

Q no 1(a)

Drag: A body which is wholly immersed in a homogeneous fluid may be subjected to two kind of force arising from relative motion between body and fluid these force are termed as drag and lift. If the force parallel to the motion then it is termed as drag force.

there are two components

1) **Pressure Drag:** (F_p) it is equal to integration of Component in direction of motion of all pressure force exerted on surface body

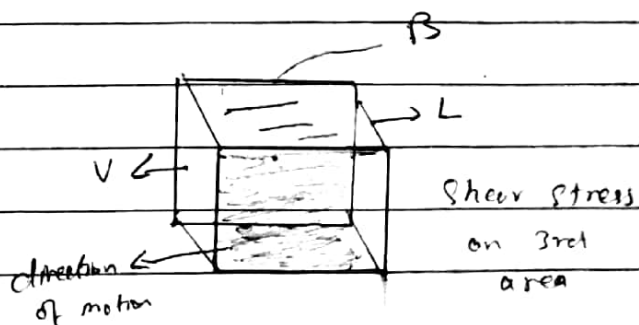
$$F_p = C_D \frac{\rho}{2} V^2 A \quad \text{where } C_D \text{ depend on shape}$$

2) **Friction Drag:** (F_f) it is equal to integration of Components of shear stress along surface of body in direction of motion

$$F_D = C_D \frac{\rho}{2} V^2 BL$$

$\underbrace{\quad\quad\quad}_L$
shear stress

$$\text{Area} = BL$$



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Friction Drag of Boundary Layer

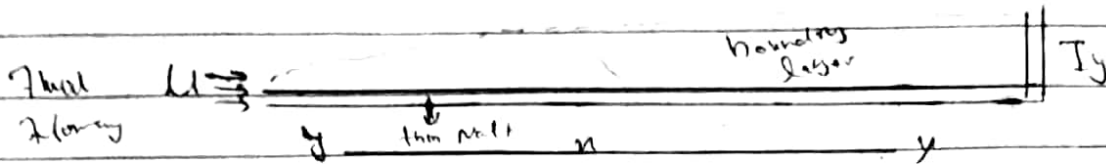
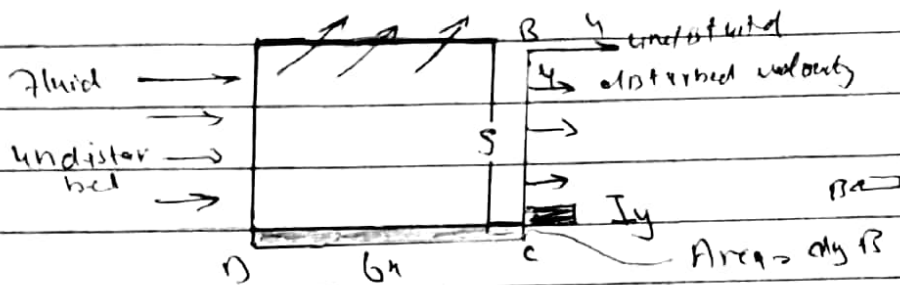


Fig shows growth of boundary layer along one side of smooth plate inside the fluid

Now consider a control volume



Where δ is thickness of boundary layer and U is undisturbed velocity. Thus $-F_x = \text{drag} = \text{rate in momentum in } x\text{-direction}$

leaving through BC + rate of momentum through AB) - rate of moment entering through DA)

$$\Delta P = P_{out} - P_{in}$$

This according to momentum

$$\sum F_x = \frac{d(mv)}{dt} = \frac{d(mou)}{dt}$$

Where $\frac{dm}{dt} = \rho Q$ thus

$$F_x = \rho Q U$$

$$F_x = \rho A U \cdot U$$

$$F_x = \rho A U^2$$

$$DA \rightarrow \rho U (U b \delta)$$

$$BC \rightarrow \rho b \int_0^\delta u^2 dy$$

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$$F_x \rightarrow \rho u \left(\rho B s - \rho B \int u \cdot dy \right)$$

