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Subject:- Mechanics of Solid - 2

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Q1) A): Given Data:-

Required :-

Location of Shear Center

As we know that

$$e = \frac{bgh^2b^2}{4I}$$

$$I = 2 \left( \frac{bh^3}{12} + Ay^2 \right) + \left( \frac{bh^3}{12} + Ay^2 \right)$$

$$I = 2 \left[ \frac{26(2)^3}{12} + (20 \times 2)(25)^2 \right] + \left[ \frac{2(50)^3}{12} + 0 \right]$$

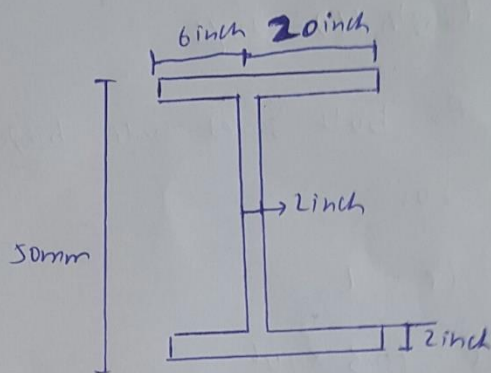
$$I = 50037.66 + 20833$$

$$I = 70867.99 \text{ mm}^4$$

$$e = \frac{2(50)^2(25)^2}{4(70867.99)}$$

$$e = 11.02 \text{ mm}$$

So Shear Center (e) = 11.02 mm Answer.



Q2) b) :: Given Data ::

$$h = 26 \text{ ft}$$

or

$$h = 312 \text{ inches.}$$

$$\sigma_t = 6000 \text{ psi}$$

$$\rho_w = 62.4 \text{ lb/ft}^3$$

or

$$\rho_w = 0.036 \text{ lb/in}^3$$

$$\text{Diameter (D)} = 22 \text{ ft}$$

Required ::

$$\text{Thickness (t)} = ?$$

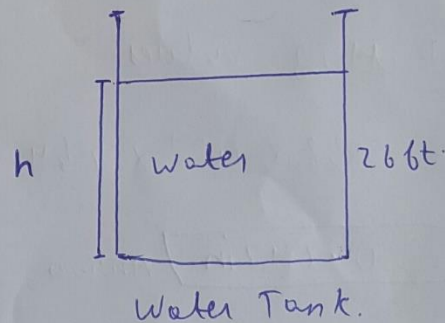
We know that,

$$P = \rho h \text{ (for water)}$$

$$\sigma_t = \frac{PD}{2t}$$

$$\sigma_t = \frac{\rho h D}{2t}$$

$$t = \frac{\rho h D}{2\sigma_t}$$



$$t = \frac{62.4}{12^3} \times \frac{(22 \times 12)}{2 \times 6000} \times D.$$

$$t = 9.38 \times 10^{-4} D$$

Dia (D) = 226t or 22 x 12 = 264 in.

We know that

$$t = 9.38 \times 10^{-4} D.$$

Putting values

$$t = 9.38 \times 10^{-4} \times 264$$

$$t = 0.247 \text{ in.} \text{ Answer.}$$

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QNo. - 2 (A):

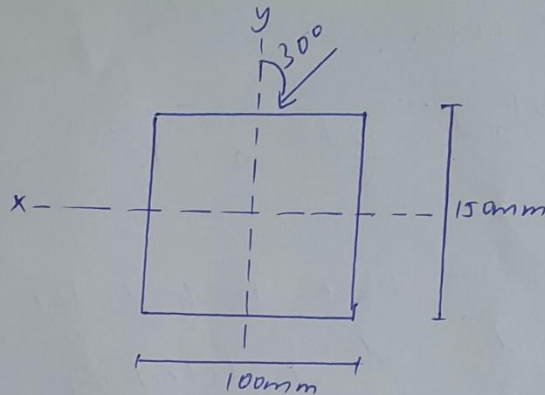
Given Data:-

$$w = 4 \text{ KN/m}$$

$$L = 3 \text{ m}$$

Required:-

Maximum Bending Stress = ?



