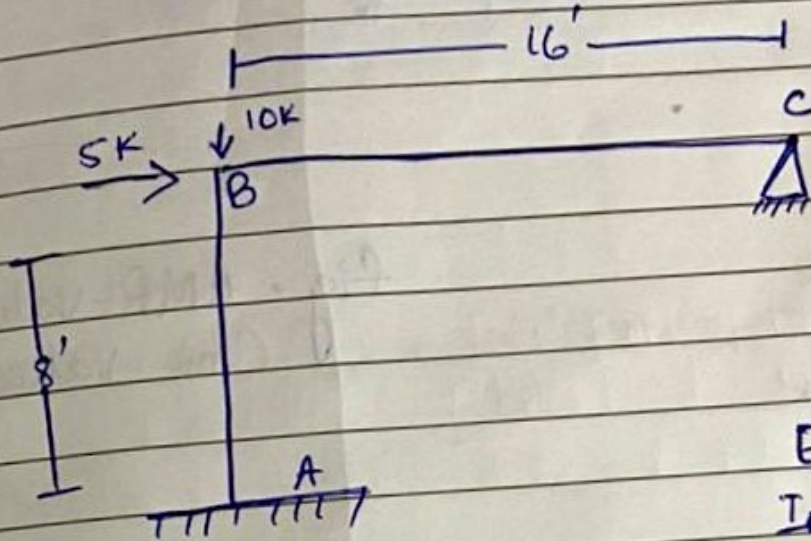
In Case of Both ended fixed support, static Indeterminacy = $6 - 3 = 3$, while Kinematic Indeterminacy = 0. So, better to start the calculation from moment Distribution Method.

(4)

Globally Displacement based
analysis of structure is
easy and used more.

QNO#3

Pb # 01:



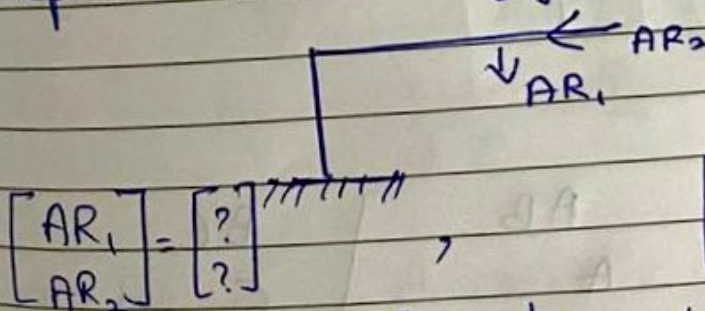
$E = \text{constant}$
 $I_c = I$
 $I_B = 2I$

Sol:-

Total statical indeterminacy

$\Rightarrow R - 3 = 5 - 3 = 2^{\circ}$

step # 01: Identify Redundant actions



$[DRS_1] = [0]$
 $[DRS_2] = [0]$

step # 02: Compute

value of $[R_L]$

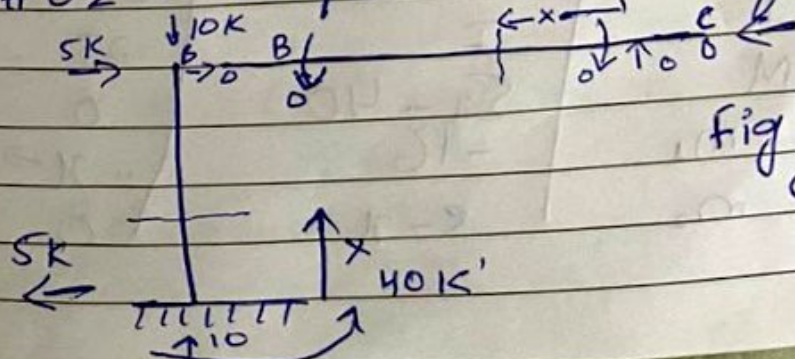


Fig: AML values (M-values).

(2)

Step # 03: [F] or [AMR]

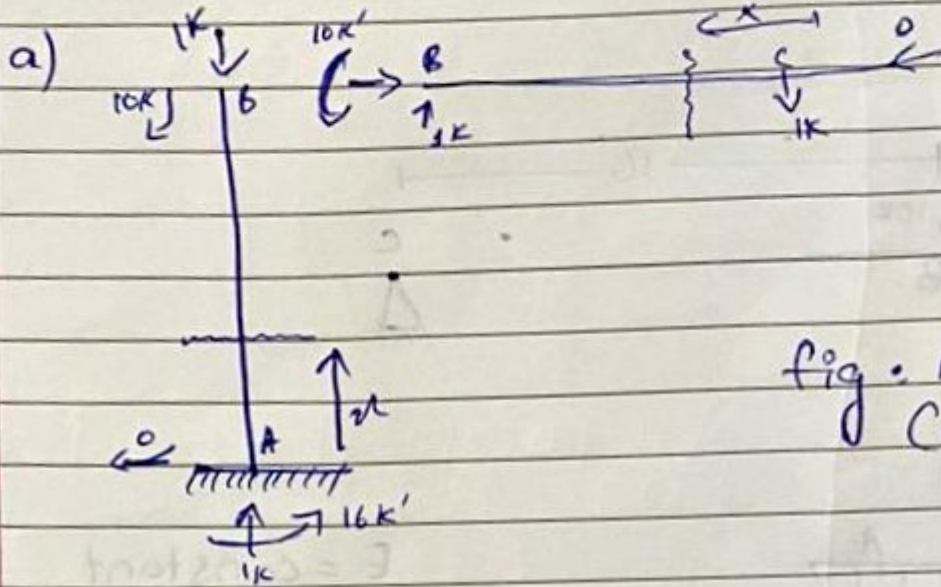


Fig: AMR-values
(m_1 values)