Putting value

$$F_x = \rho B \int u (u - u) dy$$

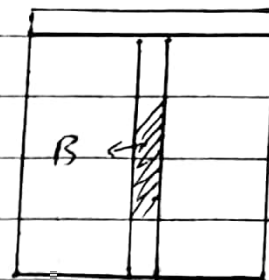
Putting it

Now to find local wall shear stress

$$\tau_0 = \frac{d(\rho u)}{B \, dn - \text{area}}$$

$$F_x = \tau_0 B u^2 s_x$$

$$\tau_0 = \tau_0 B u^2 s_x$$



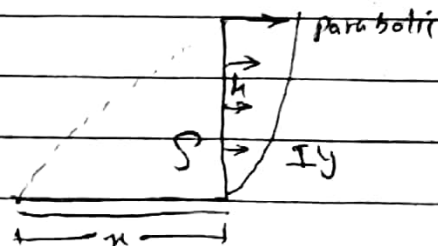
$$\tau_0 = \rho u^2 \propto \frac{ds}{dn} \text{ in general}$$

Equation of shear stress

→ Laminar boundary layer

$$\frac{\mu}{\nu} = F \left(\frac{\mu}{s} \right)$$

Assume



$$n = \frac{\mu}{s} \text{ or } y = ns$$

$$\text{thus } \frac{\mu}{\nu} = F(n) \text{ or } \mu = \nu F(n)$$

In case of laminar flow

$$\tau_0 = \mu \left(\frac{du}{dy} \right)$$

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$$V = \frac{u}{s} \frac{du}{dn} = \frac{u}{s} \left[\frac{d(\ln u)}{dn} \right]$$

Solving the equation

$$T_0 = \frac{u u \beta}{s}$$

As general equation is $T_0 = \rho u^2 \alpha \frac{ds}{dn}$

$$\text{Equating } = \rho u^2 \frac{ds}{dn}$$

or

$$s ds = \frac{u \beta}{s u} dx$$

Integrating the equation

$$\frac{s^2}{2} = \frac{u \beta}{s u} x + c$$

Now at $x=0$ $s=0$ thus $c=0$

$$\frac{s^2}{2} = \frac{u \beta}{s u} x$$

$$\text{Or } s = \sqrt{\frac{2 u \beta}{s u} x} \quad \text{or} \quad \sqrt{\frac{2 \beta}{s}} \cdot \sqrt{\frac{u x}{s u}}$$

xy and ying by u

$$s = \sqrt{\frac{2 \beta}{s}} \cdot \sqrt{\frac{u x}{s u}} \cdot \frac{x}{s u \cdot s u}$$

Where $a = 0.135$

$$\beta = 1.63$$

$$R_n = s u$$

$$s = \frac{4.91}{\sqrt{R_n}} \quad \text{or} \quad \frac{s}{u} = \frac{4.91}{\sqrt{R_n}}$$

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Now $\tau_0 = \frac{\mu U_B}{\delta}$

Now putting value

$$\tau_0 = 0.332 \frac{\mu U}{\sqrt{x}}$$

where R_{N} is Reynold's number

Now

$$F_g = B \int_0^x \frac{\tau_0 dx}{\text{stress}}$$

Putting value

$$F_g = 0.664 B \sqrt{\rho U x^3}$$

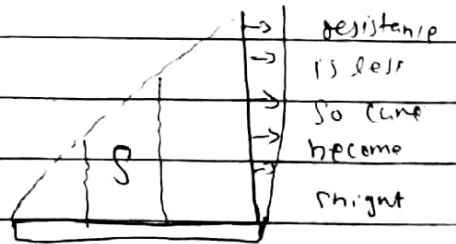
As general equation is

$$F_b = C_D \frac{\rho U^2 B L}{2} \rightarrow \text{equation both equation}$$

$$C_D = 1.328 \frac{\sqrt{x}}{\sqrt{L U}} = \frac{1.328}{\sqrt{R}}$$

Turbulent boundary layer

Figure show that velocity distribution in turbulent boundary layer has a much steeper gradient near wall and flatter through out rest of layer



the shear is greater in turbulent than in laminar layer. As we have

$$\tau_0 = \frac{7.75 \rho U^2}{8} \text{ where } U \text{ denotes average velocity of pipe}$$