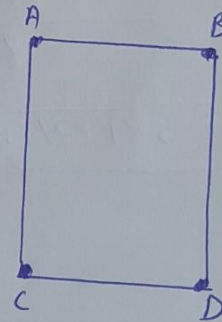
Sol:-

As the Bending Moment is maximum at extreme, so we could find stresses at A, B, C, D.

We know that,

$$\sigma = \frac{M_x y}{I_x} + \frac{M_y x}{I_y}$$

We have to find  $M_x$  and  $M_y$ .

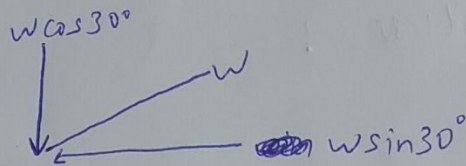


As per given data, the  $M_x$  and  $M_y$  should be found at the mid.

For simply supported,  
we have,

$$M_{mid} = \frac{wl^2}{8} \rightarrow (i).$$

Now we have to find the components of  $w$   
in  $x$  and  $y$  directions.



So,

$$M_x = \frac{(w \cos 30^\circ) \times l^2}{8}$$

$$M_x = \frac{(4 \times \cos 30^\circ) \times 3^2}{8}$$

$$M_x = 3.9 \text{ KM-m}$$

Now,

$$M_y = \frac{(4 \times \sin 30^\circ) \times 3^2}{8}$$

$$M_y = 2.25 \text{ KM-m}$$

$M_x$  is causing compression at A and B & tension at C and D.

$M_y$  is causing compression at B & D and tension at A & C.

Now,

$$I_x \text{ \& } I_y.$$

$$I_x = \frac{bh^3}{12} = \frac{0.1 \times 0.15^3}{12} = 2.815 \times 10^{-5} \text{ m}^4$$

$$I_y = \frac{bh^3}{12} = \frac{0.15 \times 0.1^3}{12} = 1.25 \times 10^{-5} \text{ m}^4$$

→ Now stresses at Extreme fibers.

$$\sigma_x = \frac{M_x y}{I_x} = \frac{3.9 \times 0.075}{2.815 \times 10^{-5}}$$

$$\sigma_x = 10390.7 \text{ KN/m}^2$$

$$\sigma_y = \frac{2.25 \times 0.05}{1.25 \times 10^{-5}}$$

$$\sigma_y = 9000 \text{ KN/m}^2$$

Now, (taking tension +)

$$\text{Stress at A} = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$= -10390.7 + 9000$$

$$= -1390.7 \text{ KN/m}^2 \text{ (comp)}$$

$$\text{At B,} \quad = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$= -10390.7 - 9000$$

$$\sigma \text{ at B} = -19390.7 \text{ KN/m}^2 \text{ (comp)}$$

Now,

$$\text{Stress at C} = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$= 10390.7 + 9000$$

$$= 19390.7 \text{ KN/m}^2 \text{ (Tension)}$$



$$\begin{aligned} \text{Stress at D} &= \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y} \\ &= 10390.7 - 9000 \end{aligned}$$

$$= 1390.7 \text{ KN/m}^2 \text{ (Tension)}$$

So the maximum stresses are on B and C.

B is under compression of 19390.7 KN/m<sup>2</sup>

And C is under tension of the same value.



Q No. 2 (B):

Given Data:-

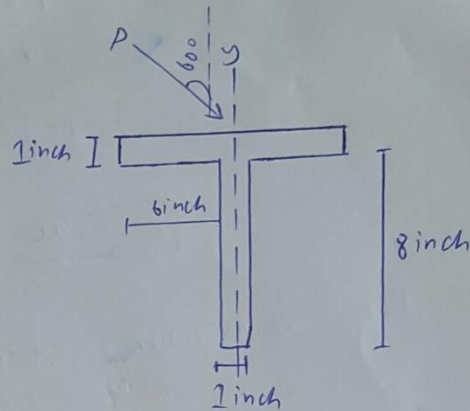
$$l = 16 \text{ ft}$$

$$I_x = 112.6 \text{ in}^4$$

$$I_y = 18.7 \text{ in}^4$$

$$E = 12000 \text{ psi}$$

$$\sigma_t = 5000 \text{ psi}$$

Sol:-

From figure, we know that maximum compression would occur on A and maximum tension at C. At B there will be tension as well as compression which will reduce the effect of each other. So, we will calculate stresses at A and C.