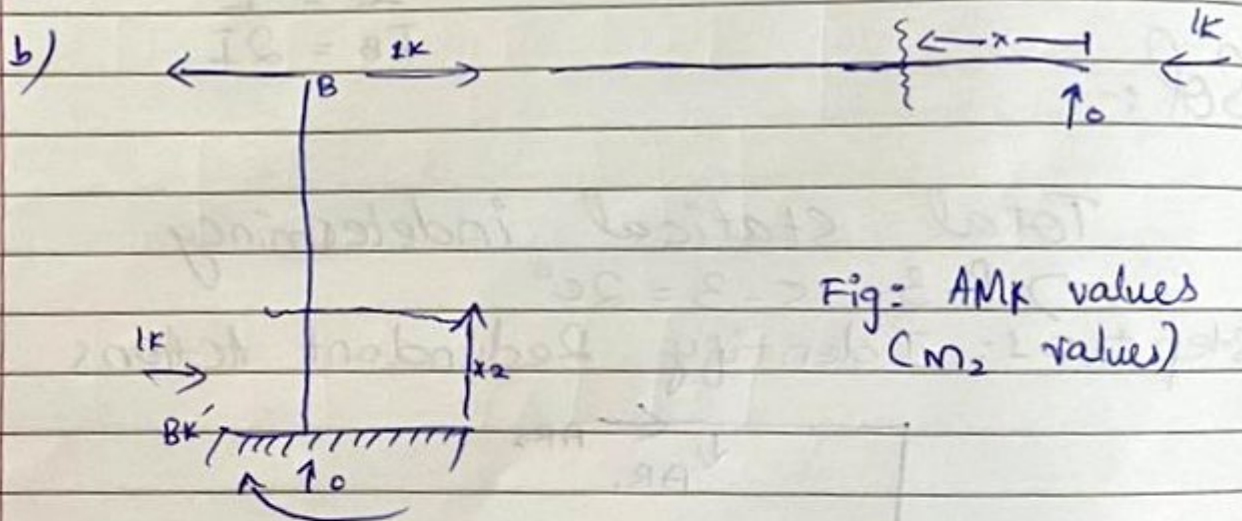


Fig: AMR values
(m_2 values)

Member	AB	BC
Origin	A	C
Limits	0-8	0-16
I	I	2I
$\leftarrow M$	$5x - 40$	0
m_1	-16	...
m_2	$8 - x$	0

Select origin should be select the support

Take x-section on AMR fig and find moment from origin

Take x-section on m_2 fig from the origin.

(3)

⇒ for Finding values of DRL's :-

$$DRL_1 = \int_0^8 \frac{M_{AB} \cdot M_1(CAB)}{EI} dx + \int_0^{16} \frac{M_{BC} \cdot M_2(BC)}{EI} dx$$

$$= \int_0^8 \frac{(5x-40)(-16) dx}{EI} + \int_0^{16} \frac{0 \cdot x dx}{E(2I)}$$

$$DRL_1 = \frac{2560}{EI}$$

$$DRL_2 = \int_0^8 \frac{(5x-40)(8-x) dx}{EI} + \int_0^{16} \frac{0 \cdot 0 dx}{E(2I)}$$

$$DRL_2 = \frac{-853.33}{EI}$$

⇒ compute Flexibility matrix :-

$$F_{2 \times 2} = \begin{bmatrix} F_{11} & F_{12} \\ F_{21} & F_{22} \end{bmatrix}$$

$$\Rightarrow F_{11} = \int_0^8 \frac{m_1^2(CAB)}{EI} dx + \int_0^{16} \frac{m_1^2(BC)}{EI} dx = \int_0^8 \frac{(-16)^2 dx}{EI} + \int_0^{16} \frac{x^2}{E(2I)} dx$$

$$F_{11} = \frac{2730.67}{EI}$$

$$F_{12} = F_{21} = \int_0^8 M_1(CAB) \cdot M_2(CAB) dx + \int_0^{16} \frac{m_2(CAB) \cdot m_2(BC)}{EI} dx$$

$$= \int_0^8 \frac{(-16)(8-x) dx}{EI} + \int_0^{16} \frac{(x)(0) dx}{2EI}$$

$$F_{12} = F_{21} = \frac{-512}{EI}$$

(4)

$$F_{22} = \int_0^8 (m_2)^2 AB \, dx + \int_0^6 (m_2)^2 BC \, dx$$

$$= \int_0^8 \frac{(8-x)^2}{EI} \, dx + \int_0^6 \frac{0^2}{2EF} \, dx$$

$$F_{22} = 170.67$$

As we know

$$[DRS] = [DRL] + [AR] \times [F]$$

$$\Rightarrow [AR] = \frac{[DRS] - [DRL]}{[F]}$$

$$(2) [AR] = [F]^{-1} \times [DRS - DRL]$$

$$= \begin{bmatrix} 2730.67 & -512 \\ -512 & 170.67 \end{bmatrix}^{-1} \times \begin{bmatrix} 0 & -2560 \\ 0 & +853.33 \end{bmatrix} = \text{Result}$$

$$\begin{bmatrix} AR_1 \\ AR_2 \end{bmatrix} = \begin{bmatrix} -0.00005 \\ 4.997 \end{bmatrix} = \begin{bmatrix} 0 \\ 5 \end{bmatrix}$$

- 1) mode → matrix
- 2) MatA (1)
- 3) select order
- 4) Put values & click =
- 5) Press [on]
- 6) Shift + 4
- 7) Press 1
- 8) Now Press 2 (matB)
- 9) select order & Put values
- 10) Press op
- 11) Shift + 45