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Now we have obtained an approximate relation between v and U by using pipe factor α at

$$\frac{v}{U_{\text{max}}} = \frac{1}{1 + 1.337r}$$

Using friction factor f of eq 2 from chart which is middle critical value

$$\text{So } U = 1.2357v$$

Now we have

$$\tau_0 = \frac{77}{8} U^2$$

As we have

$$f = \frac{0.316}{Re^{0.25}}$$

$$\text{thus } \tau_0 = 0.316 \frac{7U^2}{8}$$

$$\text{where } U = \frac{v}{1.2357} \text{ thus}$$

$$\tau_0 = \frac{0.316}{\left(\frac{D}{v} \left(\frac{v}{1.2357}\right)\right)^{0.25}} \cdot \frac{7}{8} \left(\frac{v}{1.2357}\right)^2$$

$$\epsilon D = 11^{-1}$$

$$\text{thus } \tau_0 = 0.0237 U^2$$

$$\left(\frac{5U}{v}\right)^{1/4}$$

As we have

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$$\tau_0 = \rho U^2 \alpha \frac{ds}{dx}$$

Equating both and integration For boundary
Condition For boundary Condition of $x=0$ to $x=L$

$$\text{thus } s = \left(\frac{0.0287}{\alpha} \right)^{4/5} \left(\frac{\nu}{U} \right)^{1/5} x$$

For

$$\alpha = 0.0972$$

$$\frac{\rho}{\mu} = \frac{0.377}{(Re)^{1/5}}$$

Putting value in equation

$$\tau_0 = 0.0587 \rho \frac{U^2}{2} \left(\frac{\nu}{U} \right)^{1/5}$$

$$\text{Now } F_b = B \int \tau_0 dx$$

$$F_b = 0.0735 \rho \cdot \frac{U^2}{2} \left(\frac{\nu}{U} \right)^{1/5} B L$$

$$\text{As } F_b = C_b \rho \frac{U^2}{2} B L$$

Equation both

$$C_b = \frac{0.0735}{Re^{1/5}}$$

For $Re > 10^7$

$$C_b = \frac{0.455}{(\log Re)^{2.58}}$$

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Q no 1 B

As

↳ Specific energy $E = y + \frac{V^2}{2g}$

The flow Q per unit width 'b'
Can be expressed as

↳ $Q = \frac{Q}{b}$

Now average velocity will be

↳ $V = \frac{Q}{A} = \frac{Qb}{by} = \frac{Q}{y}$

Thus

$E = y + \frac{Q^2}{2g} \Rightarrow y + \frac{1}{2g} \left(\frac{Q^2}{y^2} \right)$

$(E - y) = \frac{1}{2g} \left(\frac{Q^2}{y^2} \right)$ or $(E - y)y^2 = \frac{Q^2}{2g}$

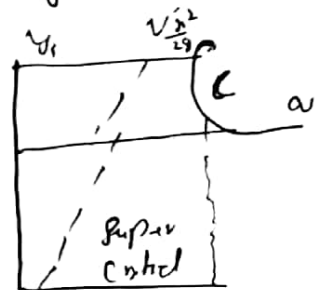
Thus point of E vs y will be parabolic for particular Q there will be two kind of possible values of y , for given E

The equation is

Cubic with three roots being negative points C represent dividing points between two regions

of flow thus for given Q & value of E is minimum of flow at that point is

critical flow depth of flow at that point is critical the y_c and velocity at that point is critical V_c .



Thus $E = y + \frac{1}{2g} \left(\frac{Q^2}{y^2} \right)$

For minimum specific energy $\frac{dE}{dy} = 0$

Thus $\frac{dE}{dy} = 1 - \frac{2}{2g} \left(\frac{Q^2}{y^3} \right) = 0$

$$\frac{v^2}{gy^3} = 1 \Rightarrow v^2 = gy^3$$

$$\hookrightarrow \frac{v^2}{gy^3} = 1 \Rightarrow v^2 = gy^3$$

Now

$$\hookrightarrow v^2 = gy^3$$

$$\& \quad v = vy \Rightarrow v^2 y^2 = gy^3$$

or

$$v^2 = gy_{cr}$$

or

$$v_{cr} = \sqrt{gy_{cr}}$$

Q no 2 Given

Water flows at rate, $Q = 3.5 \text{ m}^3/\text{s}$

Bed Slop, $S_0 = 0.0008$

$$n = 0.0219$$

Width of bed in student ID = 7549 mm

Required

Depth of rectangular channel?