So,

$$\sigma_A = \frac{Mx_y}{I_x} + \frac{My_x}{I_y} \quad (\text{comp})$$

$$\sigma_C = \frac{Mx_y}{I_x} + \frac{My_x}{I_y} \quad (\text{Tension})$$

Now  $M_x$  and  $M_y$ :-

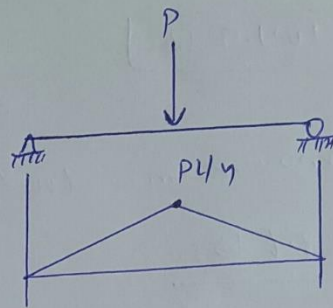
So,

$$M_x = \frac{P \cos 60 \times (16 \times 12)}{4}$$

$$M_x = 48P \cos 60^\circ$$

$$M_y = \frac{P \sin 60 (16 \times 12)}{4}$$

$$M_y = 48P \sin 60^\circ$$



Now,

$$\sigma_A = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$12000 = \frac{48P \cos 60^\circ \times 3.07}{112.6} + \frac{48P \sin 60^\circ \times 3}{18.7}$$

Solving the Equation.

$$P = 1638.6 \text{ N}$$

Now,

$$\sigma_c = \frac{M_{xy}}{I_x} + \frac{M_{yx}}{I_y}$$

$$5000 = \frac{48P \cos 60 \times (5.93)}{112.6} + \frac{48P \sin 60 \times 0.5}{18.7}$$

Solving the equations

$$P = 2104.9 \text{ Nbs}$$

So the maximum load  $P$  applied should  
be 1638.6 Nbs.

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Q No. 21

Q.No.: 3): Given Data:.

$$\text{Length } (L) = 10 \text{ ft}$$

As both sides are hinged

So,

$$L_e = L$$

$$E = 10.3 \times 10^6$$

$$FOS = 2$$

$$b = 0.75 \text{ inch}$$

$$h = 2 \text{ inch}$$

Required:

Determine safe load, -?

$$P_{cr} = \frac{\pi^2 EI}{L_e^2}$$

We know that

$$I = A r^2$$

$$\text{or}$$

$$r = \sqrt{\frac{I}{A}}$$

$$r = \frac{\sqrt{\frac{bh^3}{12}}}{bh} \Rightarrow \sqrt{\frac{b^2}{12}}$$

$$\delta = \frac{b}{2\sqrt{3}} = \frac{0.75}{2\sqrt{3}}$$

$$\delta = 0.216 \text{ inch}$$

$$P_u = \frac{\pi^2 EA}{(L_e/b)^2}$$

$$= \frac{(3.14)^2 (10.3 \times 10^6) (1.5)}{(10/0.216)^2}$$

$$P_u = 853.8343$$

$$\text{Safe load} = \frac{\text{Crippling load}}{F.O.S.}$$

$$= \frac{853.8343}{2}$$

$$\text{Safe load} = 426.917$$

① For Fixed Ended Column:

$$L_e = L/2 = 10/2$$

$$L_e = 5 \text{ ft}$$

$$P_u = \frac{\pi^2 EA}{(l_e/b)^2} = \frac{(3.14)^2 \times (10.3 \times 10^6) (1.5)}{(60/16)^2}$$

$$P_u = 1974.207$$

$$\text{Safe load} = \frac{P_u}{\text{F.O.S.}}$$

$$\text{Safe load} = \frac{1974.207}{2}$$

$$\text{Safe load} = 987.103 \text{ Answer.}$$

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