Critical Depth $y_c = ?$

Critical Velocity $V_c = ?$

Flow is critical Sub-critical = ?

Sol

Manning Equation

$$Q = \left(\frac{1}{n} R n^{2/3} S_0^{1/2} \right) A \rightarrow 0$$

Area $7549 \times d$

Perimeter = $d + 7549 + d$

Hydraulic Radius $R_h = \frac{\text{Area}}{\text{Perimeter}}$

Perimeter

$$\frac{7549d}{2d + 7549}$$

$$2d + 7549$$

Put value in eq 1

$$Q = \left(\frac{1}{n} R n^{2/3} S_0^{1/2} \right) A$$

$$3.5 = \left(\frac{1}{0.0219} \times \frac{7549d}{2d + 7549} \right)^{2/3} \times (0.0008)^{1/2} \times 7549d$$

$$\left(\frac{3.5 \times 0.0219}{(0.0008)^{1/2}} \right)^{3/2} = \left(\frac{7549}{2d + 7549} \right)^{2/3} \times 7549d$$

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$$\left(\frac{3.5 \times 0.0219}{\sqrt{0.008}} \right)^{3/2} = \frac{56.98d^2}{9d + 7.549}$$

$$4.461(9d + 7.549) = 56.98d$$

$$8.922d + 33.67 = 56.98d^2$$

$$56.98 - 8.922d - 33.67 = 0$$

$$d = 0.719 \text{ m}$$


So the depth of channel is $\boxed{0.383 \text{ m}}$

Now

As $q = \text{discharge per unit width}$

$$q = \frac{Q}{b}$$

$$= \frac{3.5}{7.549} = 0.46$$

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$$q = 0.46$$

→ critical depth
using equation

$$y_c = \left(\frac{q^2}{g} \right)^{1/3}$$

$$y_c = \left(\frac{(0.46)^2}{9.81} \right)^{1/3}$$

$$y_c = 0.28 \text{ m}$$

Critical Velocity
using equation

$$V_c = g y_c$$

$$V_{cr} = \sqrt{(9.81)(0.28)}$$

$$V_{cr} = 1.73 \text{ m/s}$$

$$V = \frac{Q}{A} = \frac{3.5}{7549 \times 0.383}$$

$$V = 1.21 \text{ m/s}$$

$$y = 0.383 \text{ m} \quad y_c = 0.28 \quad V_{cr} = 1.73 \text{ m/s}$$

As $y > y_c$

and

$$V < V_{cr}$$

Signature: So flow is sub critical flow

Q no 3 Given data

Width of smooth plate, $B = 200 \text{ mm}$
 $= 0.2 \text{ m}$

Length of smooth plate, $L = 800 \text{ mm}$
 $= 0.8 \text{ m}$

Oil with specific gravity, $S = 0.89$

Underspeed velocity $U = 5 \text{ m/s}$

kinematic viscosity $\nu = 0.93 \times 10^{-4} \text{ m}^2/\text{s}$

Required data

Friction drag on one side of a smooth plate $F_D = ?$

Sol Check the flow

$$\text{As } \nu = 0.93 \times 10^{-4} \text{ m}^2/\text{s}$$

$$R = \frac{LU}{\nu} = \frac{(0.8)(5)}{0.93 \times 10^{-4}}$$

$$R = 43010.75 < 500,000$$

thus flow is laminar

Now

$$C_f = \frac{1.328}{\sqrt{R}} \Rightarrow \frac{1.328}{\sqrt{43010.75}}$$

$$C_f = 6.403 \times 10^{-3}$$

$$C_D = 0.0064$$

$$\Rightarrow F_D = C_D \int \frac{\rho U^2}{2} B L$$

$$(0.0064) (S_{oil} \times \rho_{water}) \times \frac{5^2}{2} (0.2) (0.8)$$

$$(0.0064) (0.89 \times 1000) \times \frac{5^2}{2} \times (0.2) (0.8)$$

$$F_D = 11.392 \text{ N}$$